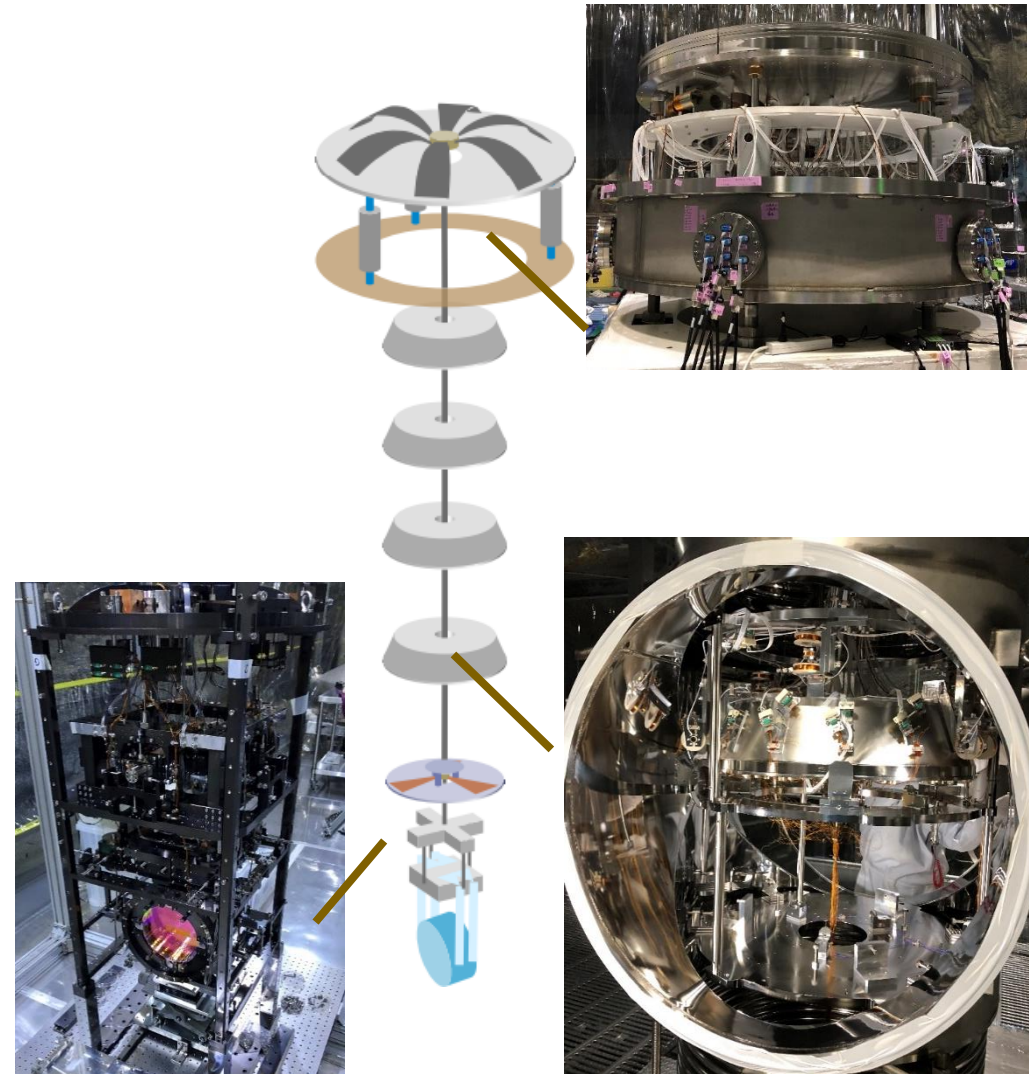


Fast localization of coalescing binaries with gravitational wave detectors and
**Low frequency vibration
isolation for KAGRA**

Yoshinori Fujii (U. of Tokyo, NAOJ)

KAGRA F2F meeting



Thesis contents

1. Introduction

2. Benefit of adding detectors to the observation network

Fast localization simulation with one template search in hierarchical approach using HLVK-network [\[ref HLV\]](#)

3. Low frequency vibration isolation

4. KAGRA seismic Attenuation system

5. Suspension control design

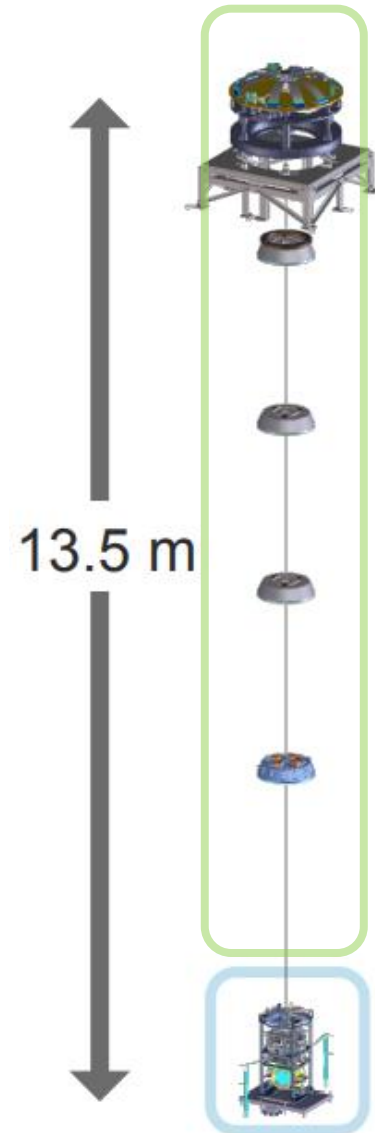
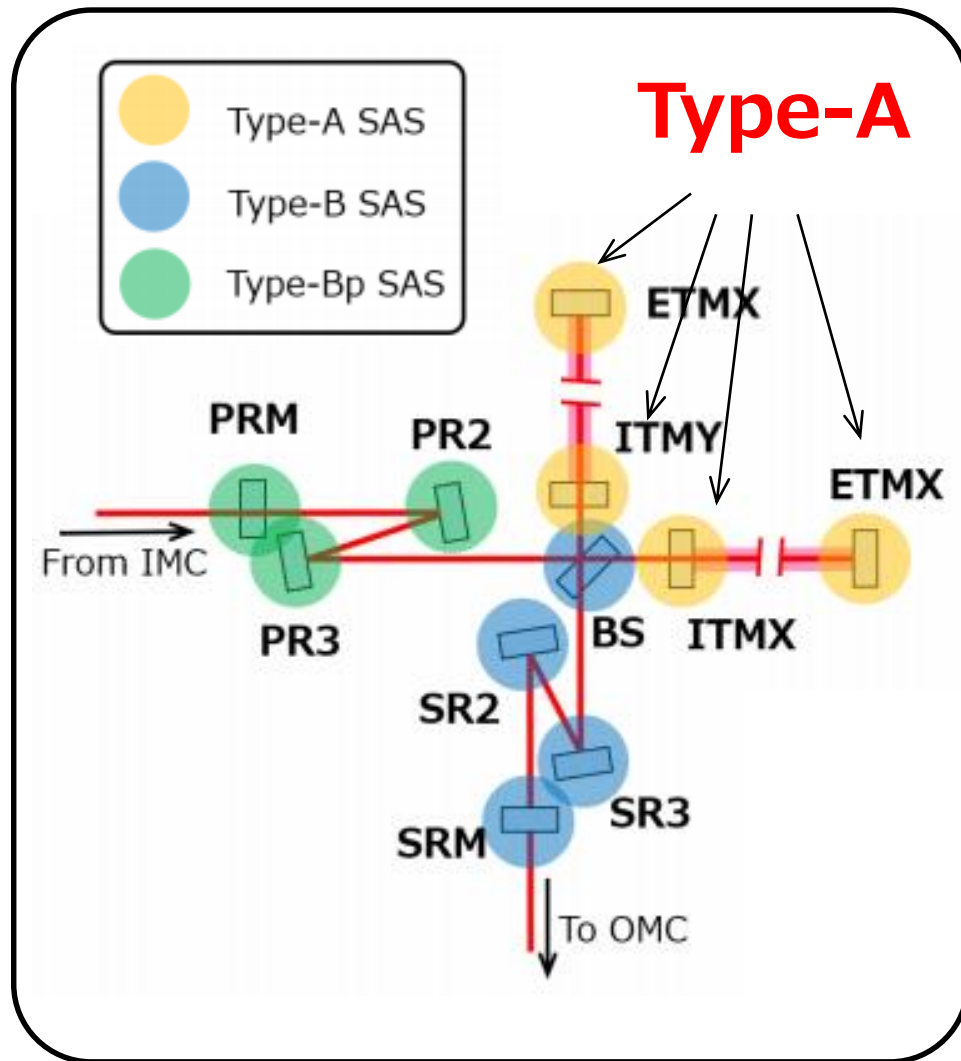
Today's topic

6. Performance test of local control for KAGRA Type-A suspension

7. Summary

→ **TypeA suspension & Controls toward lock acquisition**

Before starting, What's Type-A?

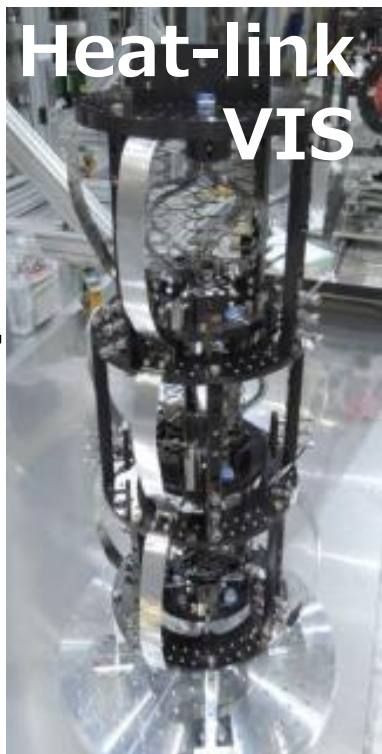
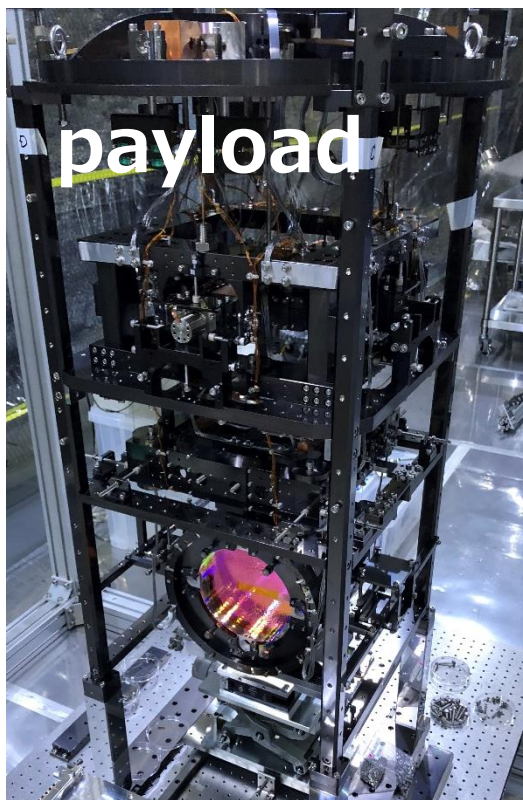
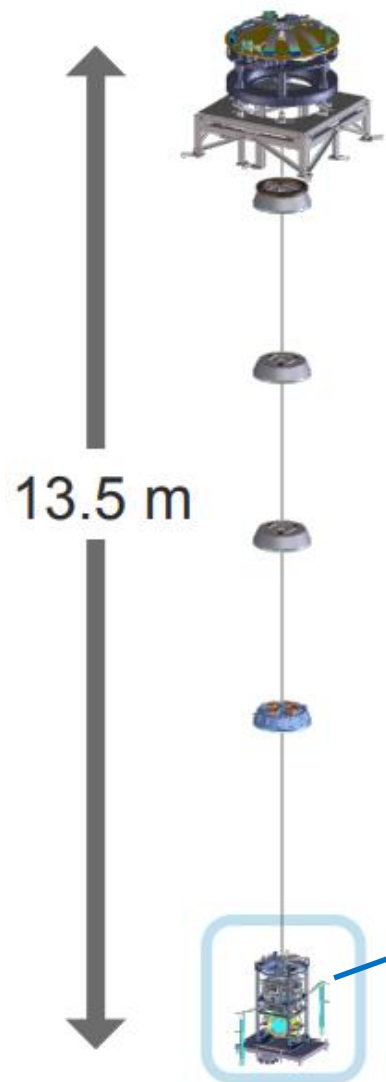


→ The longest suspension in KAGRA

- Upper 5 stages: room-temperature
- Lower 4 stages: cryogenic-temperature

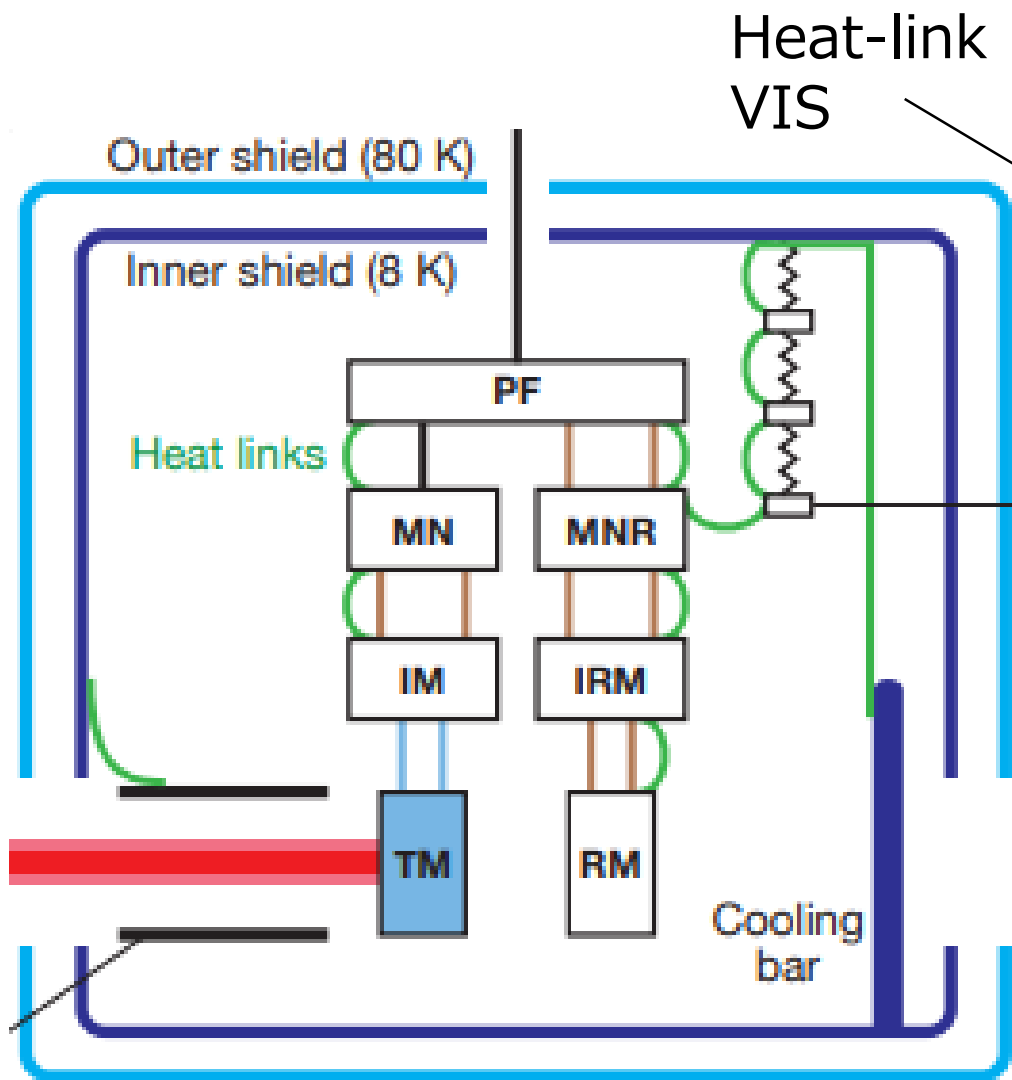
And, Type-A has cryogenic part

Inside cryostat



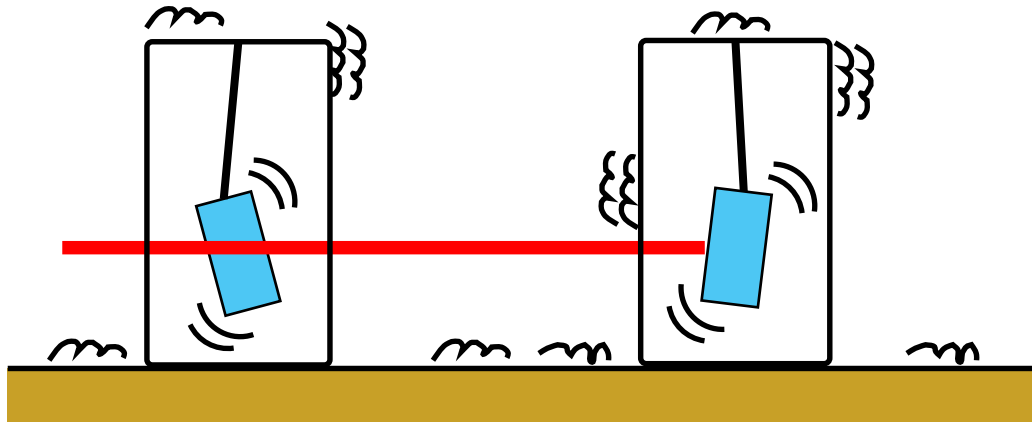
+

(+ Wide-angle baffle)

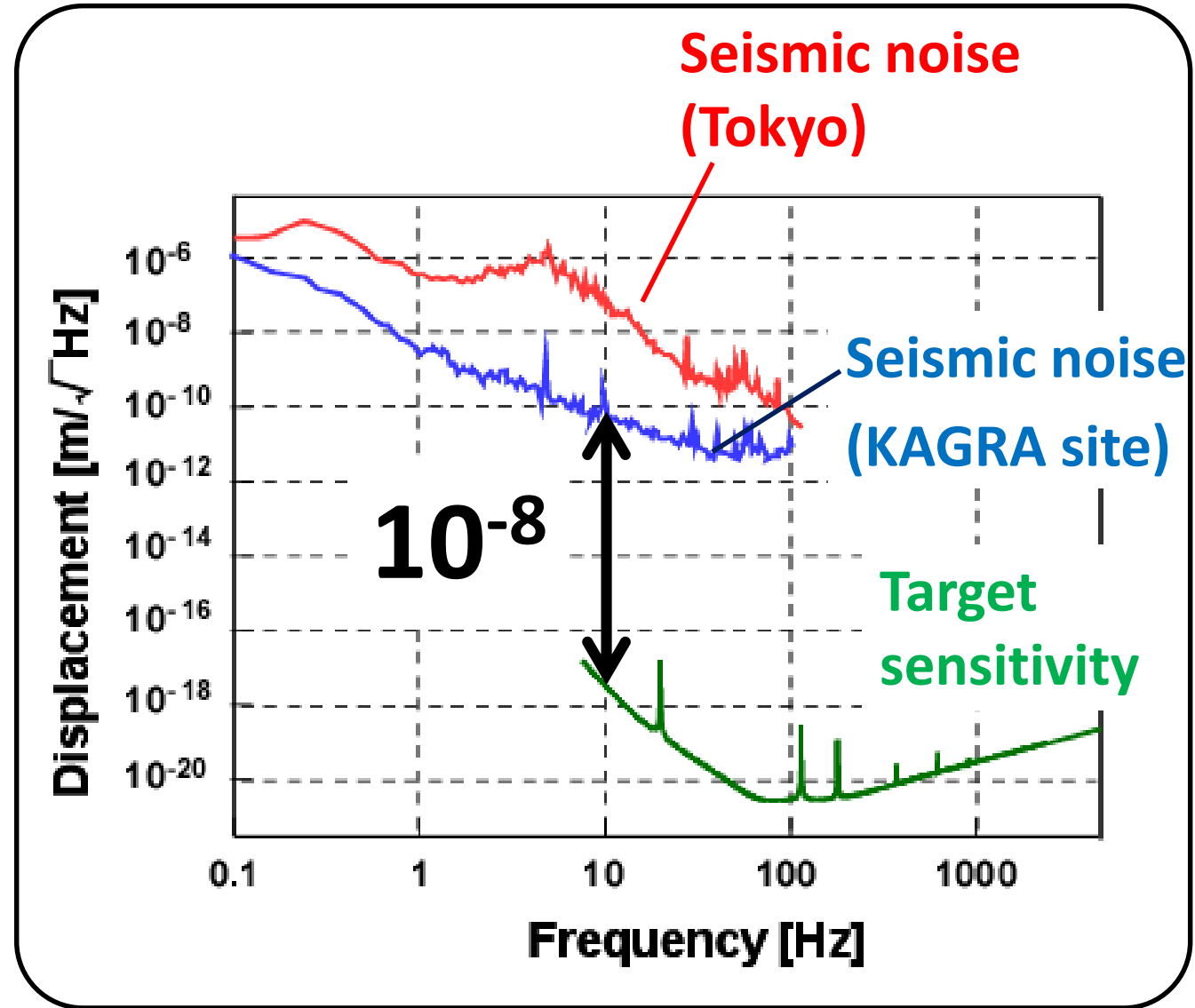


JGW-P1809347

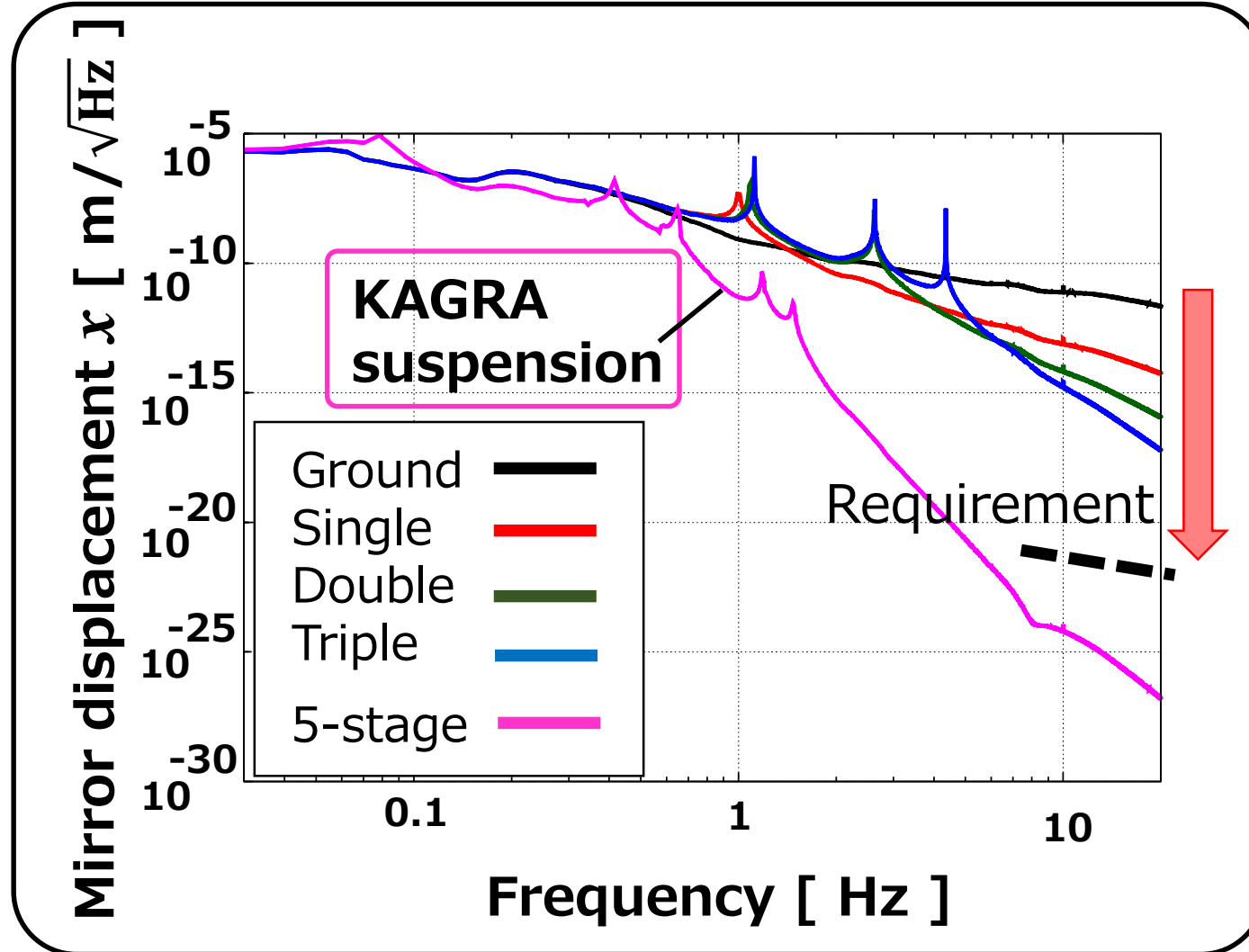
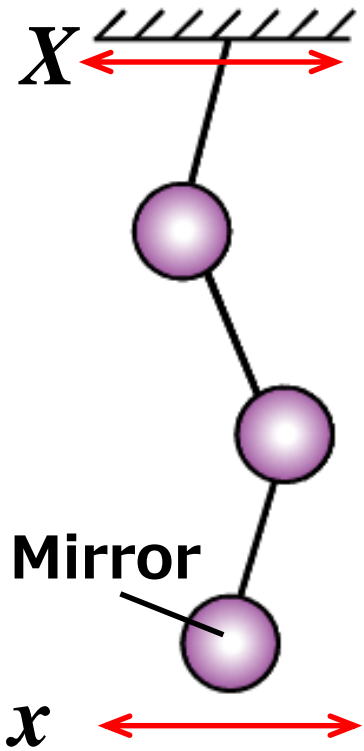
Why pendulum?



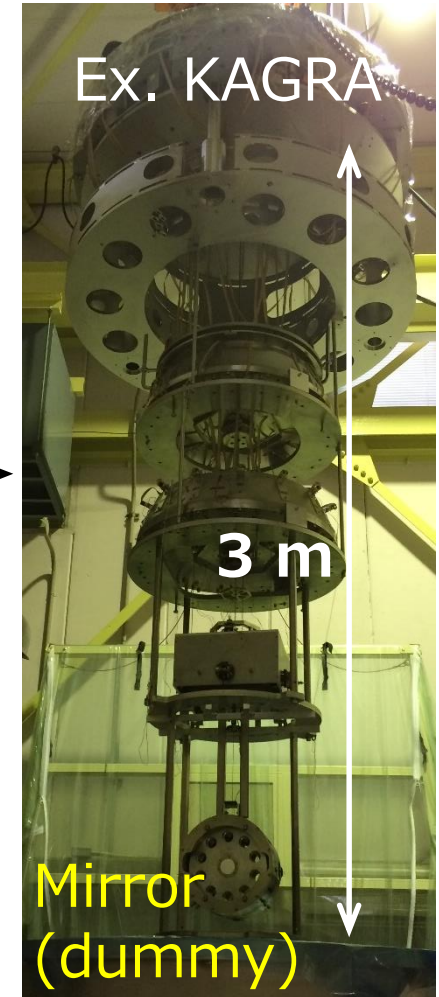
→ To treat seismic noise



Seismic noise attenuation → pendulum



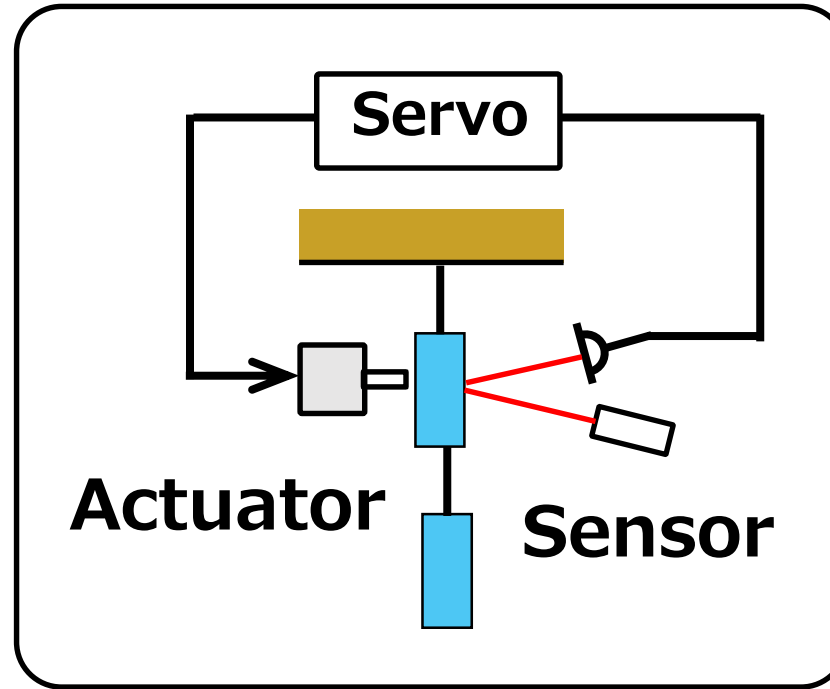
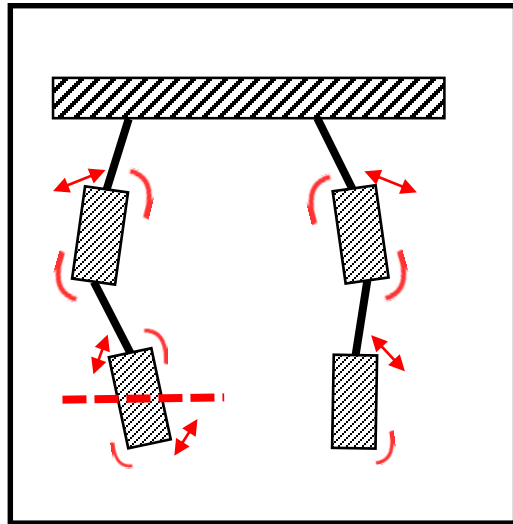
(BS/SRs-suspension case)



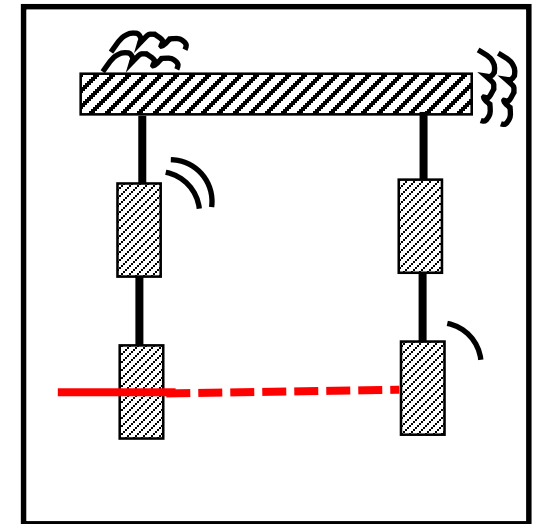
Resonant damping / Mirror Alignment: necessary

→ Control system

~~interferometer operation~~



Stable operation

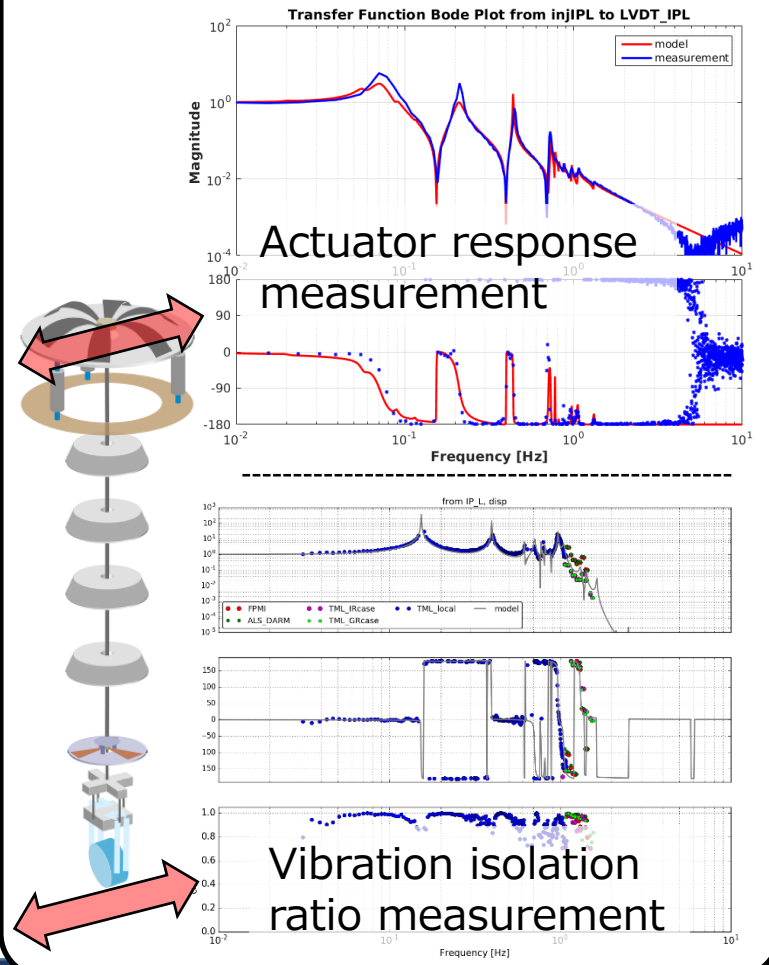


- * DAMP resonances.
- * Don't care the controls noise.

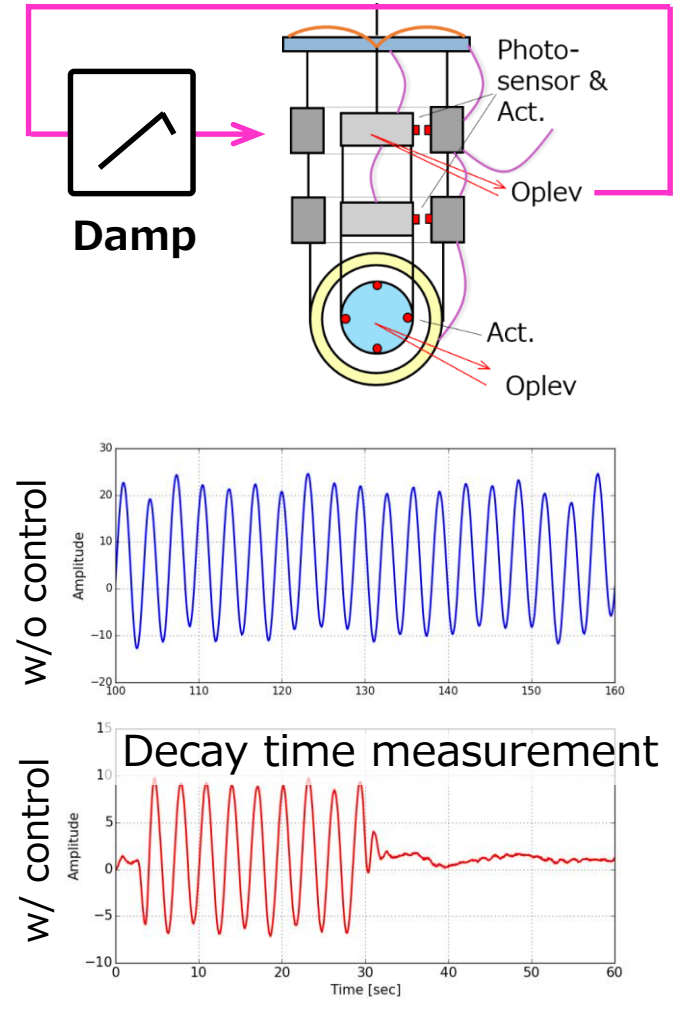
- * Freeze the mirror
- * Keep low noise

My work: Constructing controls toward interferometer lock

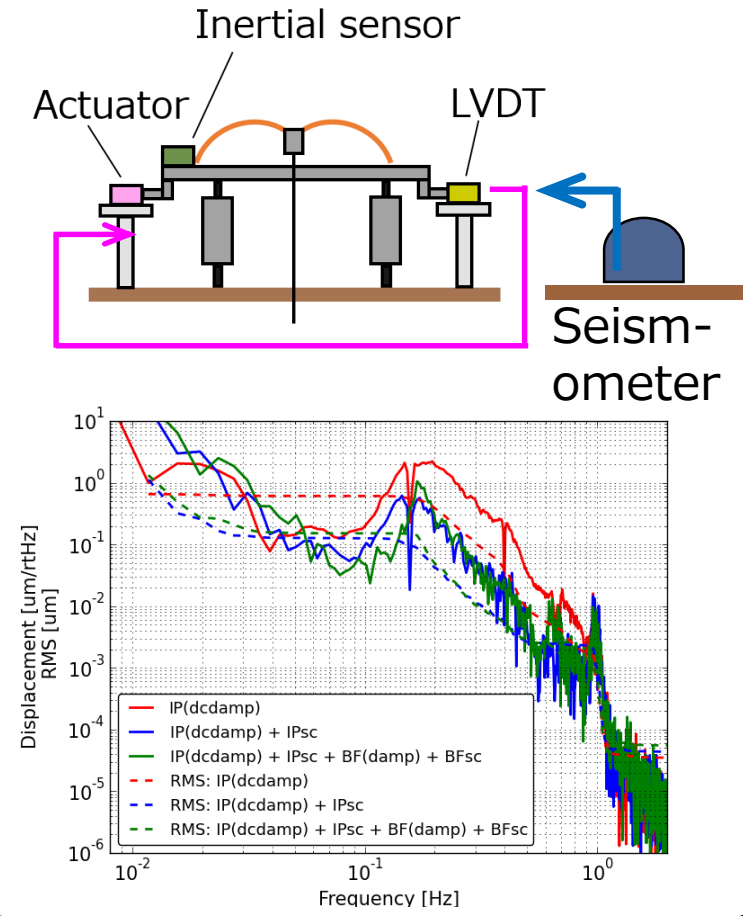
1. Mechanical system characterization



2. Resonance damping

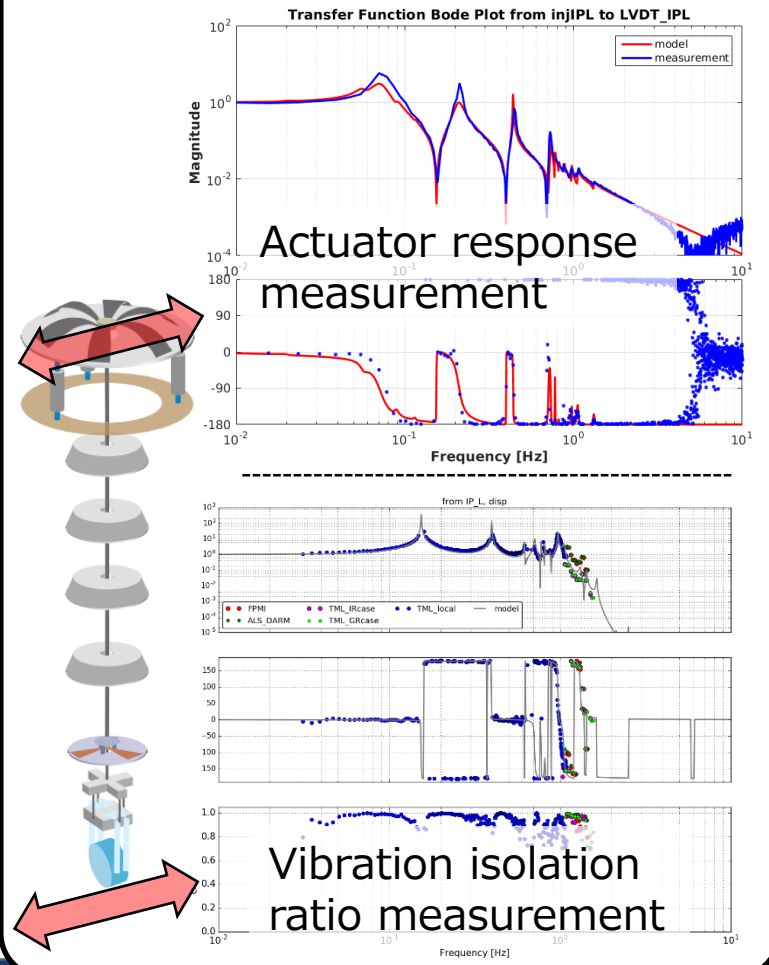


3. Mirror residual suppression

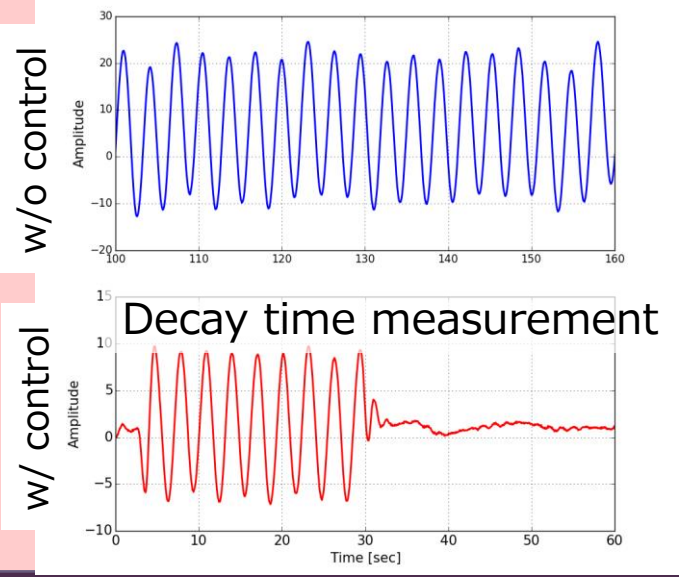
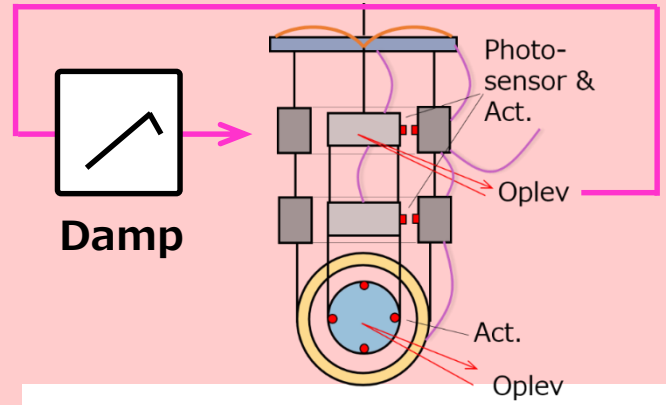


My work: Constructing controls toward interferometer lock

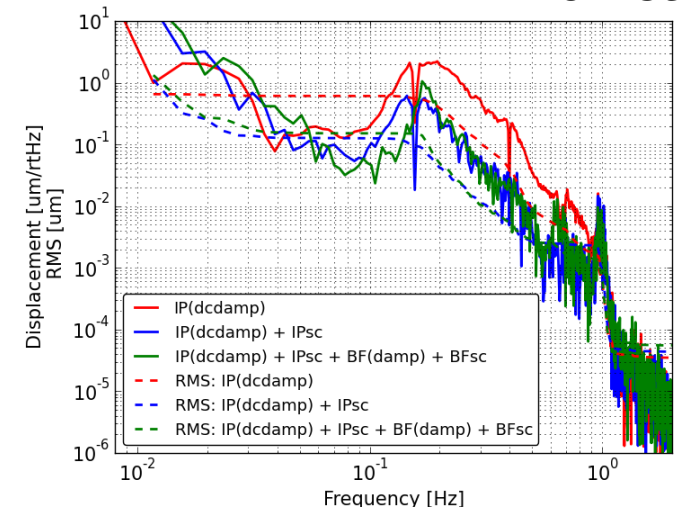
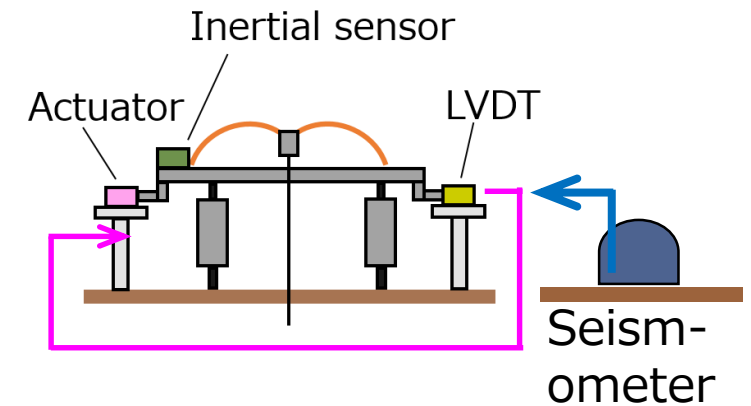
1. Mechanical system characterization



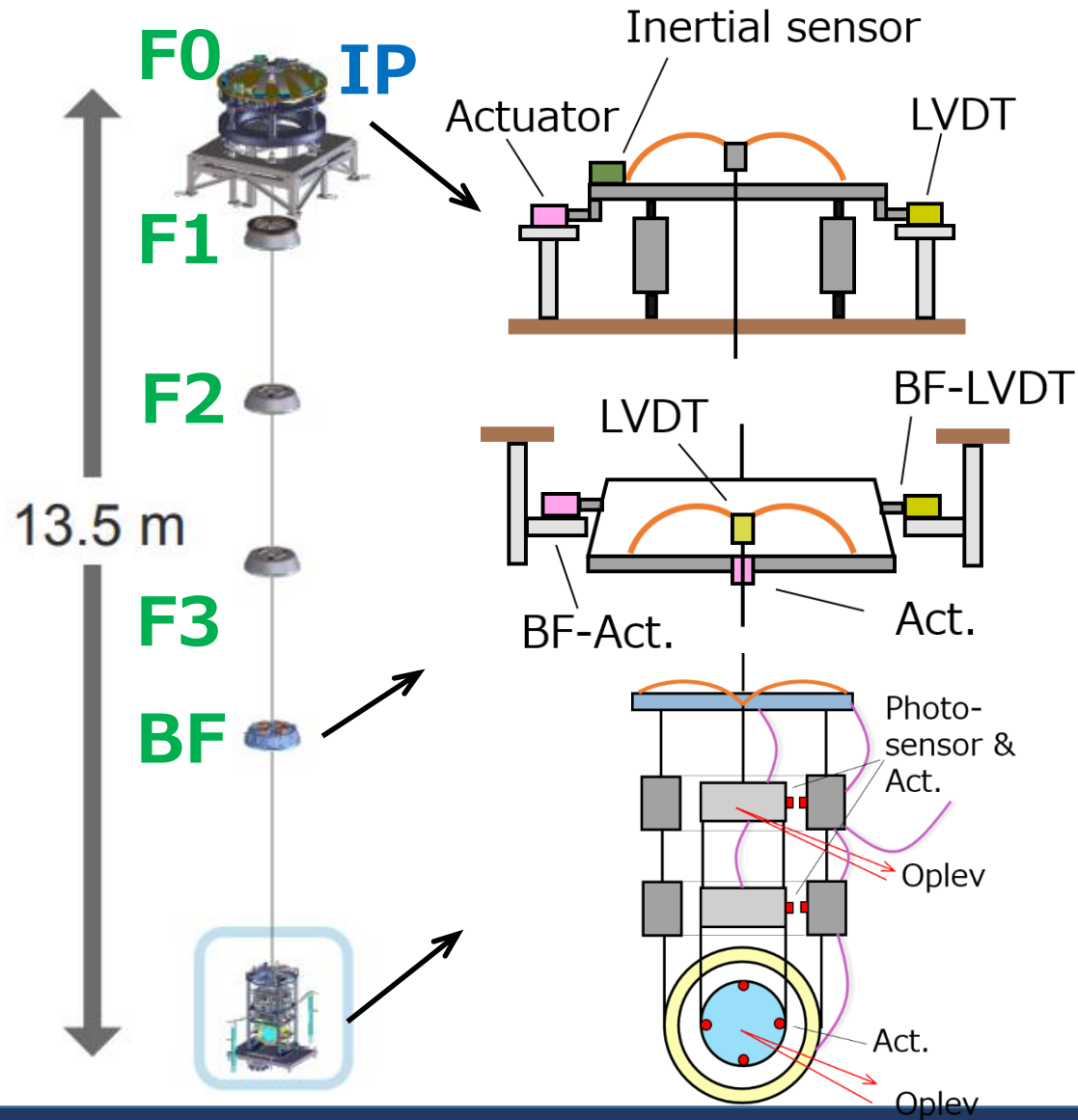
2. Resonance damping



3. Mirror residual suppression



Control system for damping



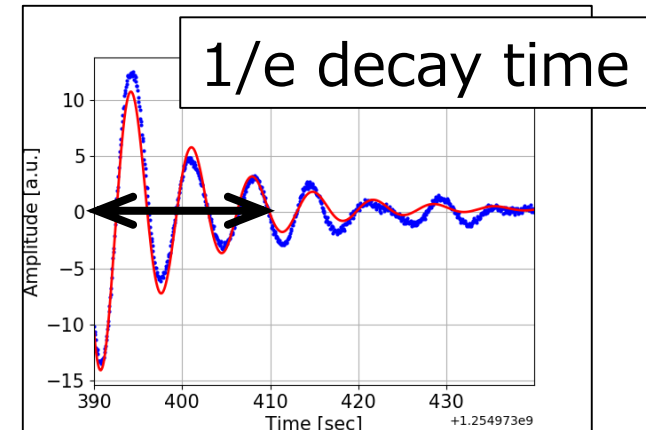
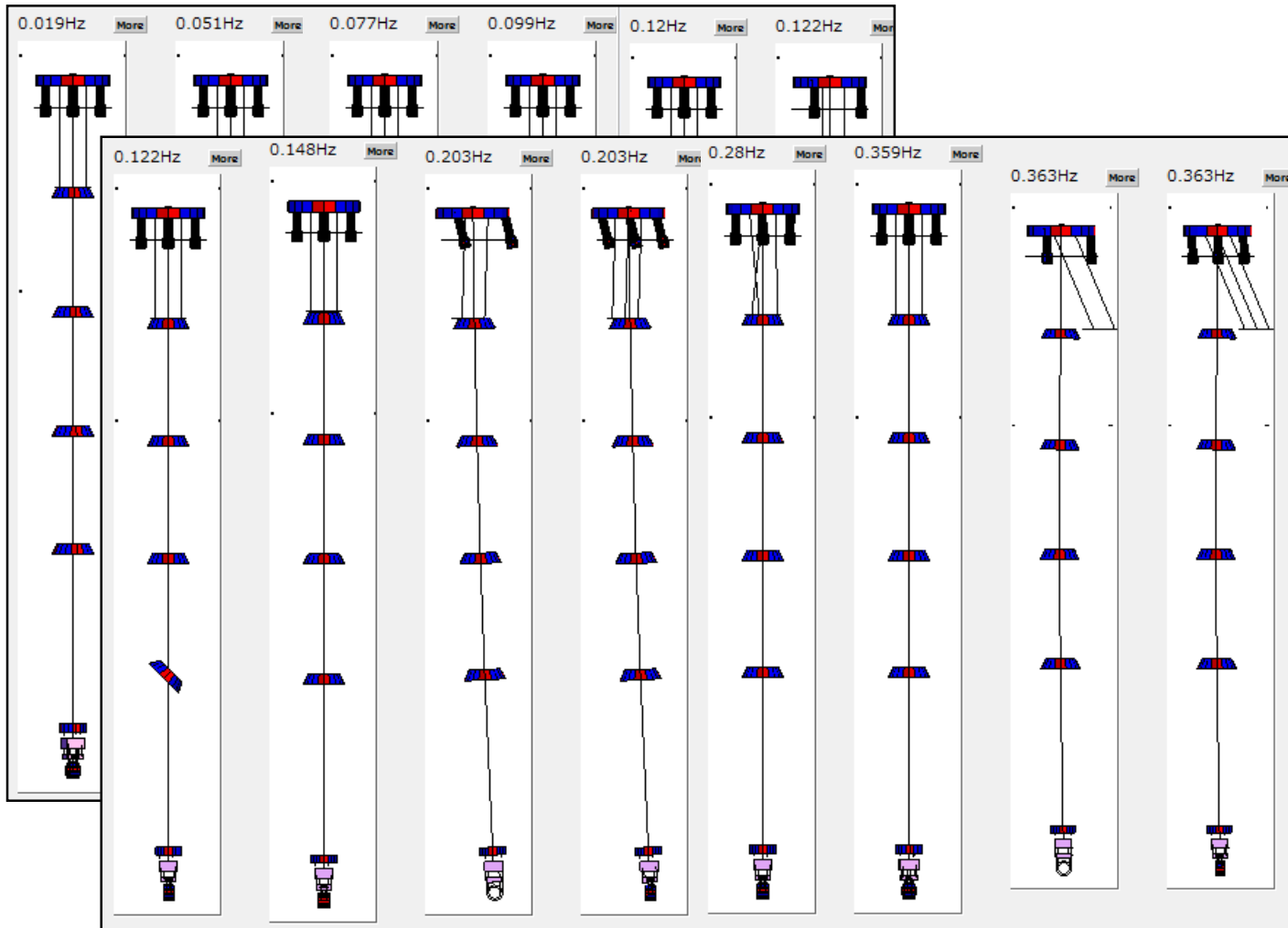
Tower part:

- * controlled at each stage
- * DC or (DC+damping) control

Cryogenic payload part:

- * Actuated at IM(+MN) stages
- * Mainly optical levers are used.
- * Most of nuisance RM-chain modes
→ Finely tuned band-pass filters

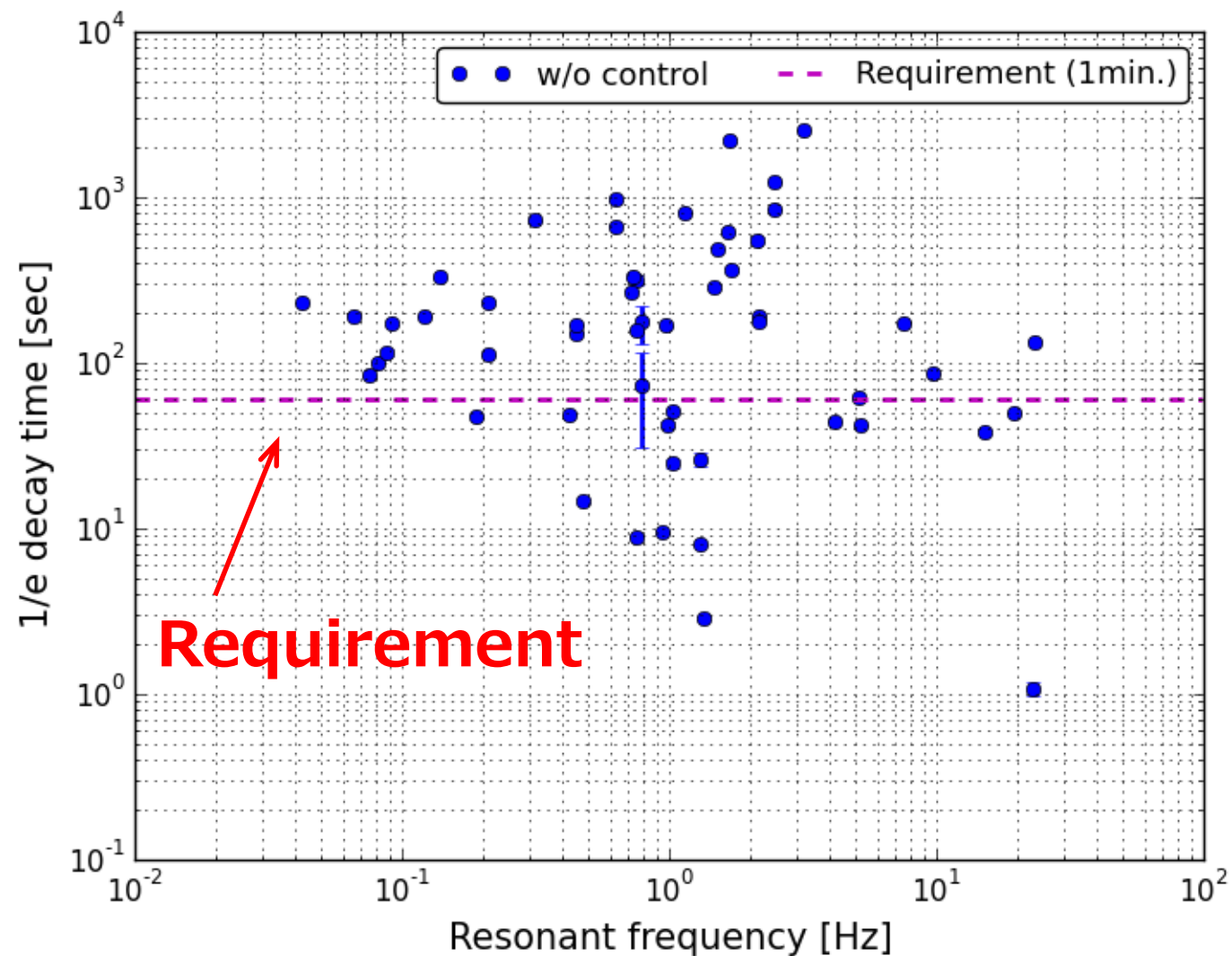
Target: **Damp all the resonances which disturbs the lock acquisition, within 1min.**



Assuming rigid-bodies
→ 75 modes

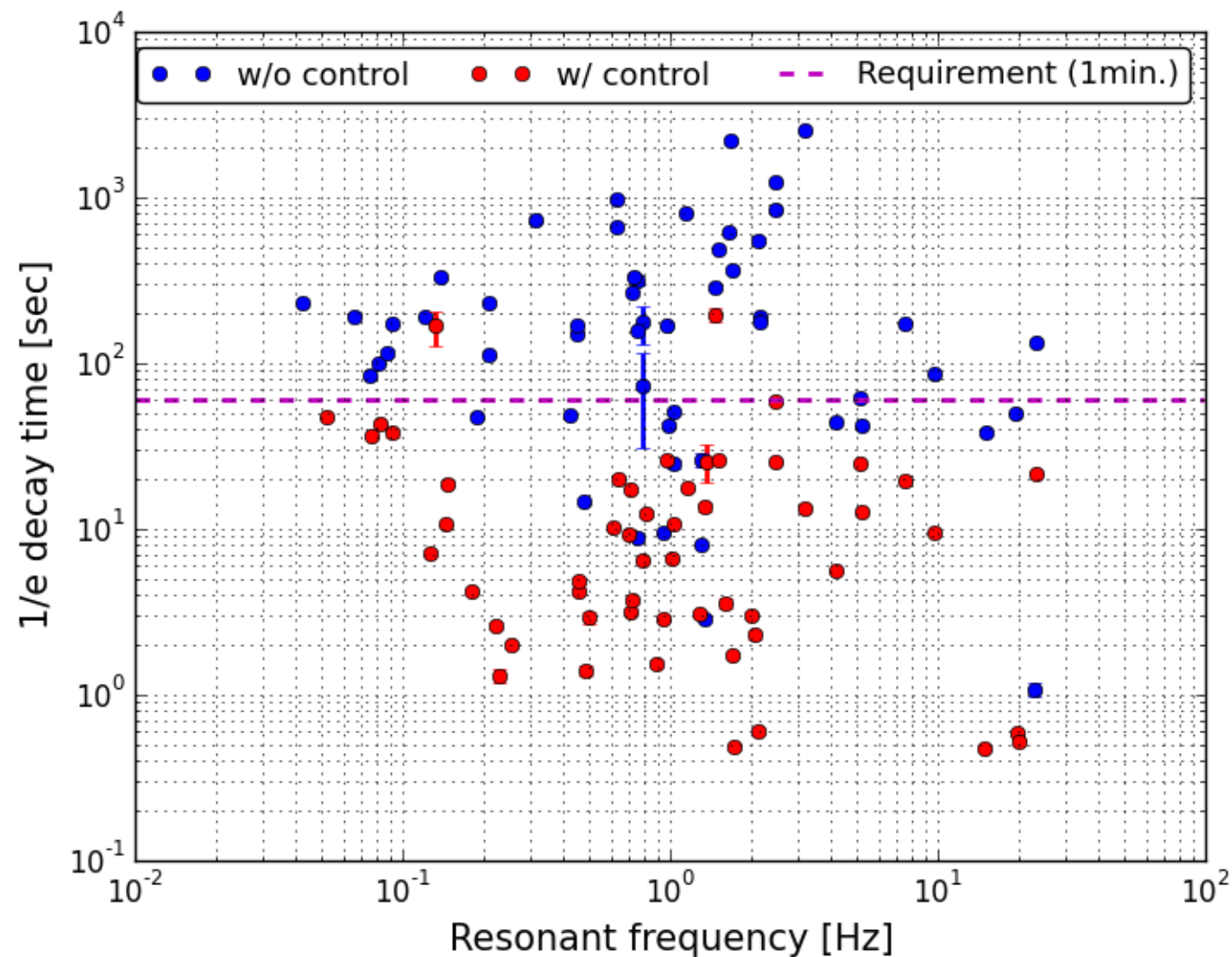
Measureable:
→ For 53 modes

Result: resonant frequency vs. decay time



w/o control

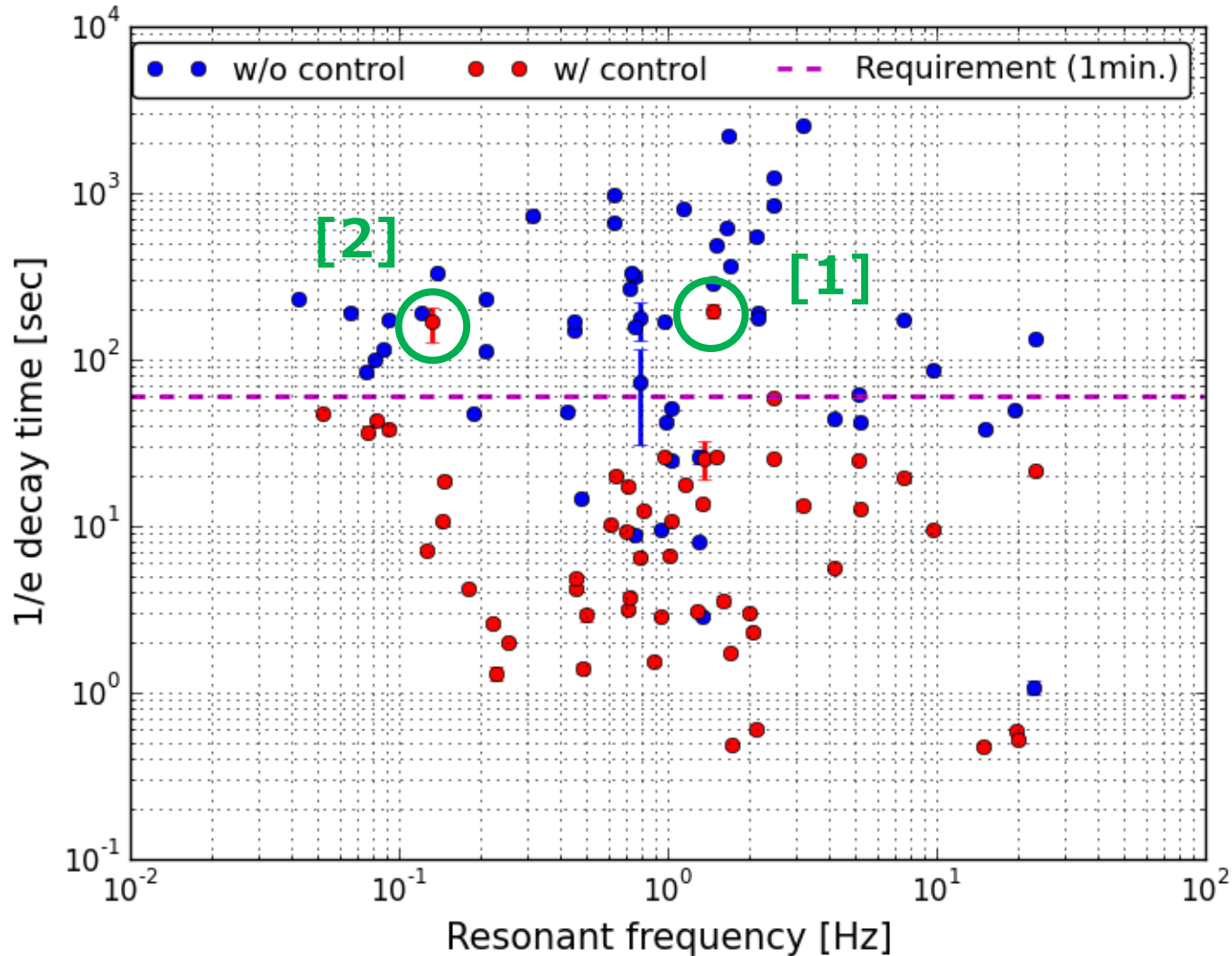
Result: resonant frequency vs. decay time



w/o control

w/ control

Result: resonant frequency vs. decay time

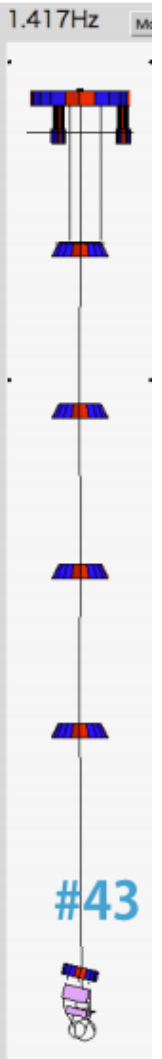


[1] #43: 1.5Hz

Roll / Trans. motion
→ Not disturbed
the lock acquisition.

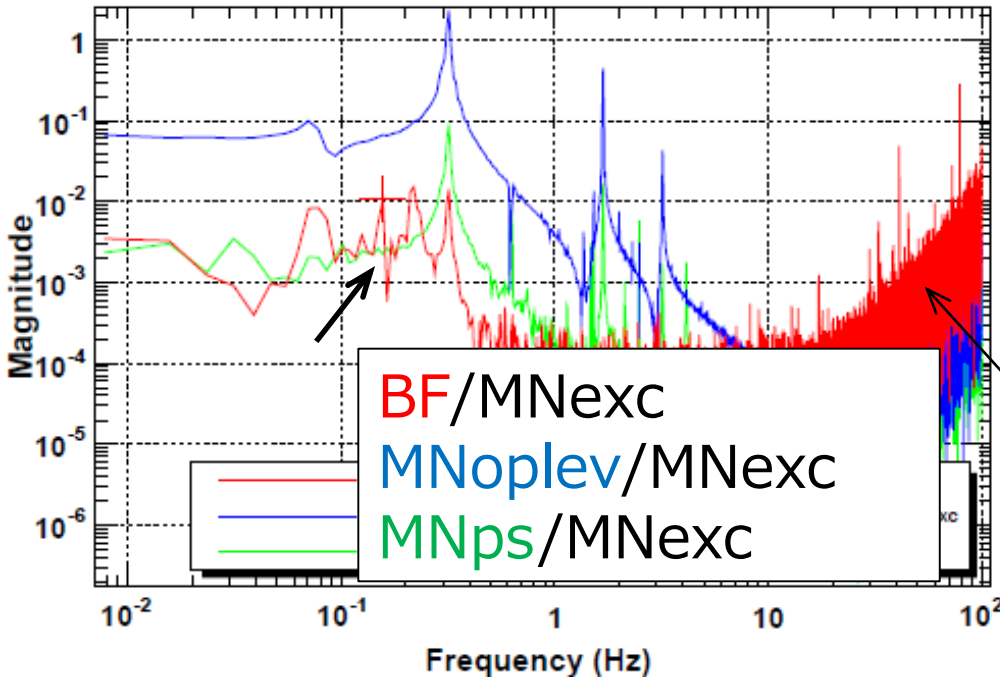
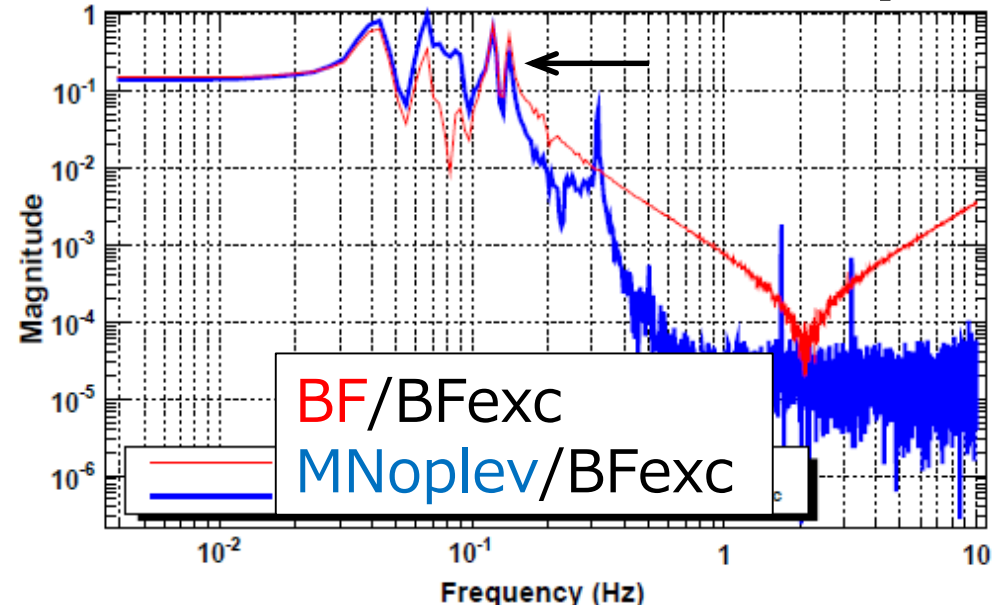
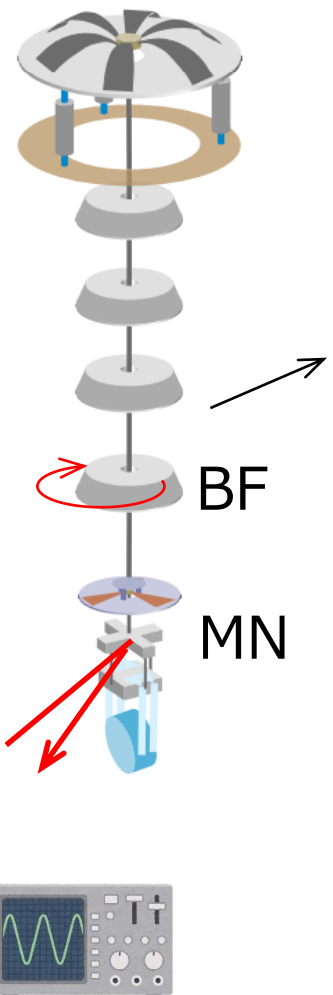
[2] 0.14Hz

unknown Yaw motion
→ failed to predict
w/ the model
w/o heat-link system.
→ visible at BF-stage,
not effectively damped
w/ BF-stage.



Reference mode list: <https://gwdoc.icrr.u-tokyo.ac.jp/DocDB/0078/G1807866/001/sumconTypeA20161114.pdf>

So, is this unknown Yaw mode problematic?

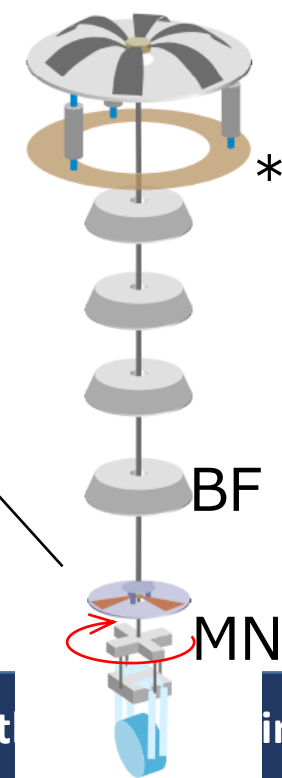


* When white noise injected at BF-stage:
→ The mode looks excited.

* When white noise injected at MN-stage:
→ Not (clearly) excited in TM-chain.

* The decay time ~ 3 min.

→ Not problematic for (current) lock-recovery.

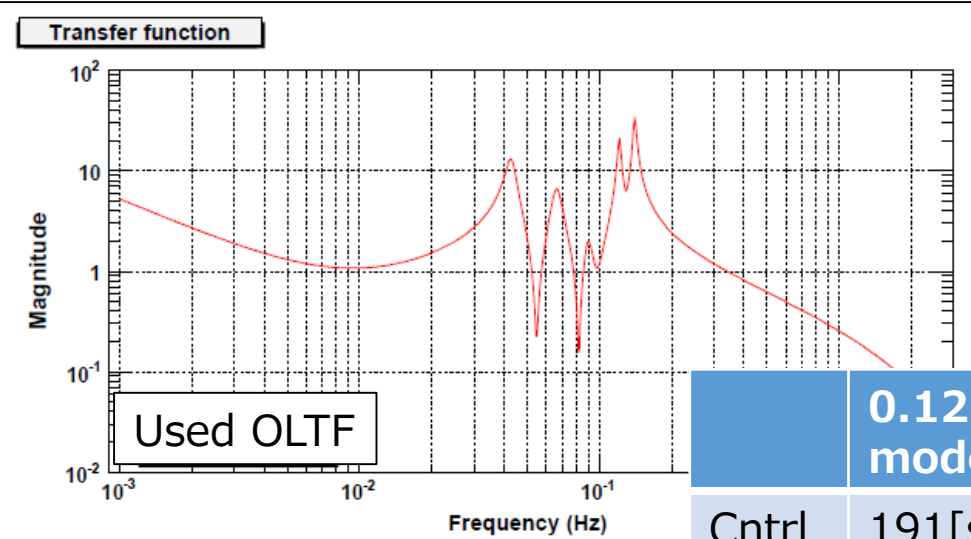
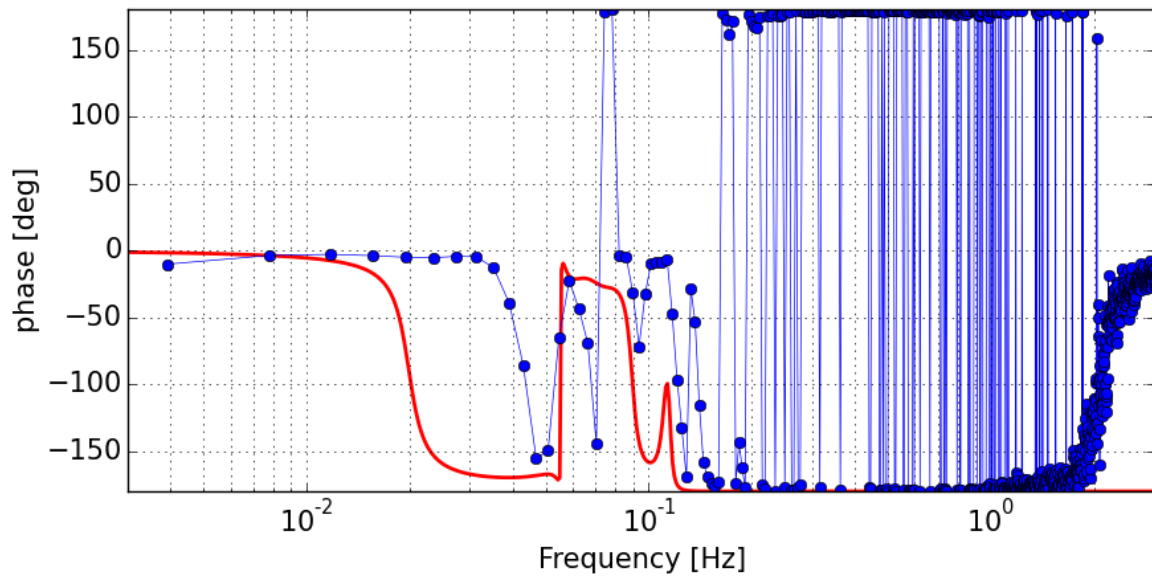
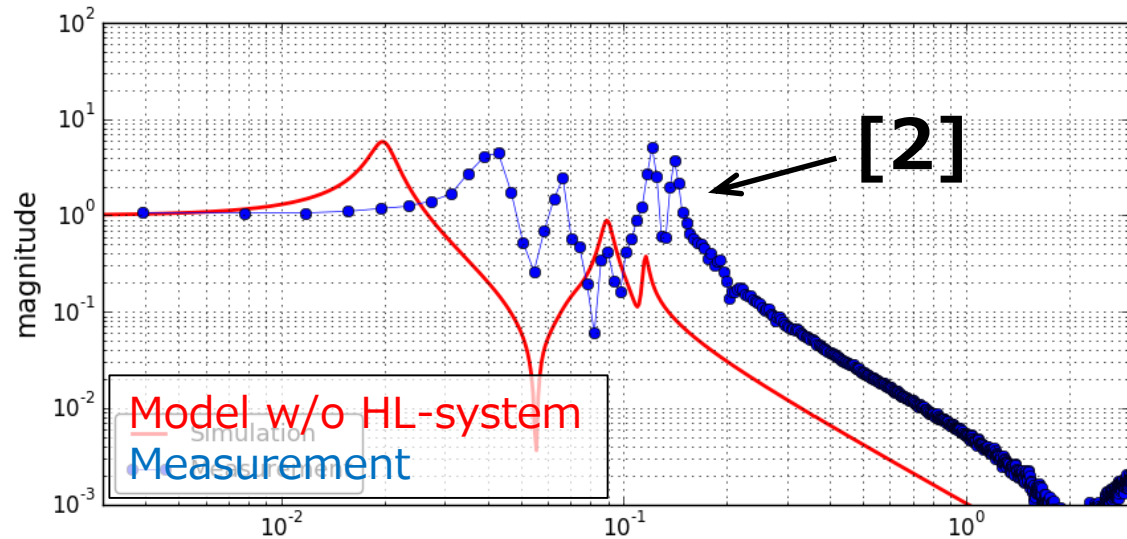


Summary (for damping resonances):

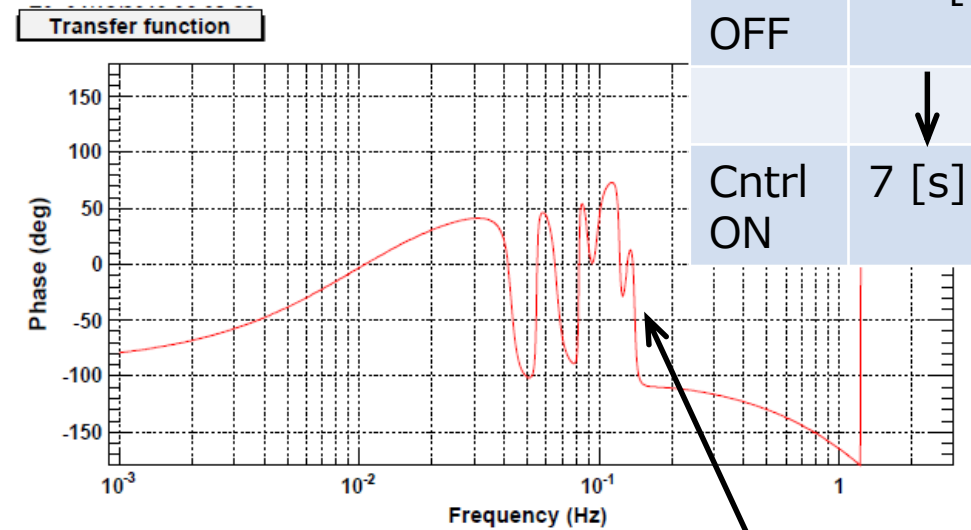
- The installed damping control system damped all the resonances which disturbs the lock acquisition **for the lock-recovery mode.**
- exception: one mode
 - This mode looks from HL-system
 - This would be problematic when upper stages are used for the global controls. → Further improvement
- In this work, the payload damping system is constructed mainly with the optical levers (relatively small linear range).
 - better to utilize Photo-sensors more effectively.

Other notes

unknown Yaw mode [2]?



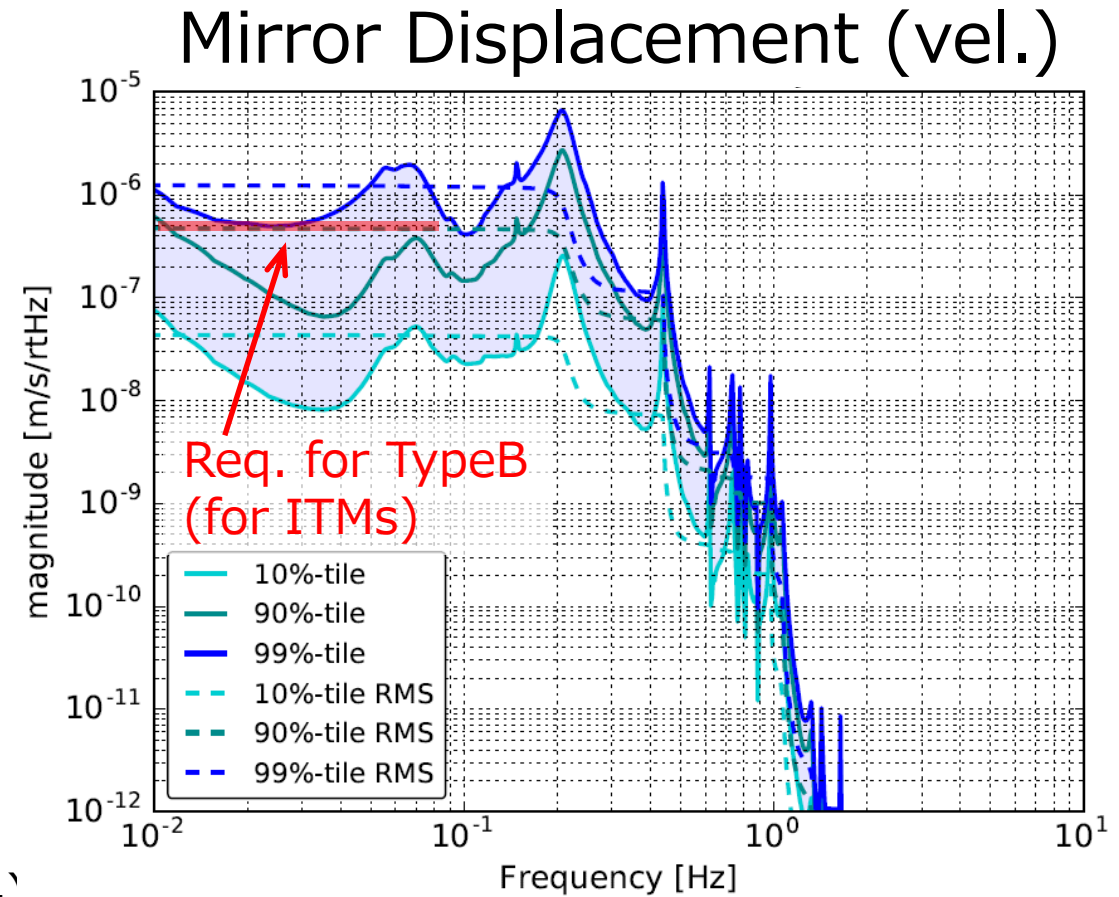
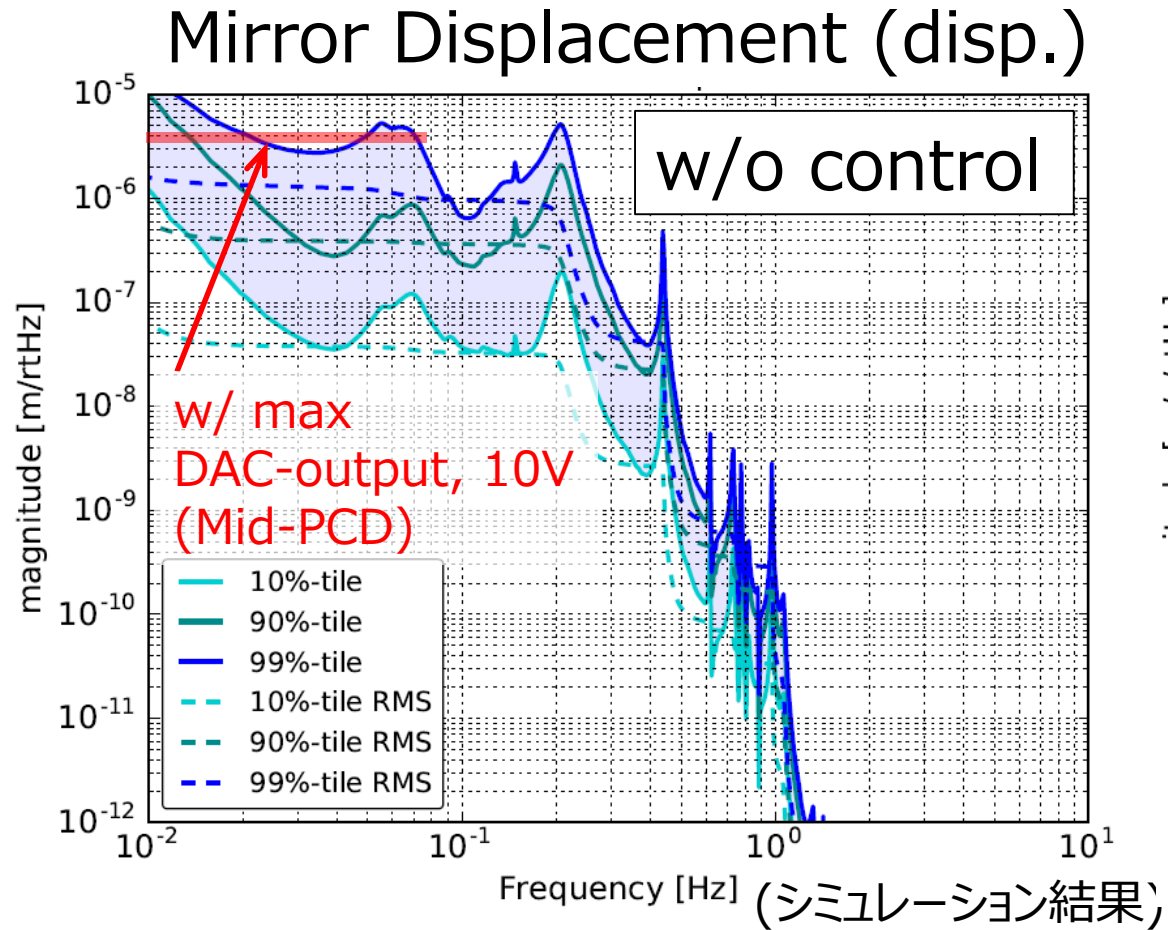
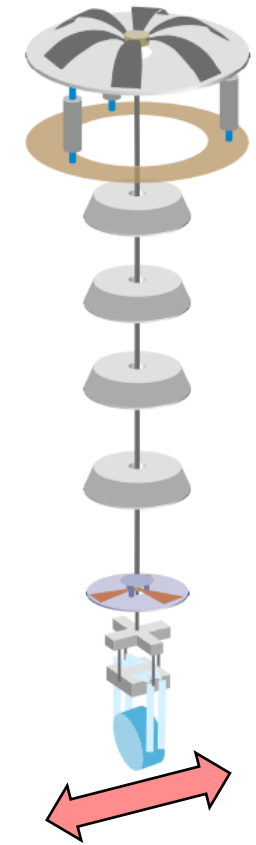
	0.12Hz mode	0.14Hz mode
Cntrl OFF	191 [s]	330 [s]
	↓	↓
Cntrl ON	7 [s]	180 [s]



More finely tuned phase compensation might work

Notes: RMS suppression

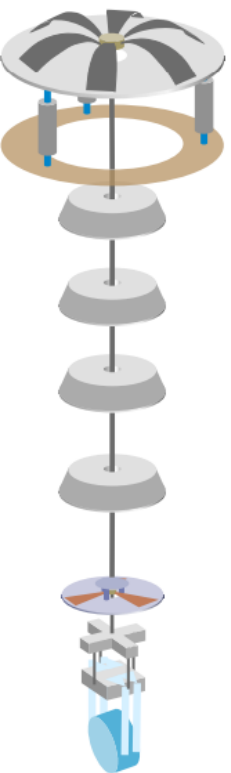
Ground motion and suspension response at KAGRA



Ref: <https://gwdoc.icrr.u-tokyo.ac.jp/cgi-bin/private/DocDB/ShowDocument?docid=10436>

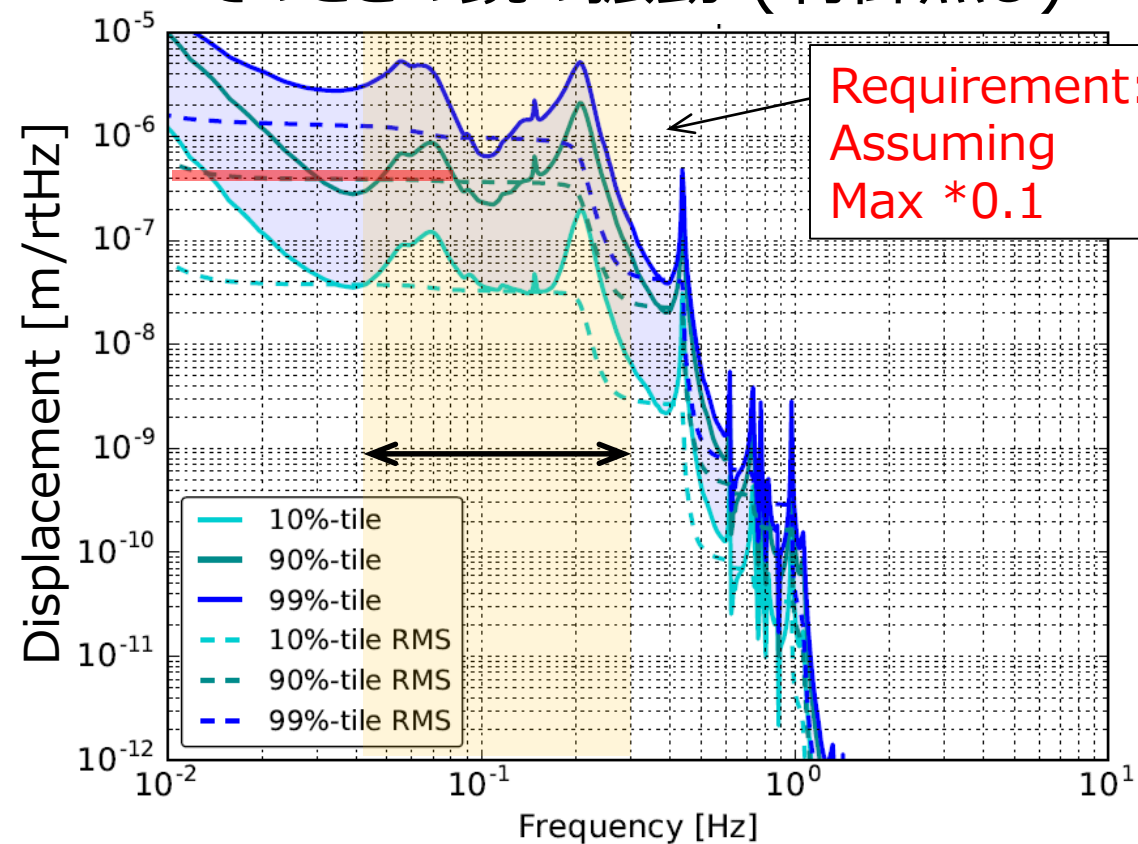
神岡での地面振動レベルと、サスペンションの応答

- 好天候時
→ 要求値を満たせる
- 悪天候時(特に冬場)
→ 要対処 at 上段
- RMSの寄与が大
→ 抑制が必要
→ at 上段

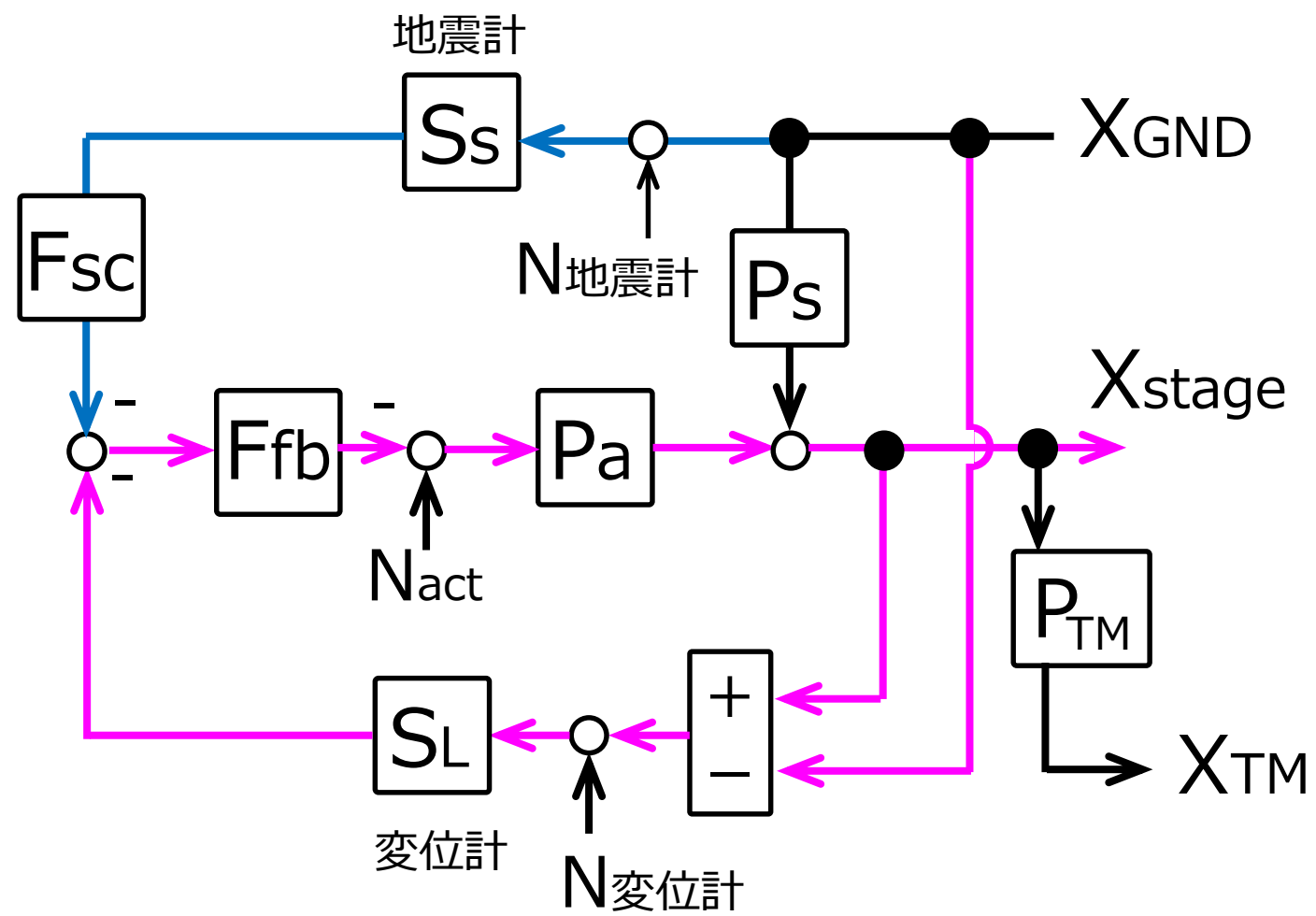
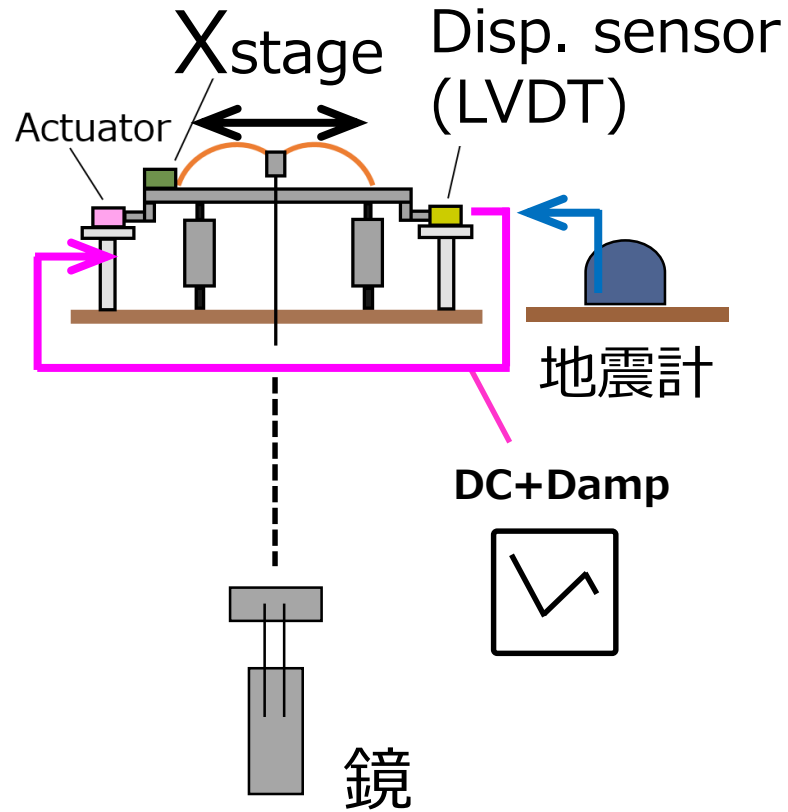


0.2 Hz
mode

そのときの鏡の振動 (制御無し)



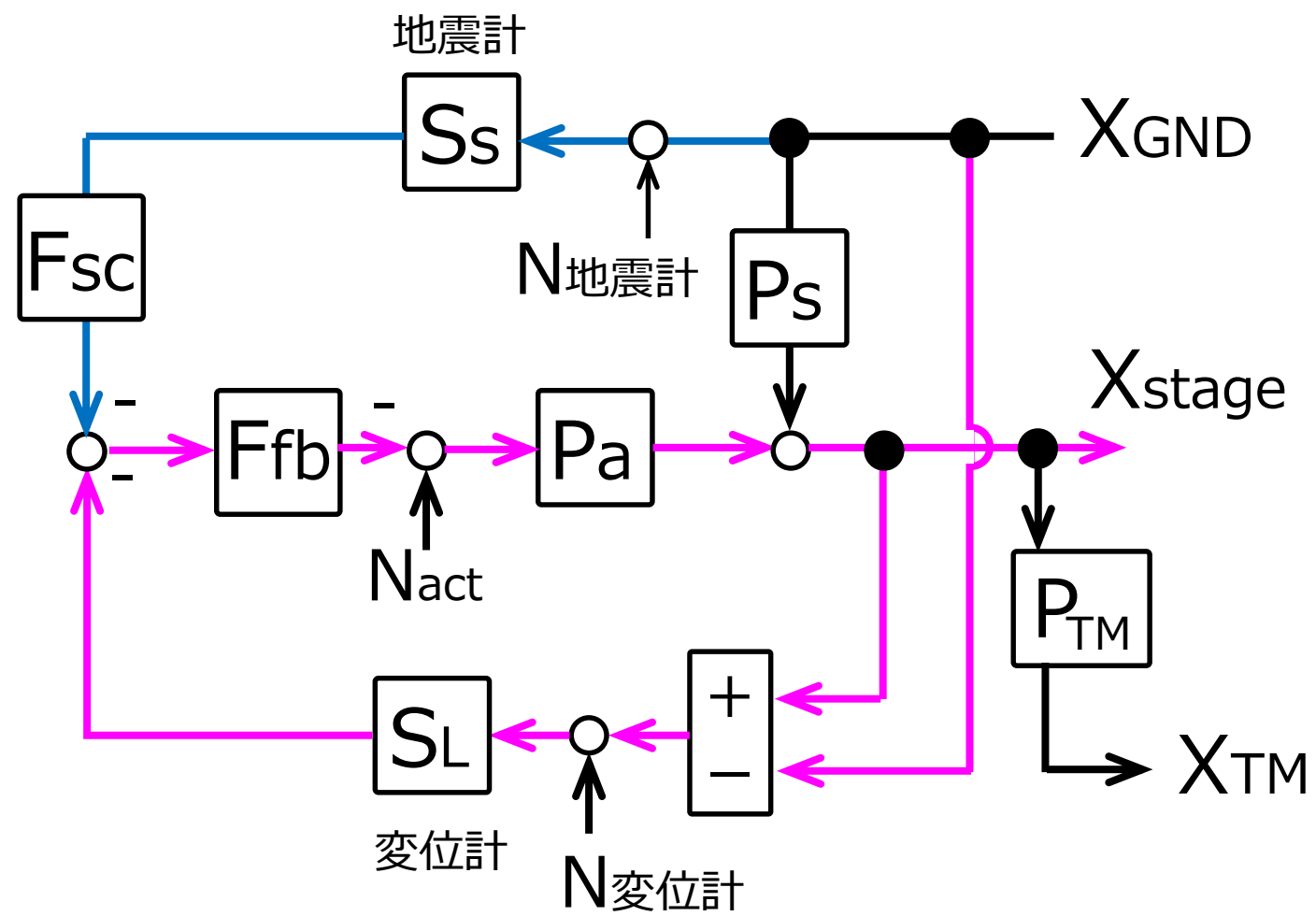
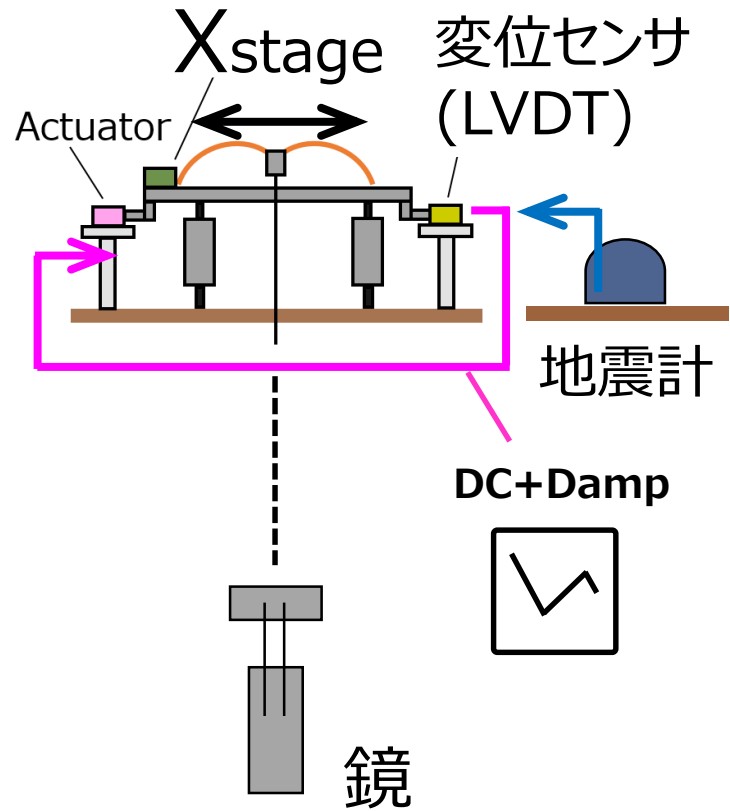
今回考える制御系 (Lのみ):



$$X_{stage} = \frac{1}{1 + G} \left[G \left(1 - F_{sc} \frac{S_s}{S_L} \right) + P_s \right] X_{GND}$$

$$(G = P_a F_{fb} S_L)$$

今回考える制御系 (Lのみ):



$$F_{sc} = S_L/S_s \text{ にて}$$

:

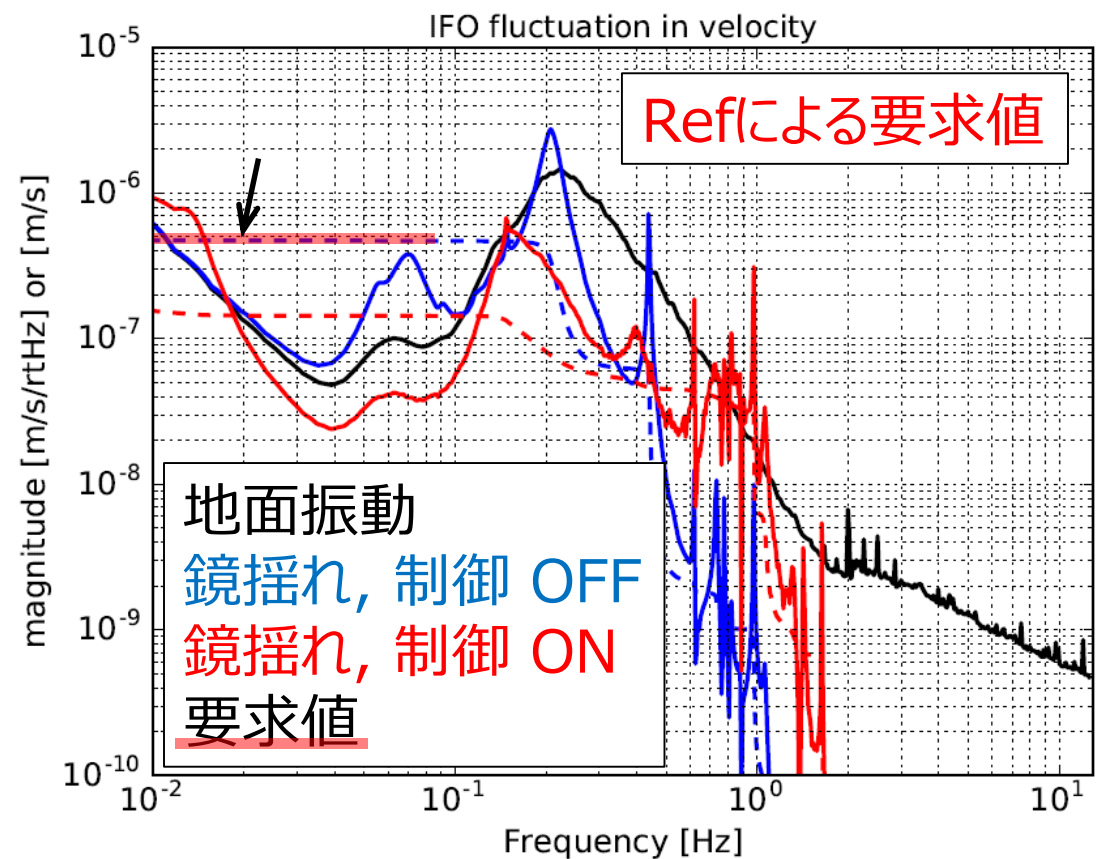
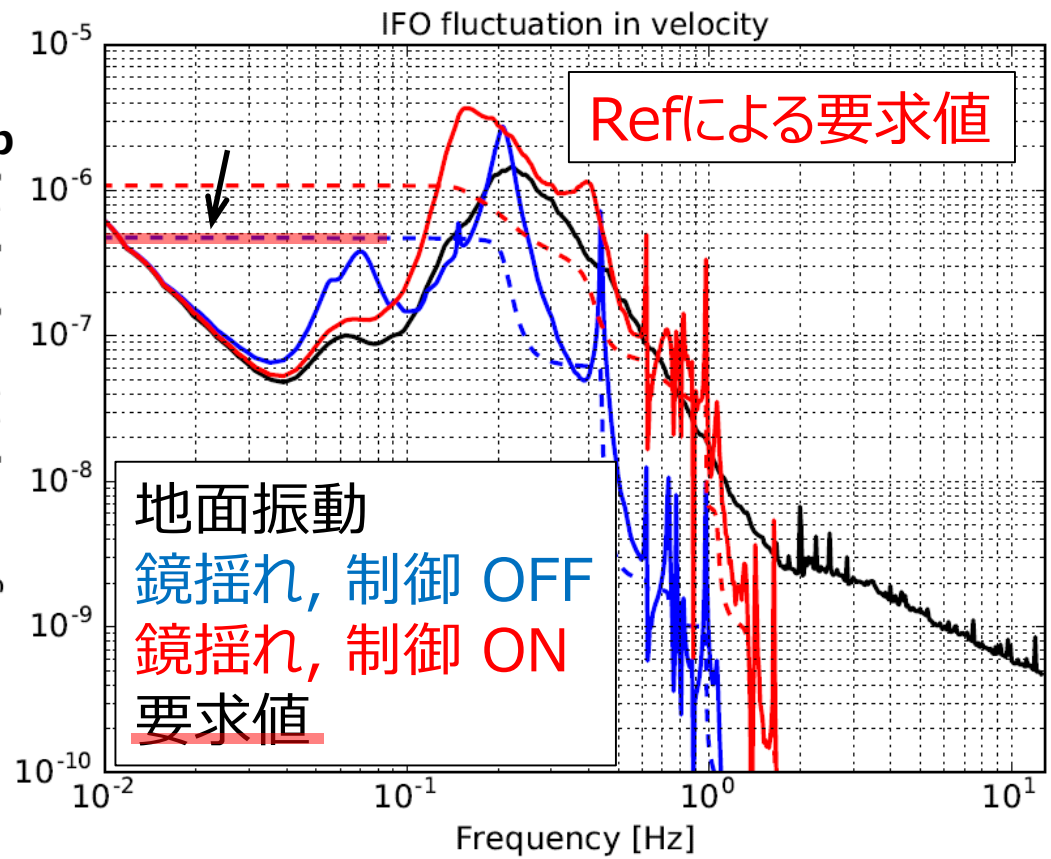
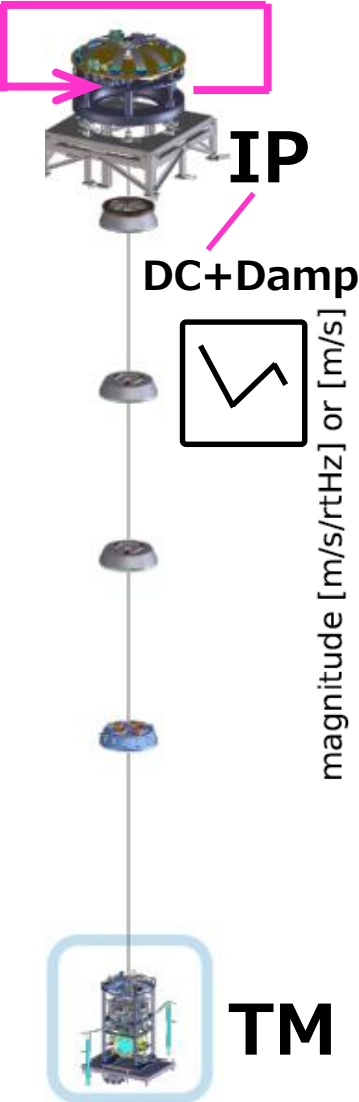
$$X_{stage} = \frac{1}{1 + G} \left[G \left(1 - F_{sc} \frac{S_s}{S_L} \right) + P_s \right] X_{GND}$$

Cut the seismic noise injection via LVDT
→ Sensor correction

シミュレーション: 90%tile 地面振動を仮定のときの、鏡揺れ(速度)

Sensor correction **OFF**

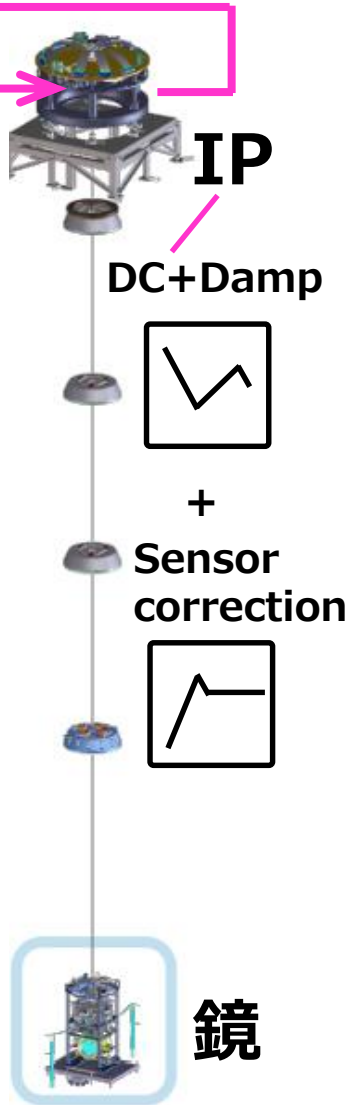
Sensor correction **ON**



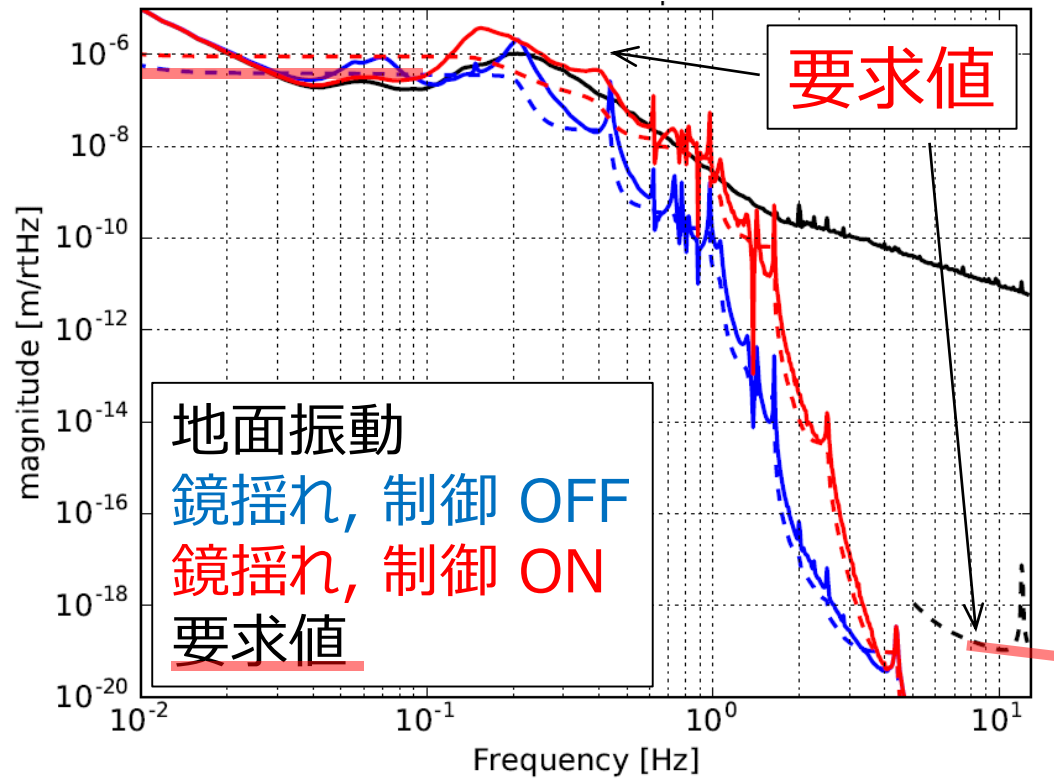
→ Sensor correction を用いて、要求値を満たせる。

Ref: p.55 in <https://gwdoc.icrr.u-tokyo.ac.jp/cgi-bin/private/DocDB/ShowDocument?docid=6060>,
or p.90 in <https://gwdoc.icrr.u-tokyo.ac.jp/cgi-bin/DocDB/ShowDocument?docid=4155>

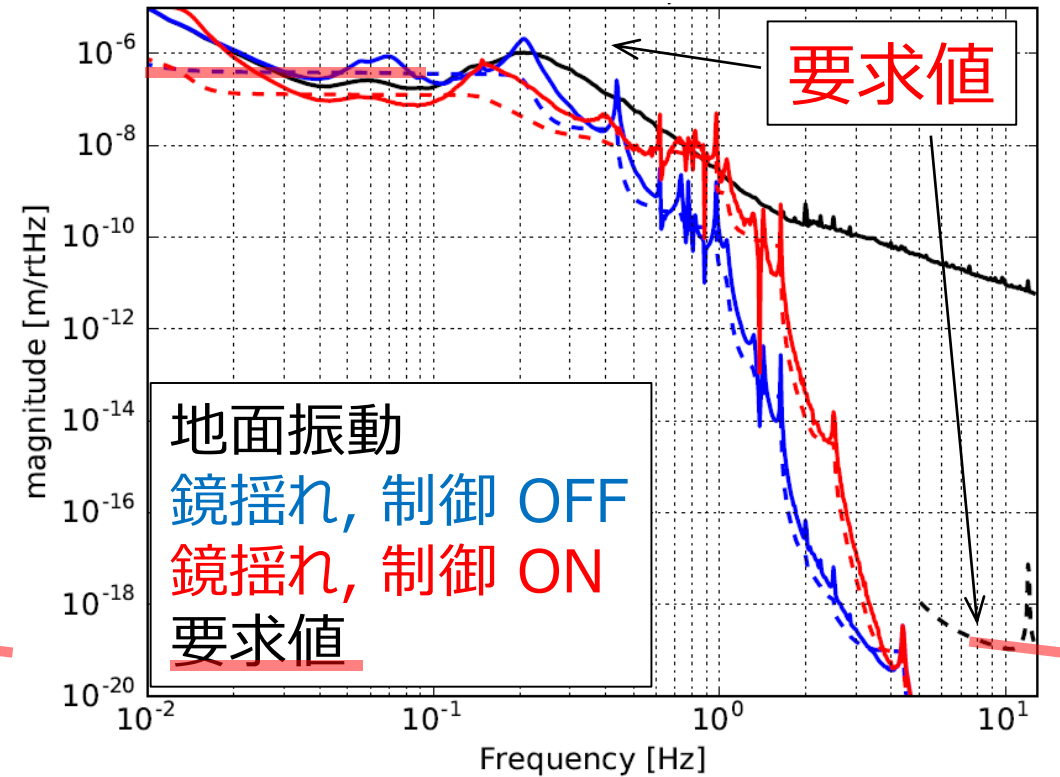
シミュレーション: 90%tile 地面振動を仮定のときの、鏡揺れ



Sensor correction OFF

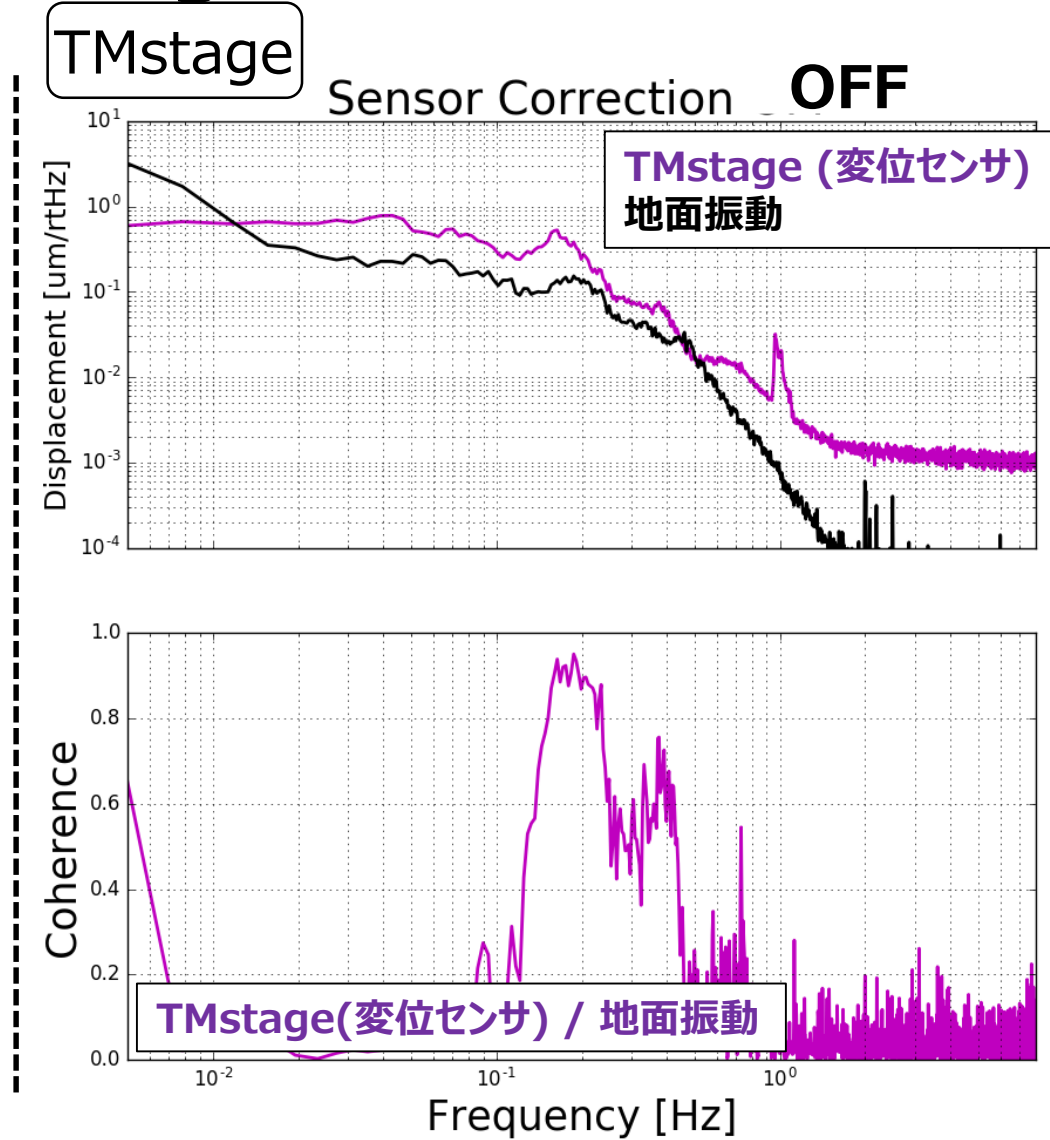
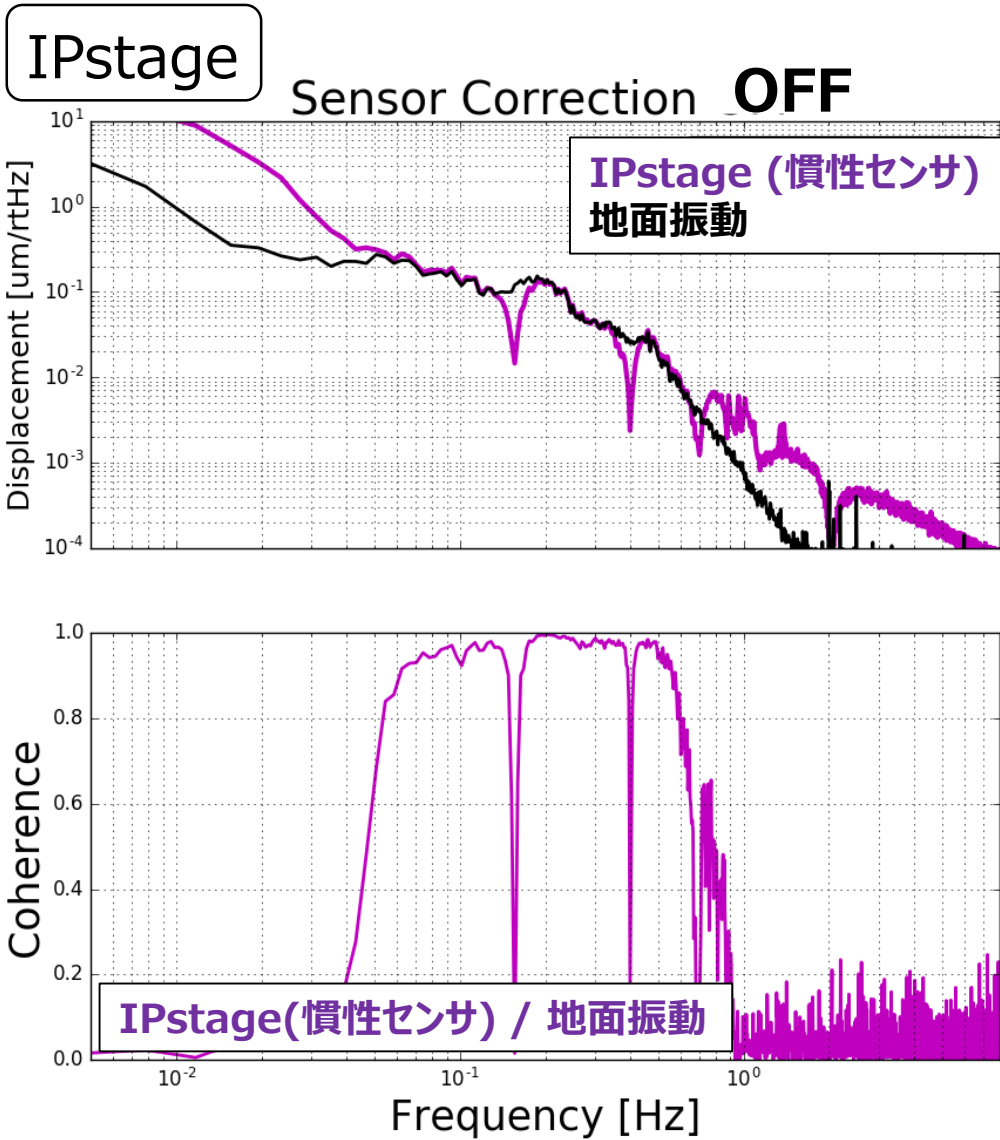
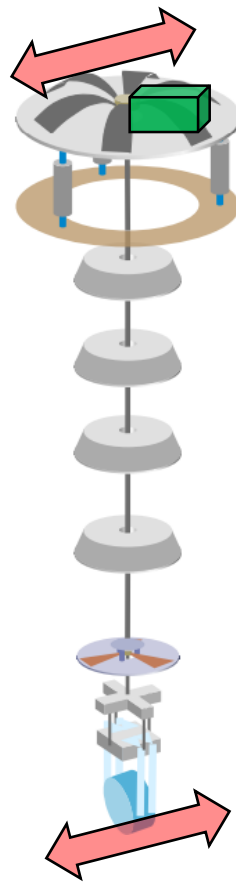


Sensor correction ON

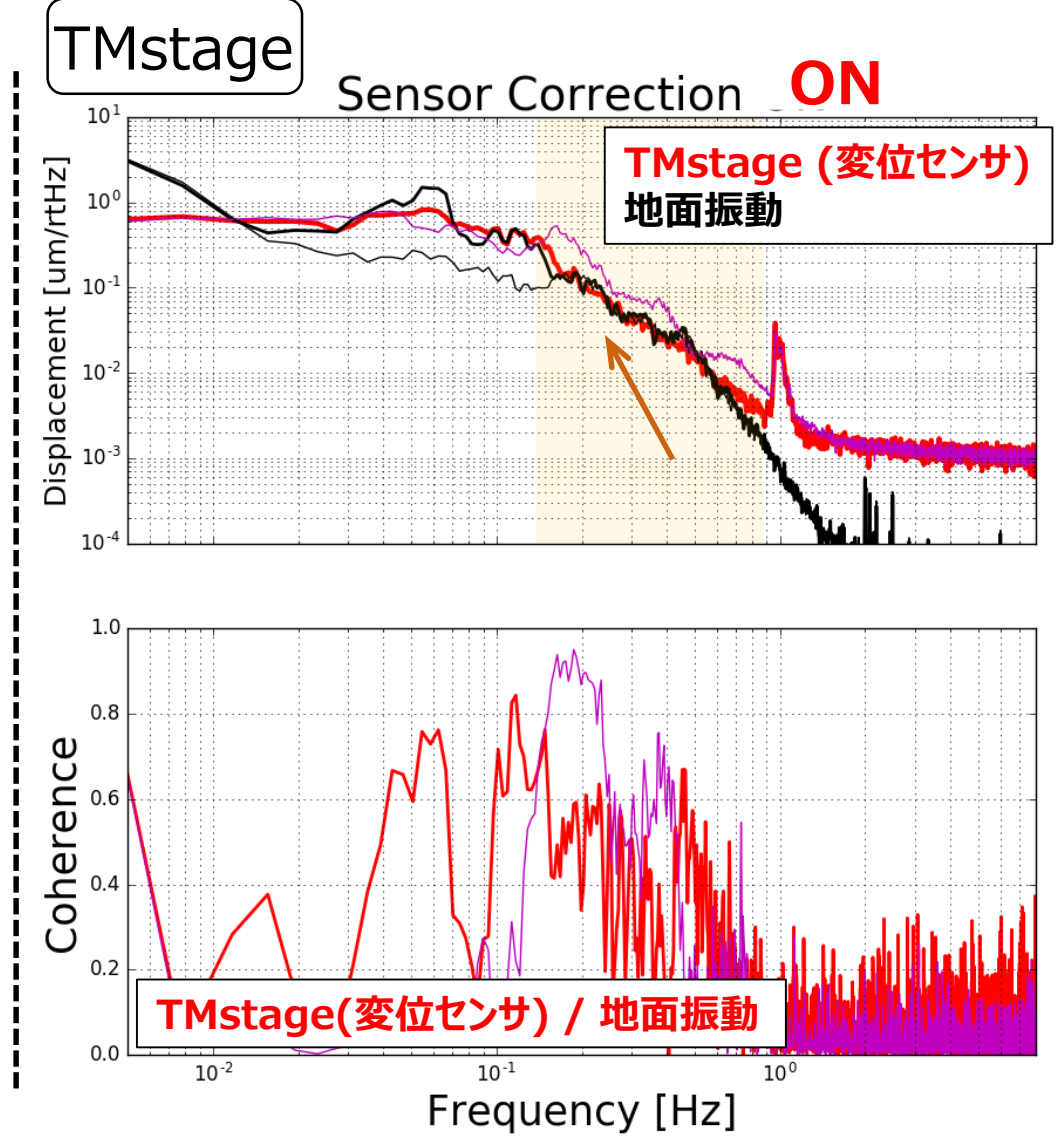
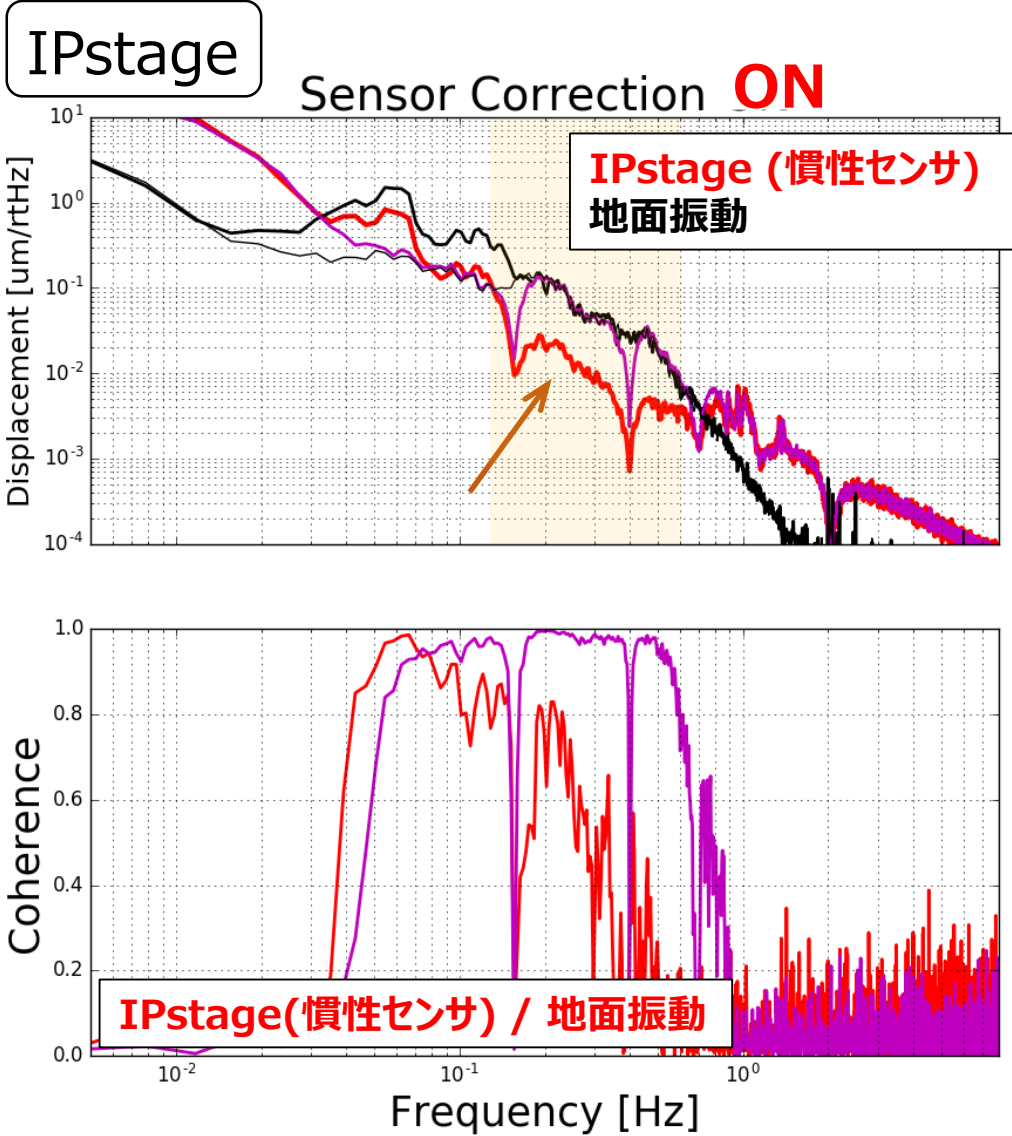
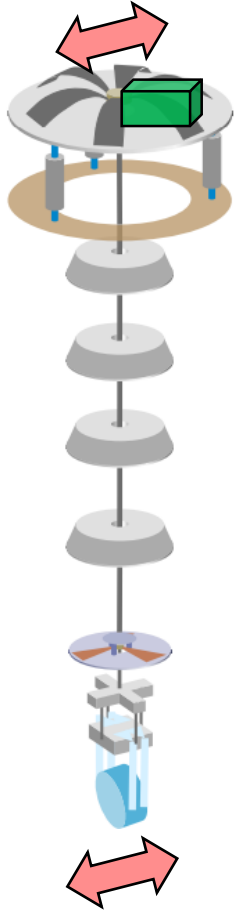


→ Sensor correction を用いて、要求値を満たせる。

テスト実装: 1つのサスペンションのIP-stageに実装

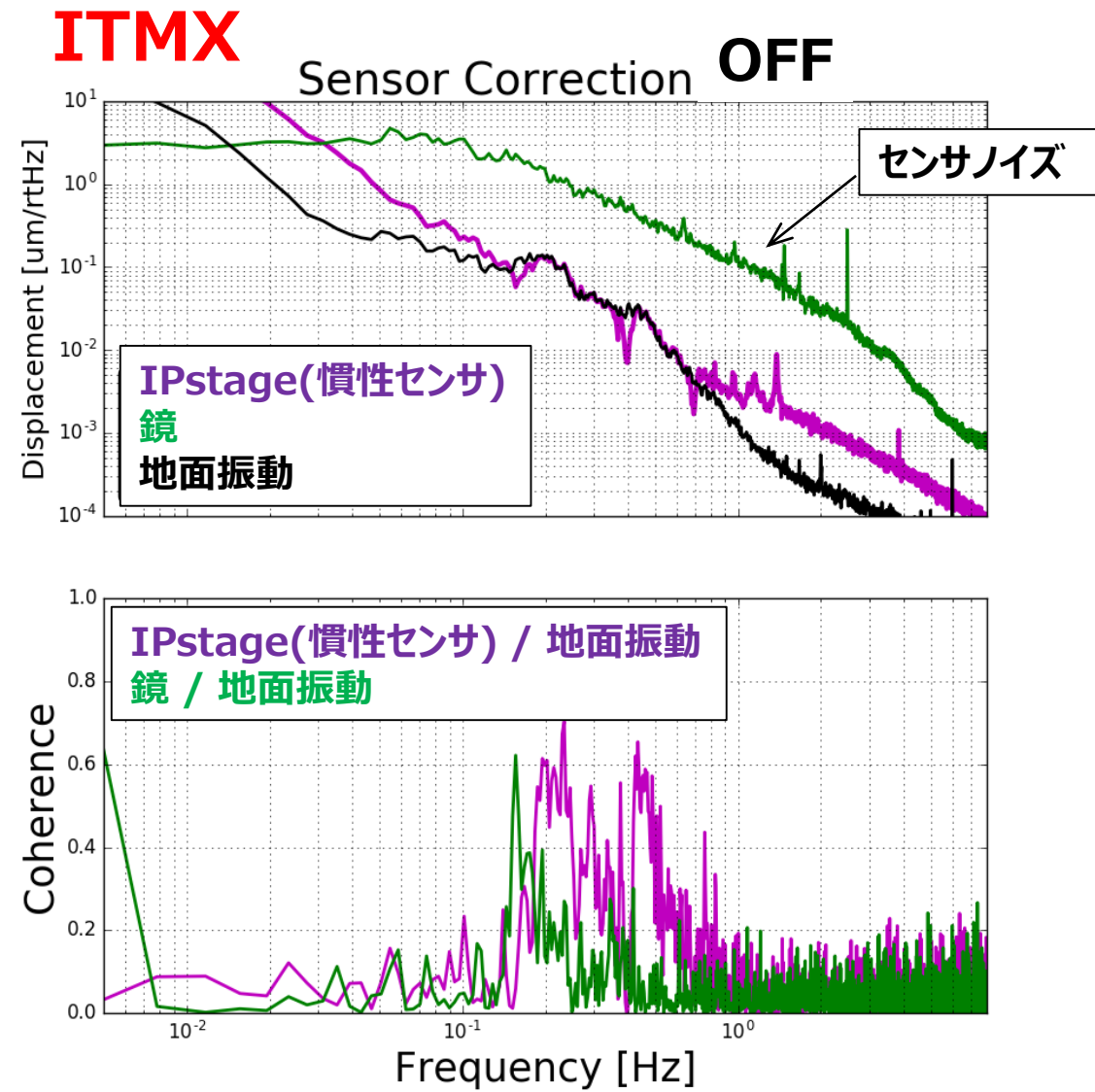
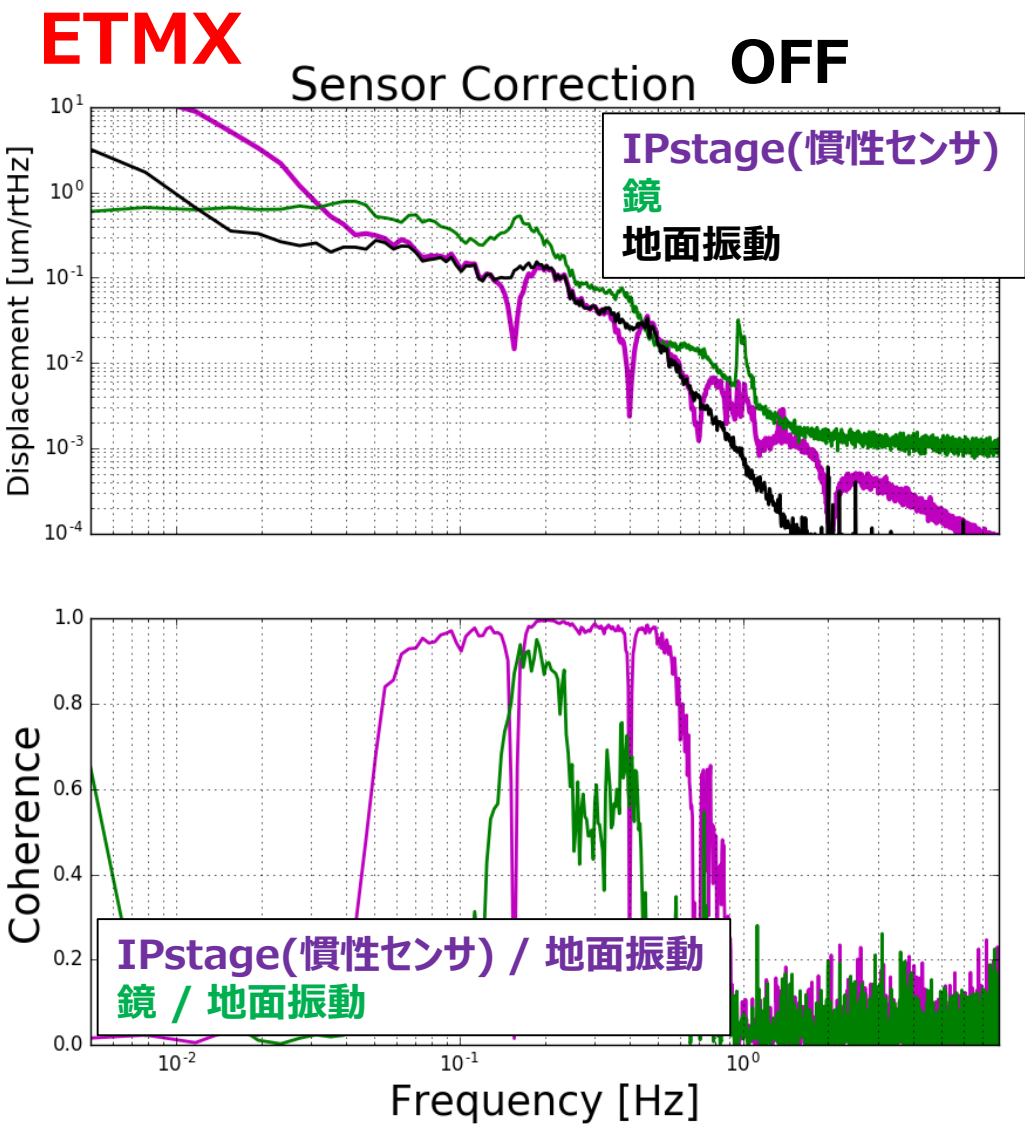
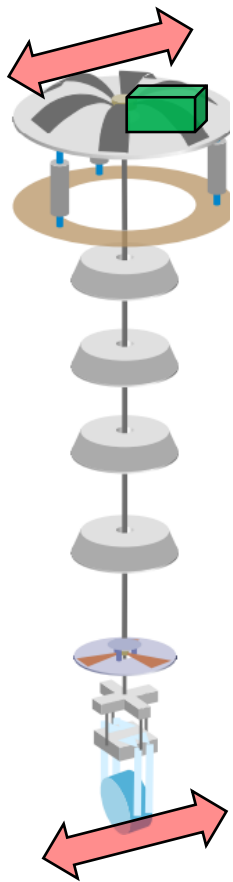


テスト実装: 1つのサスペンションのIP-stageに実装

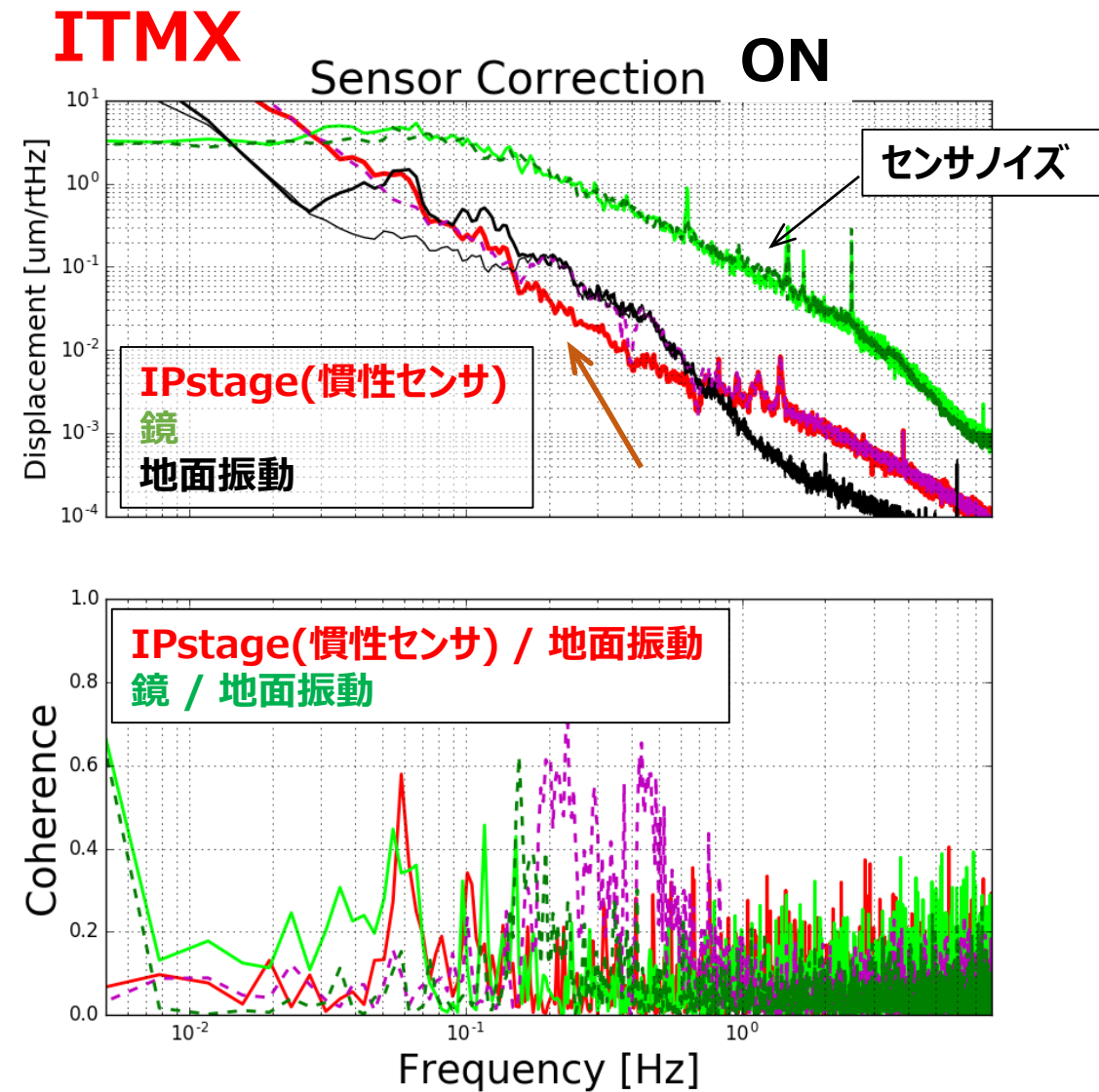
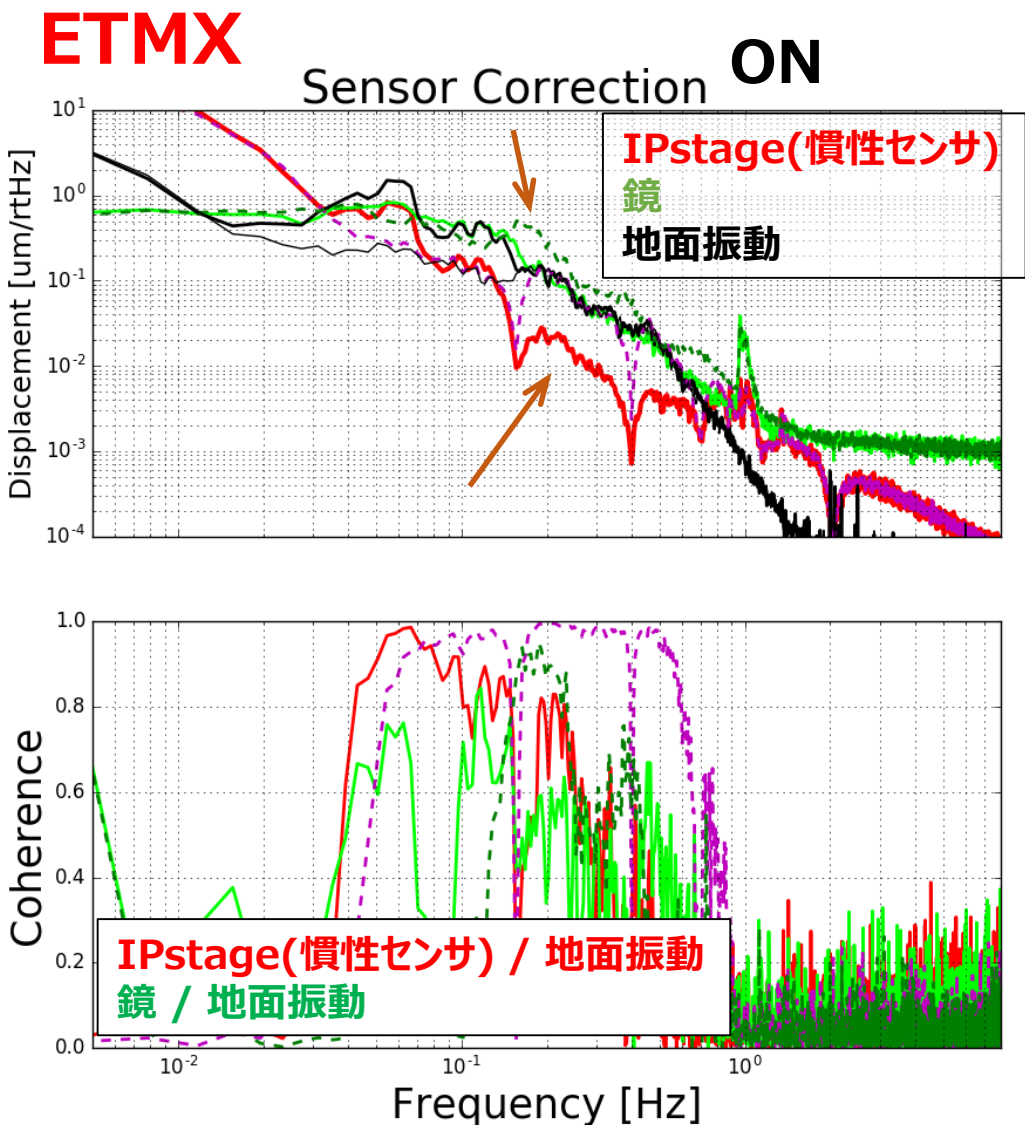
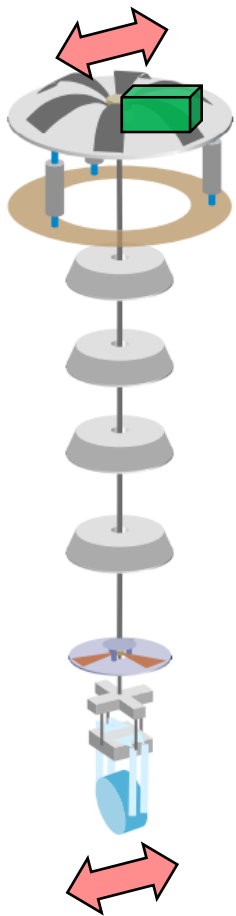


→ SC worked at 0.15 ~ 0.7Hz (only).

テスト実装: 2つのサスペンションのIP-stageに実装、localで見ると

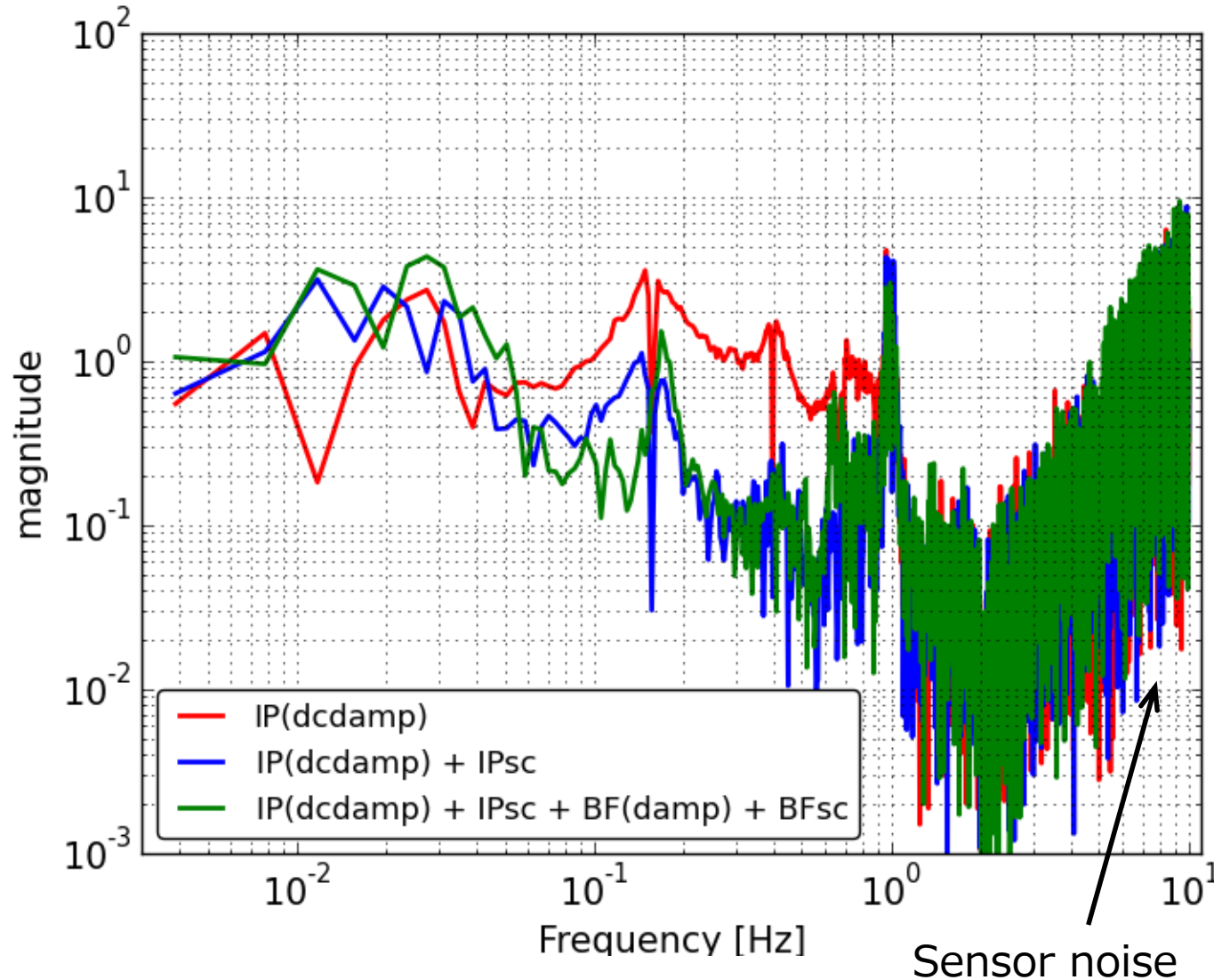


テスト実装: 2つのサスペンションのIP-stageに実装、localで見ると



→ 0.12 - 0.7Hz で揺れが低減された様子。詳細はセンサノイズ/地面振動に隠れた。

By combining $(TML/IPL) * (IPL/GNDL)$



Displacement TF:
From GND to TML

(measured at ETMX)

Is notch real?

→ Not sure.

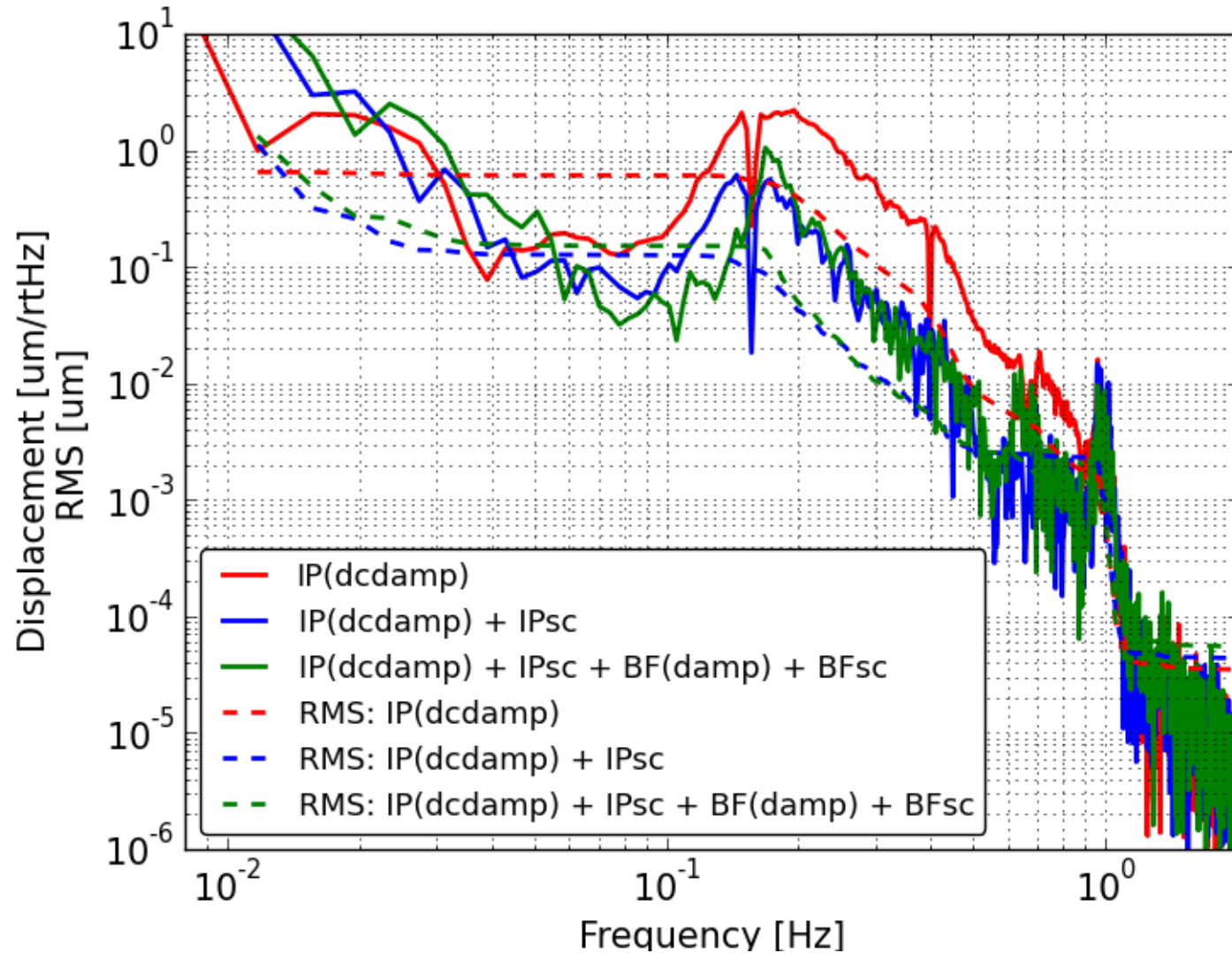
→ tilt coupling from Geophone

Or

→ Actual mechanical response

Whichever, the conclusion is same.

By combining (TML/IPL)*(IPL/GNDL)*(GNDL_model)

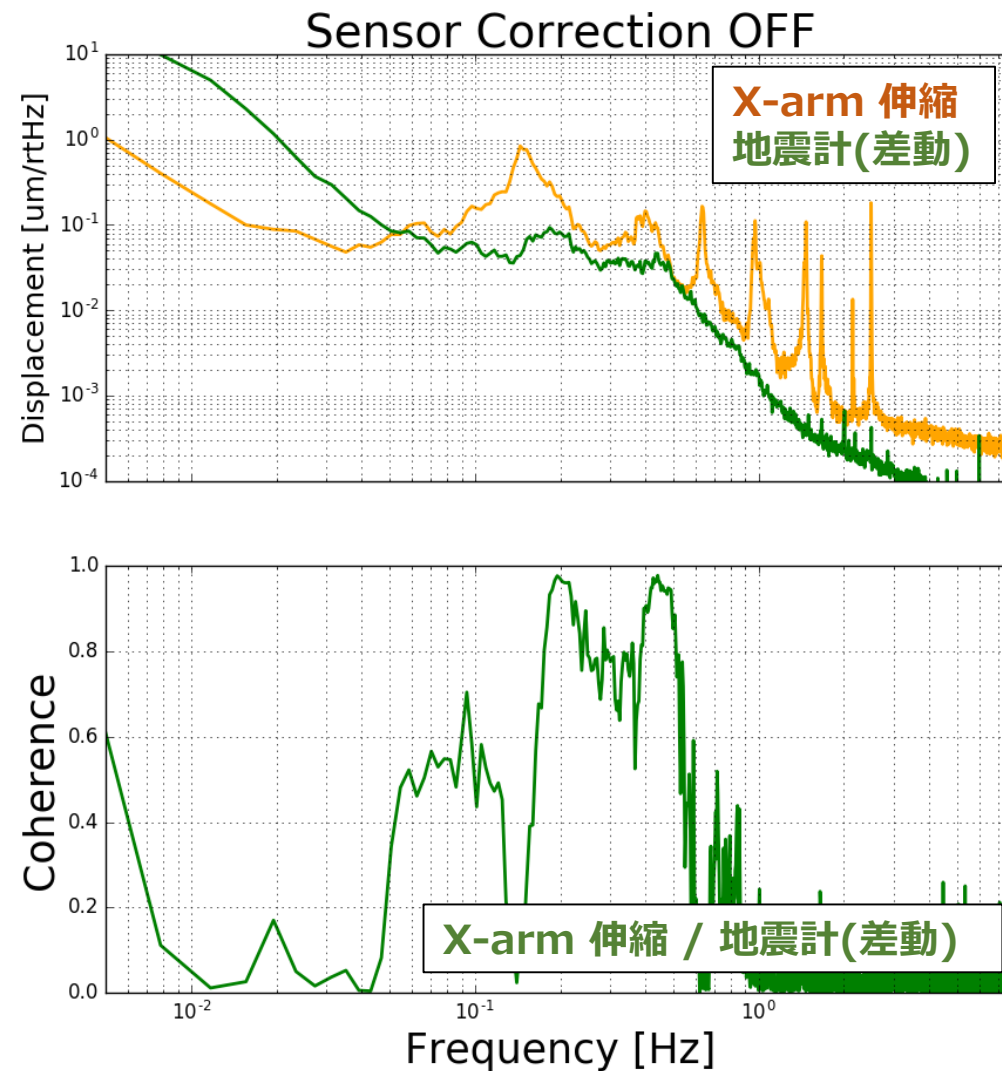
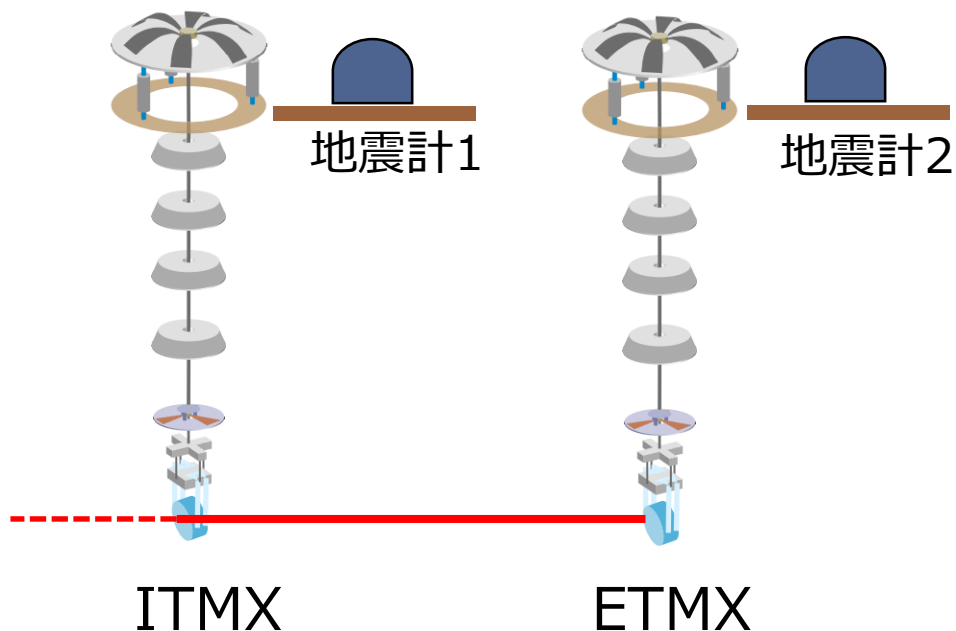


Disp. TF:

From GND to TML

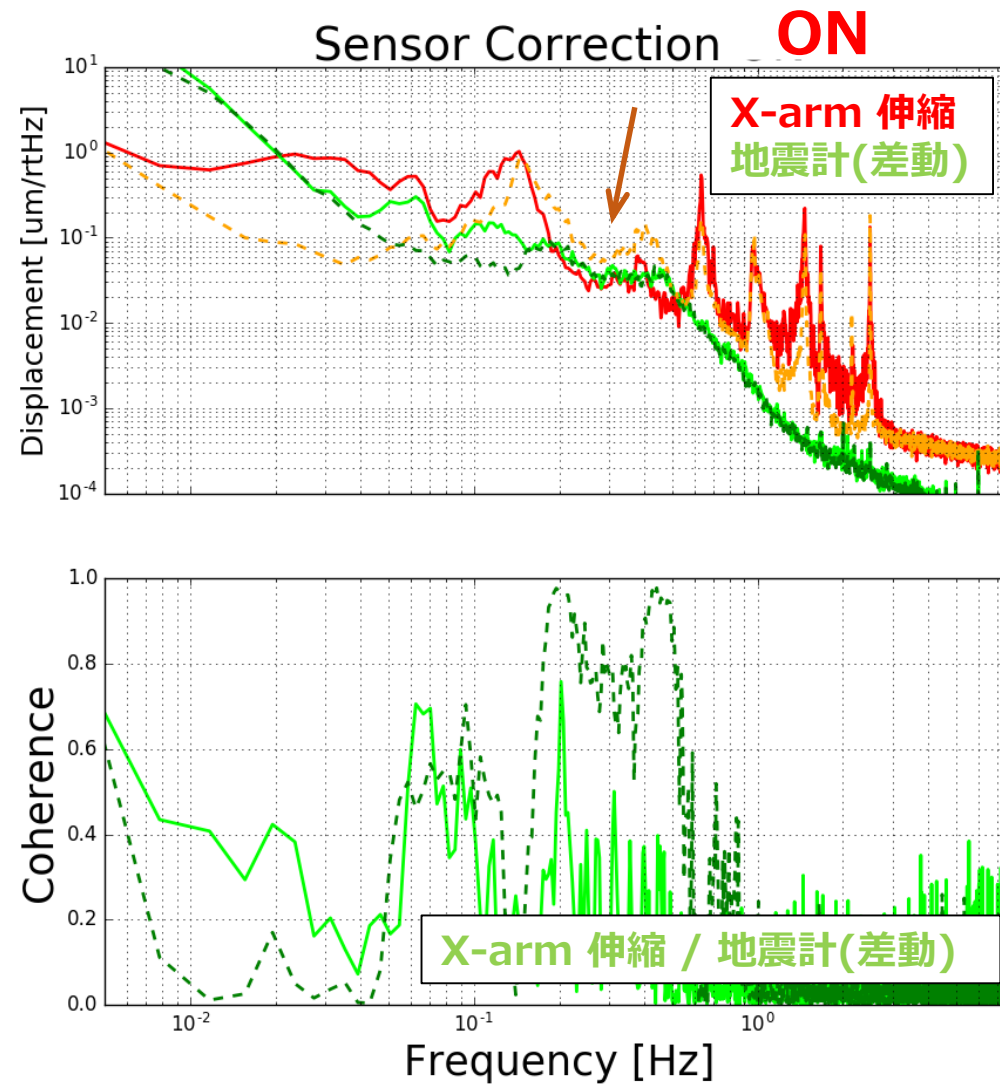
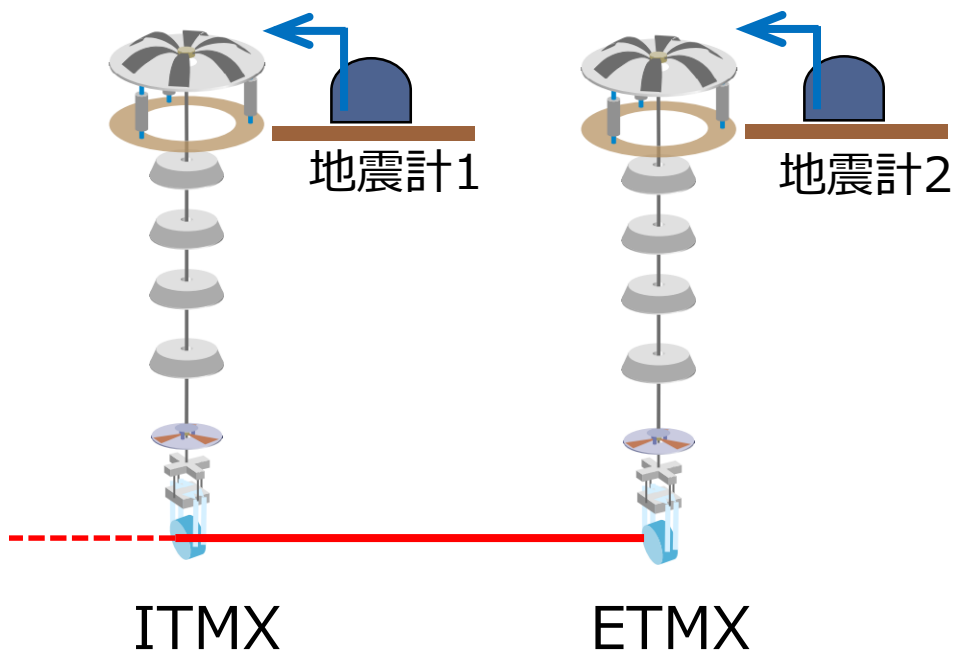
テスト実装: 2つのサスペンションのIP-stageに実装、Xarm で見ると

- ETMX と ITMX に実装
- 0.15 ~ 0.7 Hzにて同様な改善あり
- X-arm (FP cavity) の共振器長の確認



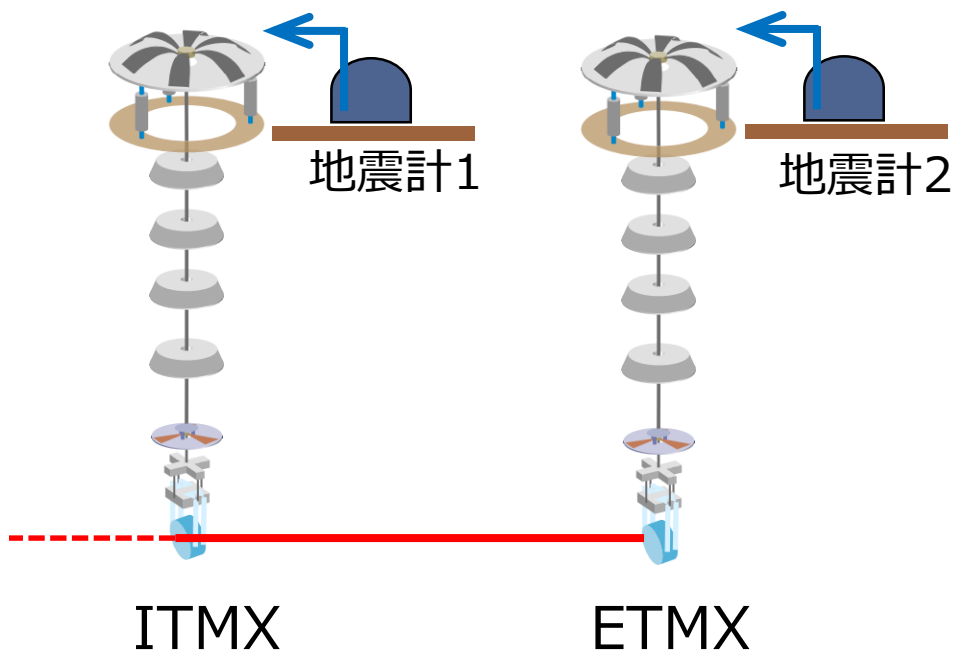
テスト実装: 2つのサスペンションのIP-stageに実装、Xarmで見ると

→ X-arm (FP cavity) の共振器長では

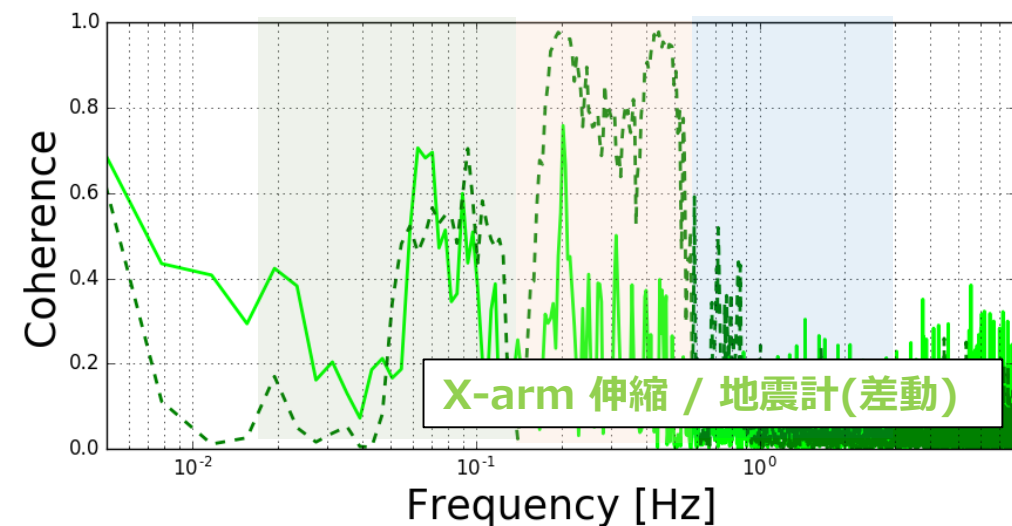
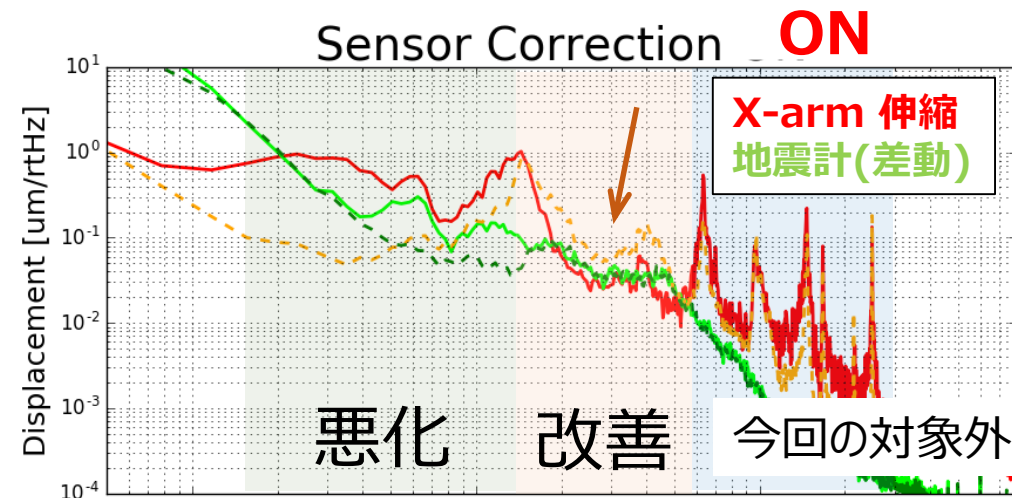


テスト実装: 2つのサスペンションのIP-stageに実装、Xarm で見ると

→ X-arm (FP cavity) の共振器長では



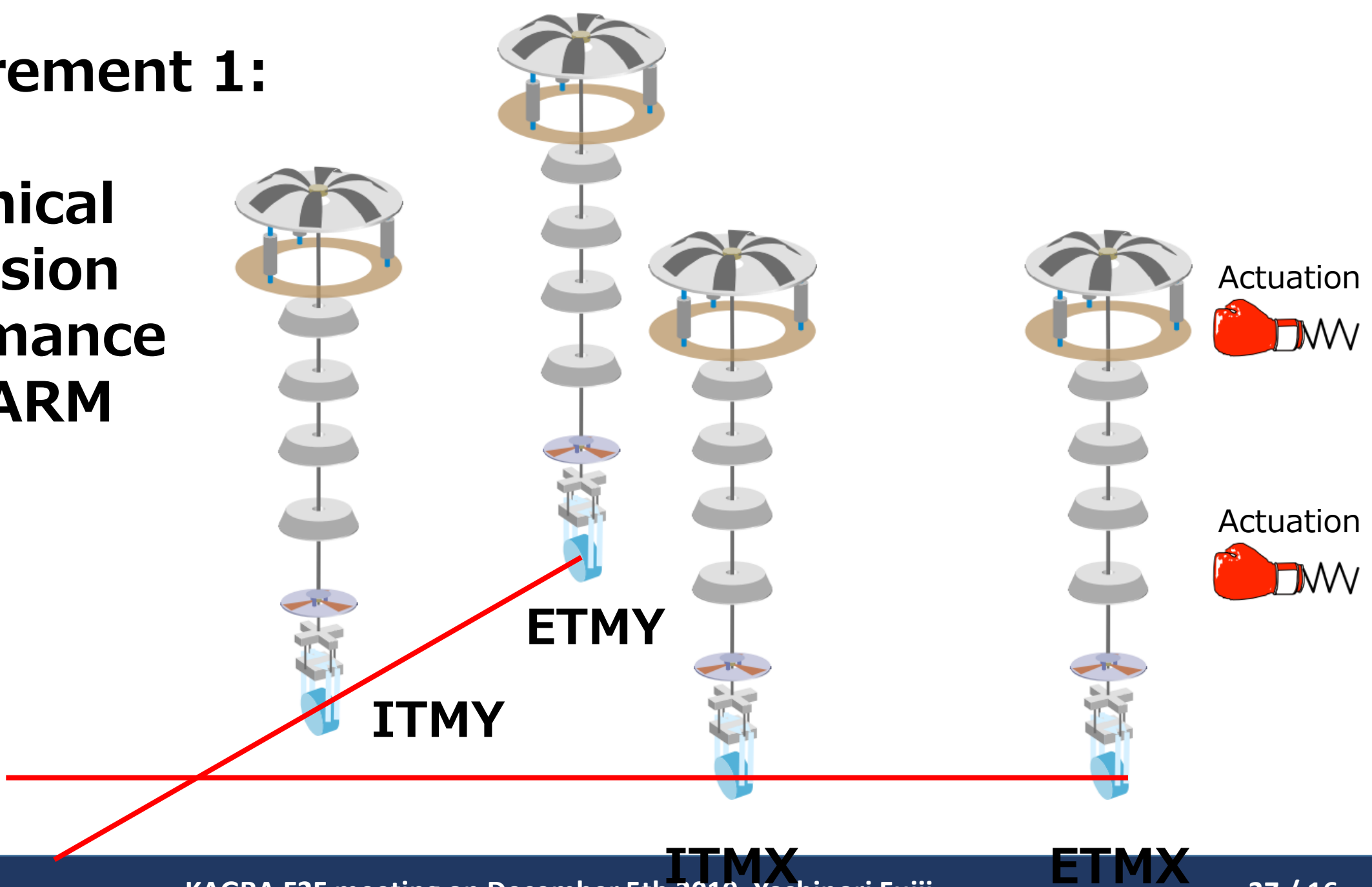
- SC worked at 0.15 ~ 0.6Hz.
- 低周波: 悪化
 - SC filterからの地震計の雑音の流入ほか
 - To be investigated.



Verification of suspension performance

Measurement 1:

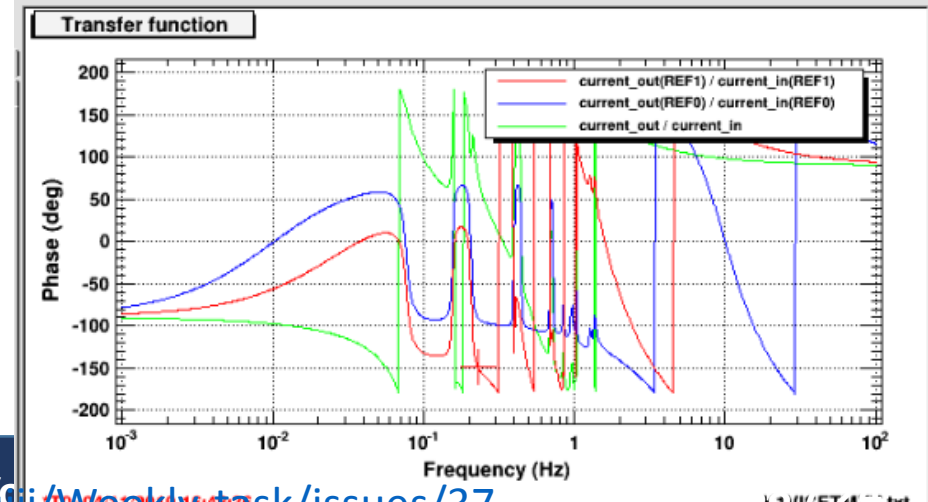
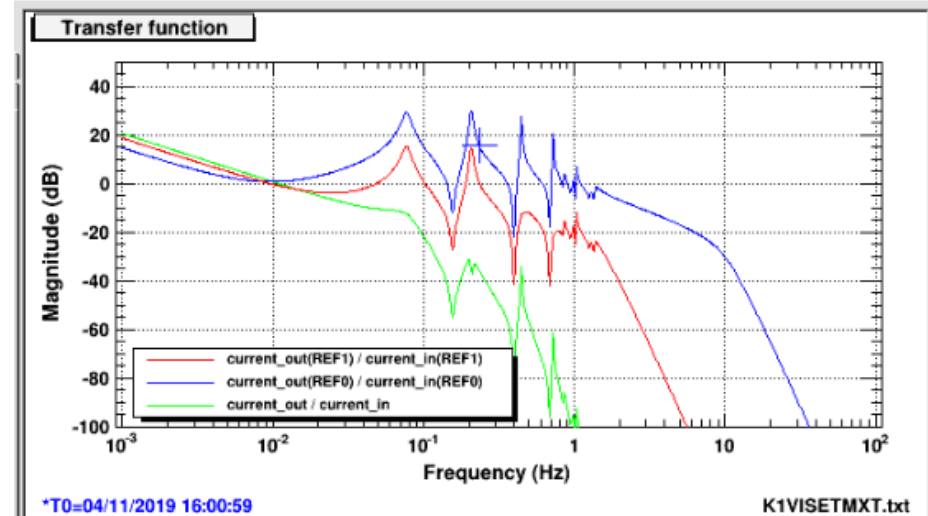
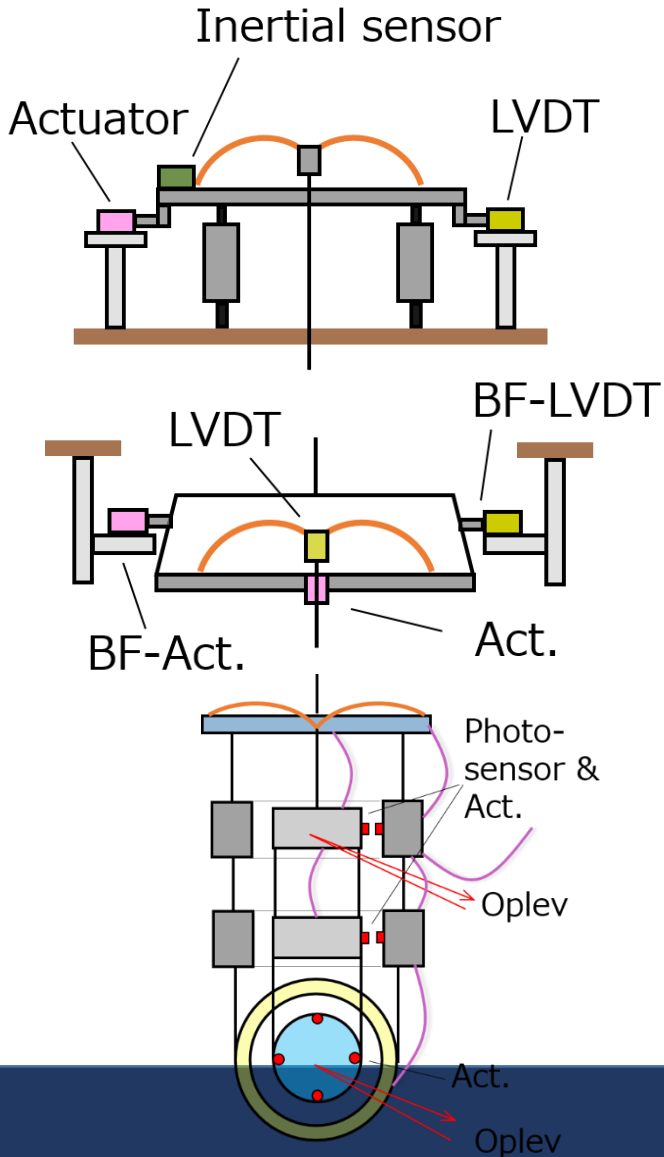
Mechanical suspension performance with DARM



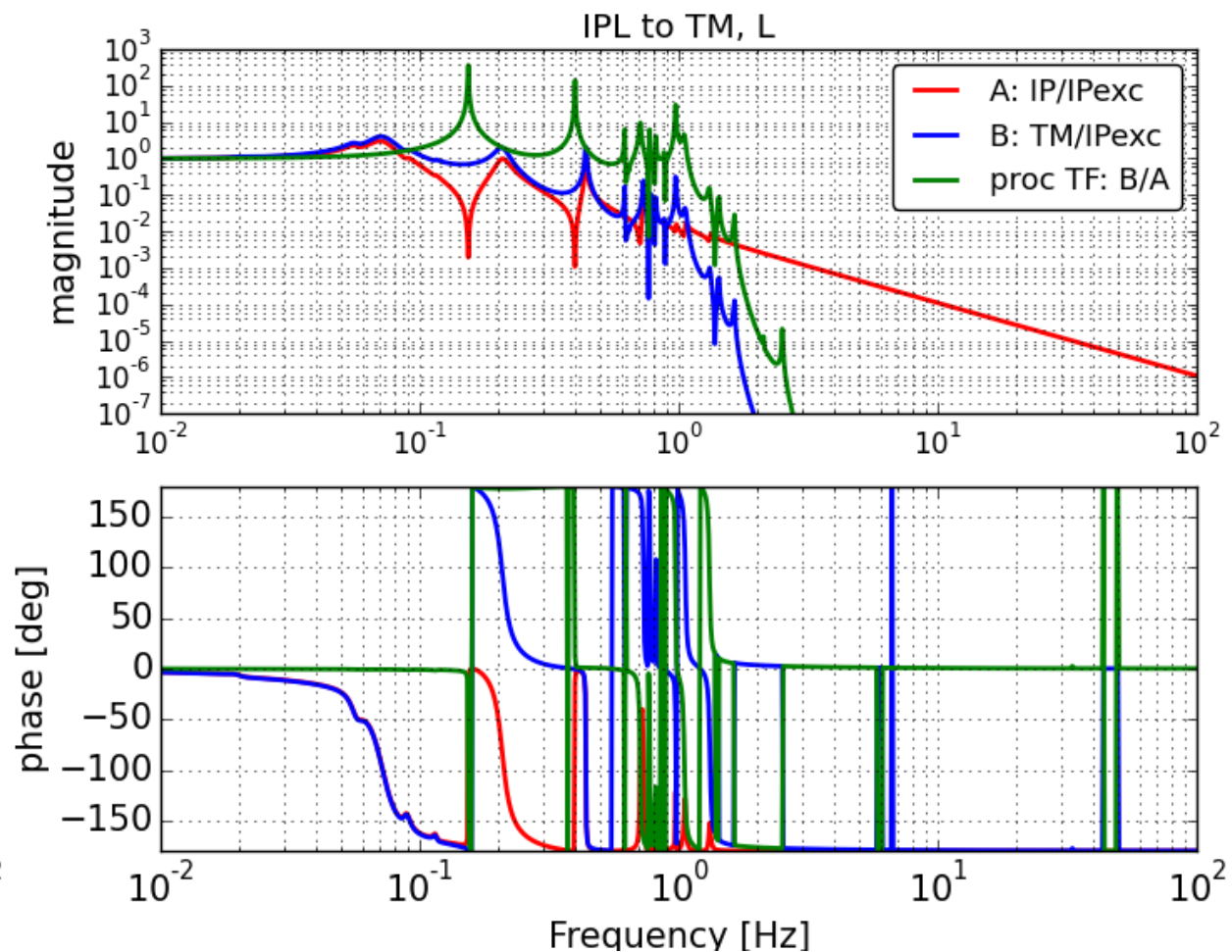
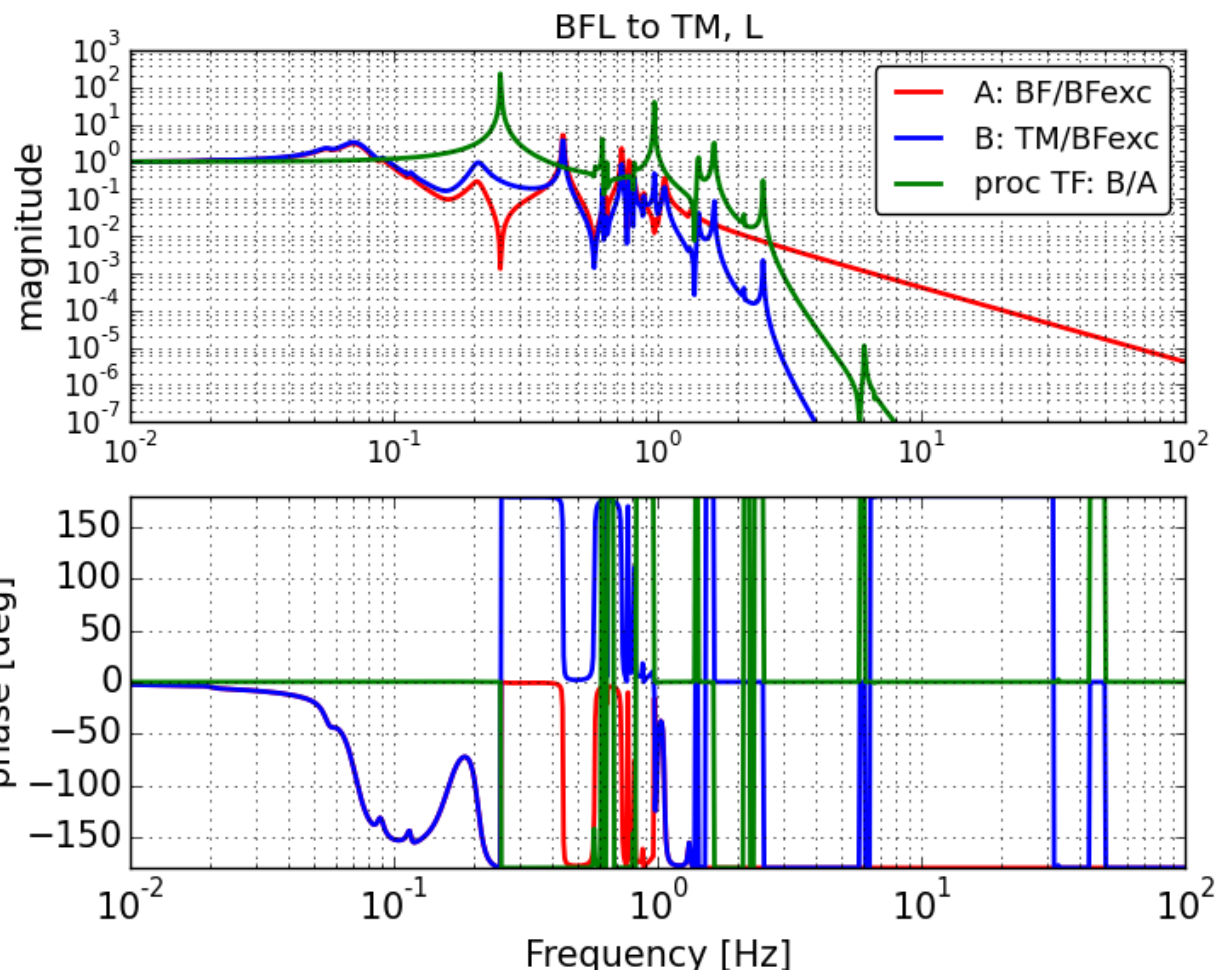
Settings for the measurement w/o controls:

With Tower-damped state,

- ordinal L-loop (blue) was opened at IP satge.
- instead,
- green curve loop was used for the ALS_DARM measurement,
- red curve loop was used for the FPMI_DARM measurement.

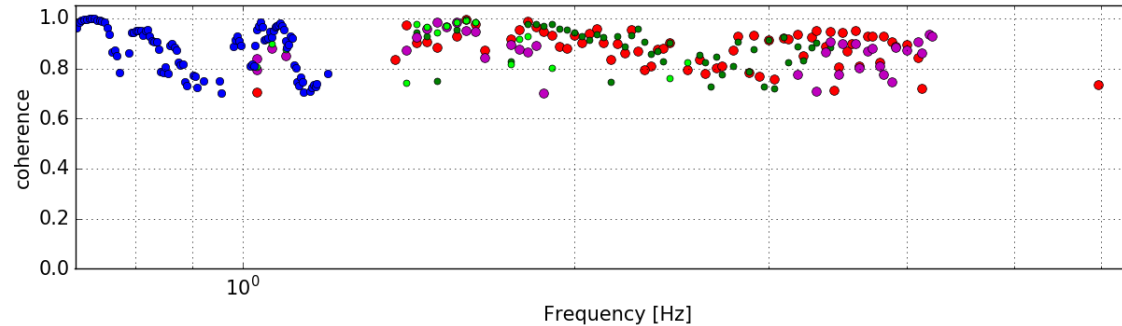
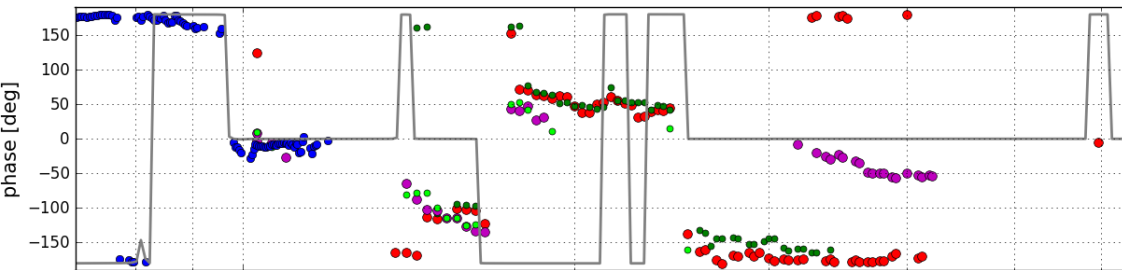
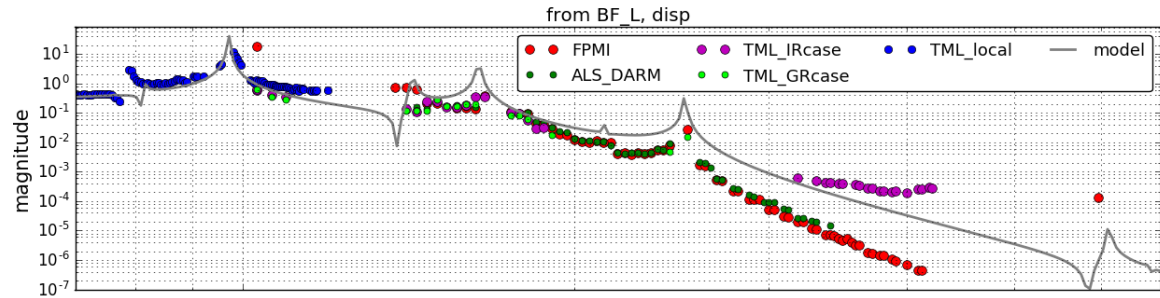


Simulation models:

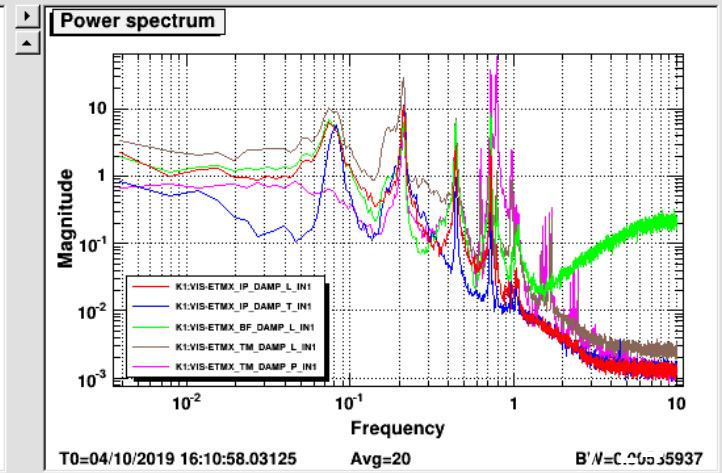
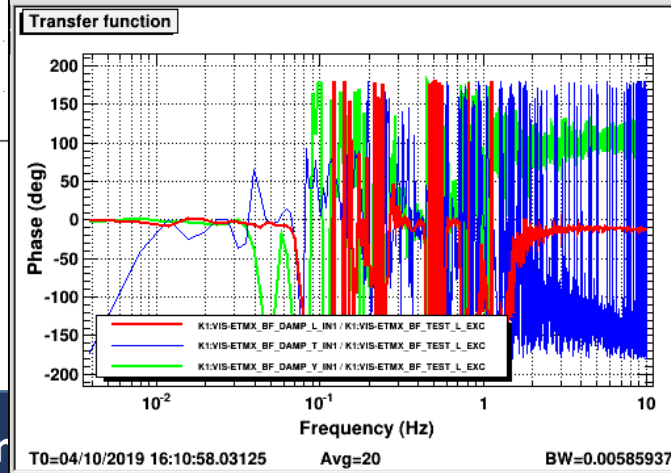
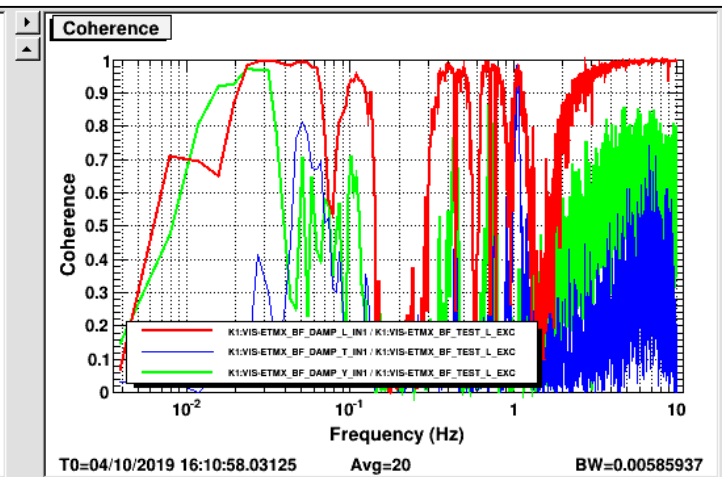
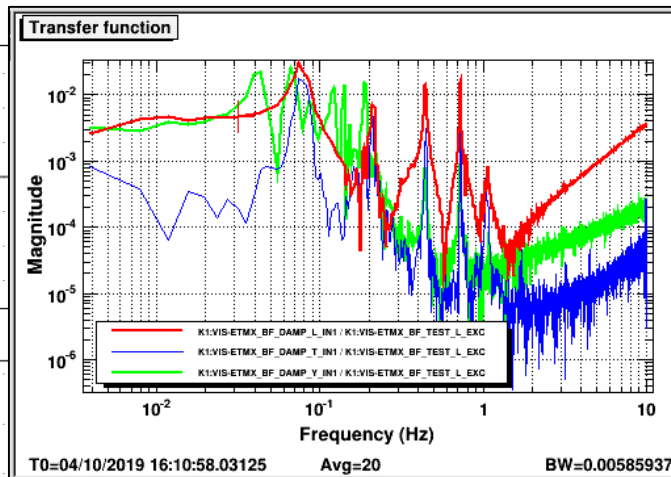


Results: displacement transfer functions, from BFL

It seems that the discrepancy comes from the issue of LVDT, the direct sensor-actuator coupling.

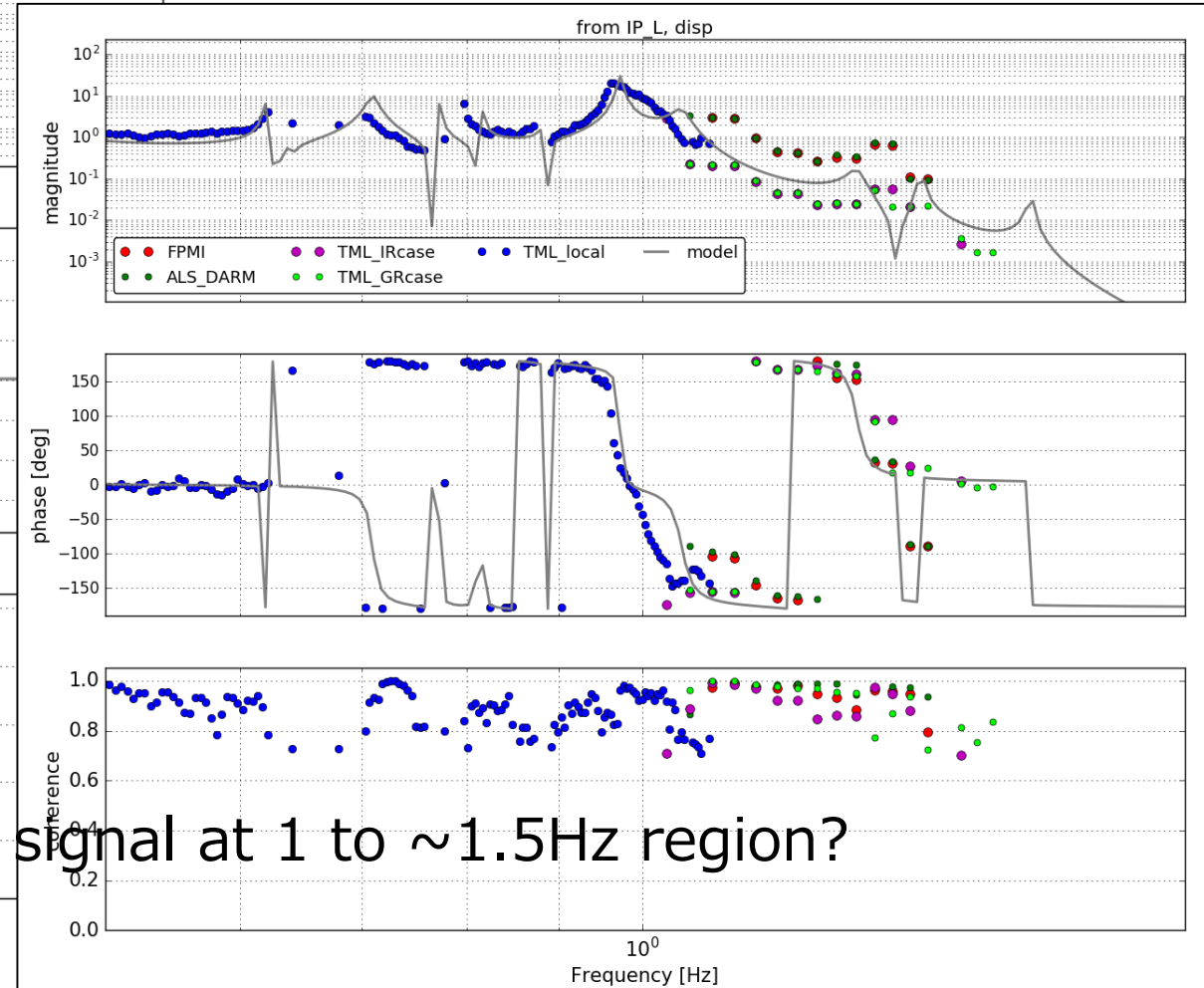
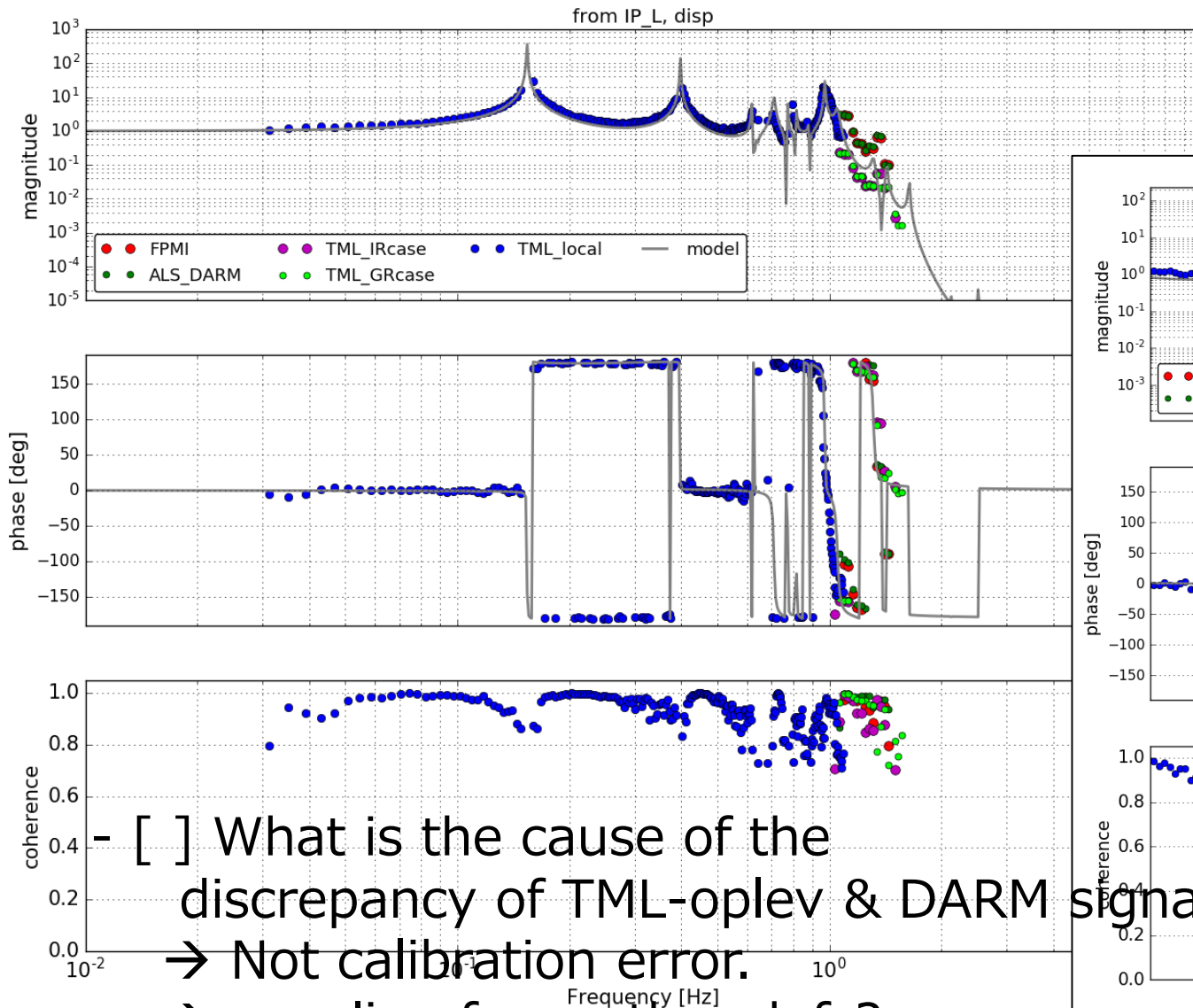


→ Not reliable.



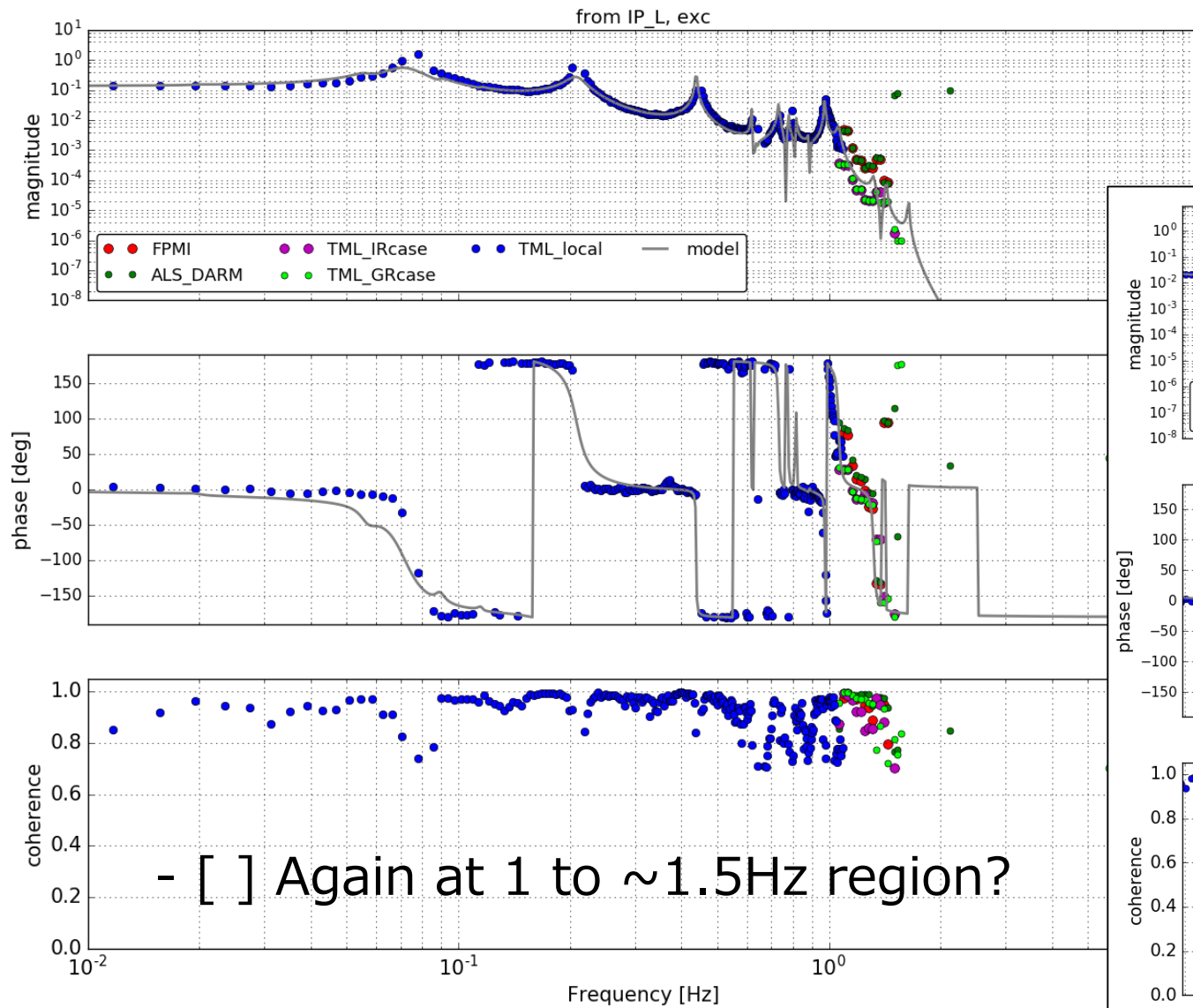
Results: displacement transfer functions, from IPL

It seems that model have to be tuned at 1 to ~1.5Hz region.

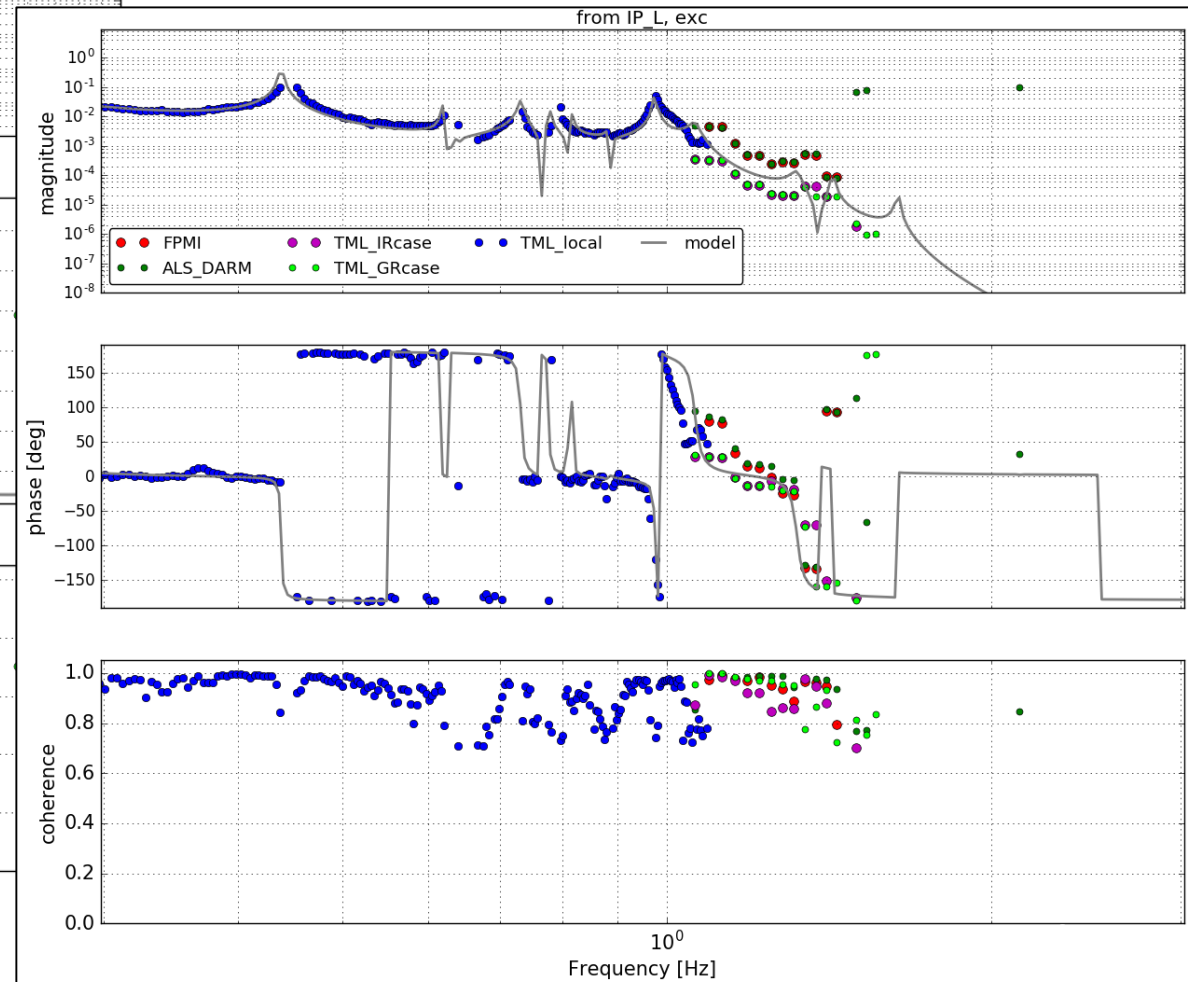


- [] What is the cause of the discrepancy of TML-oplev & DARM signal at 1 to ~1.5Hz region?
 → Not calibration error.
 → coupling from other dofs?

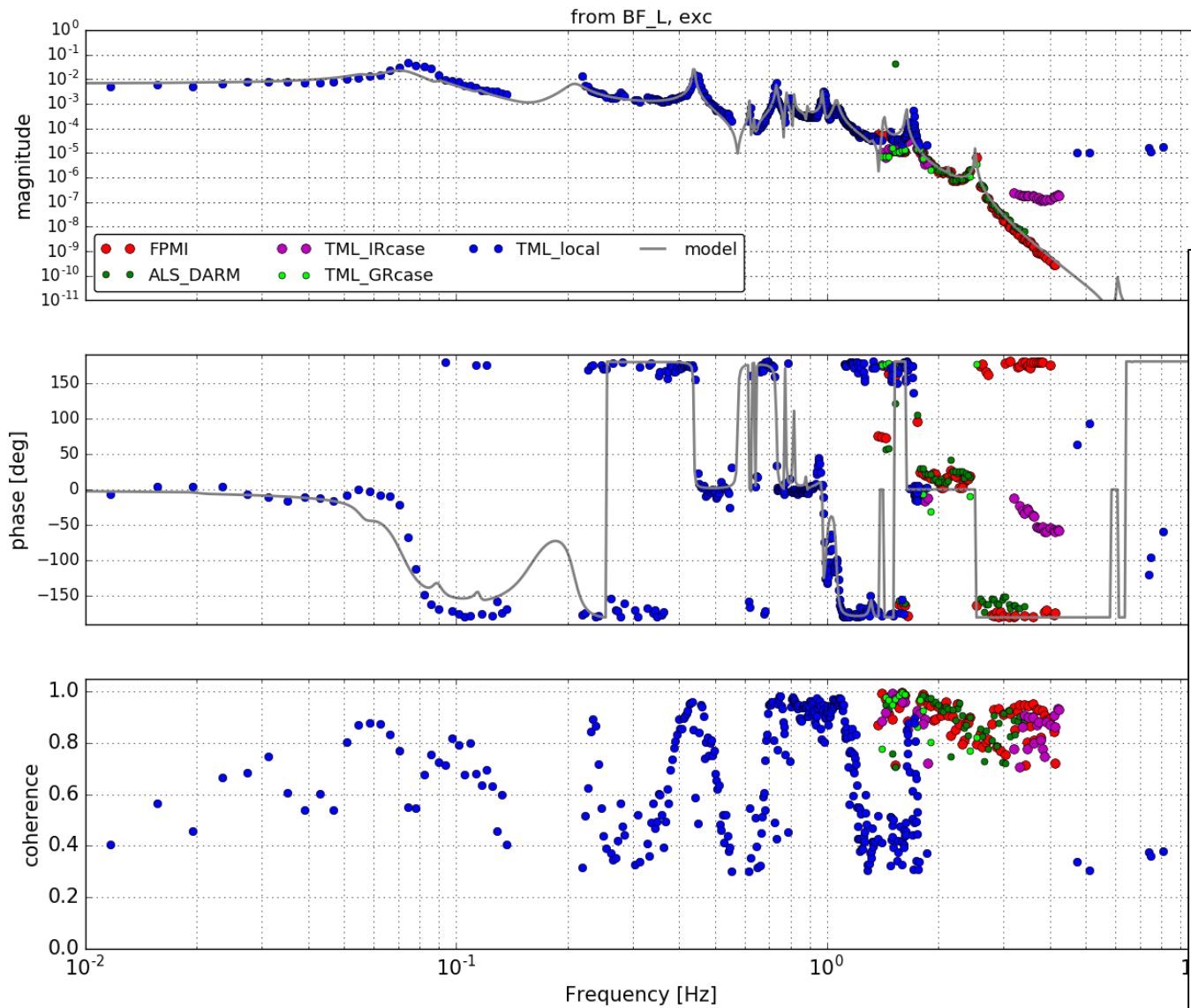
Results: force transfer functions, from IPL



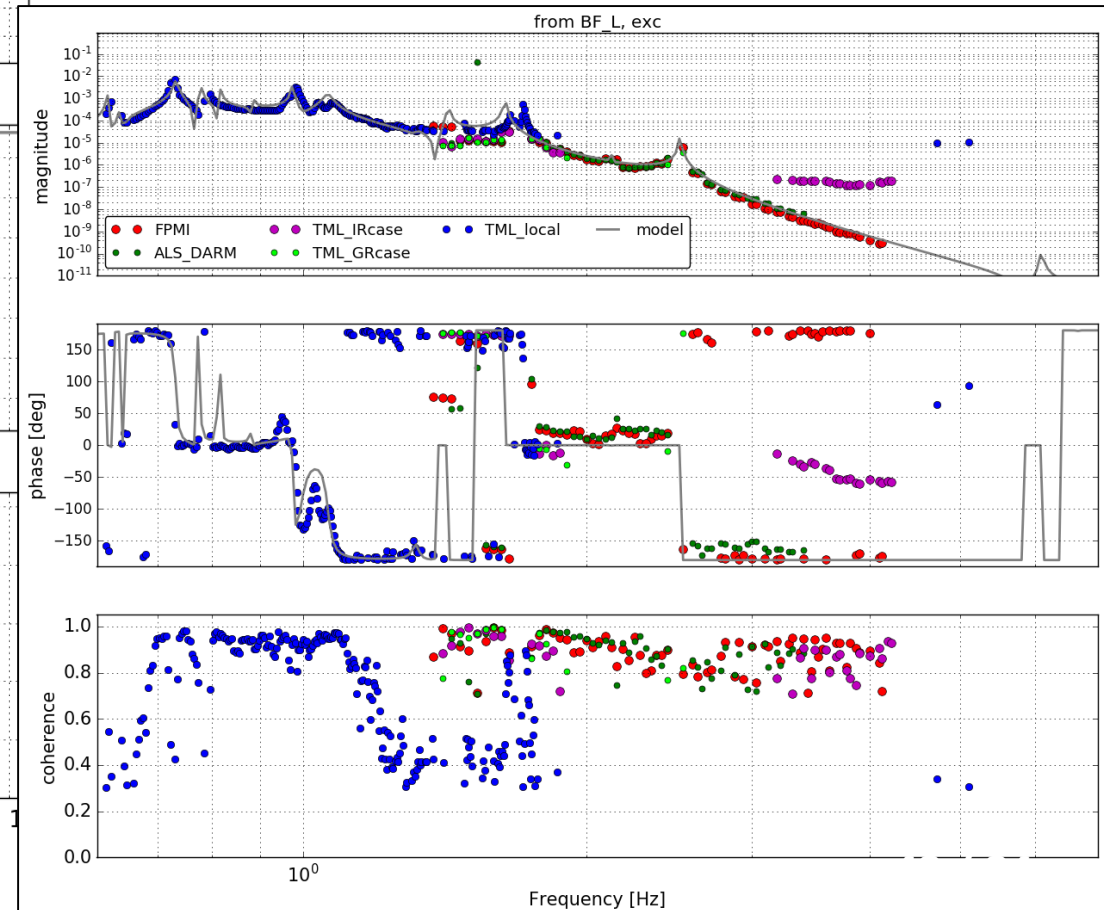
- [] Again at 1 to ~1.5Hz region?



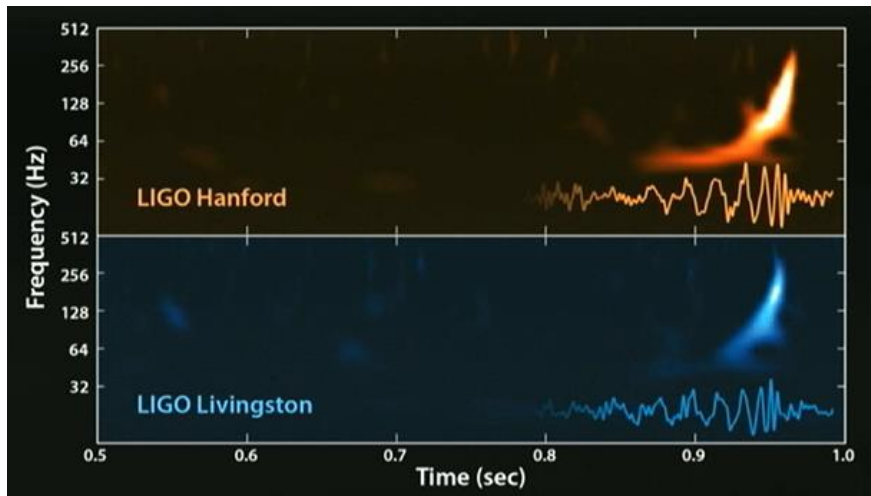
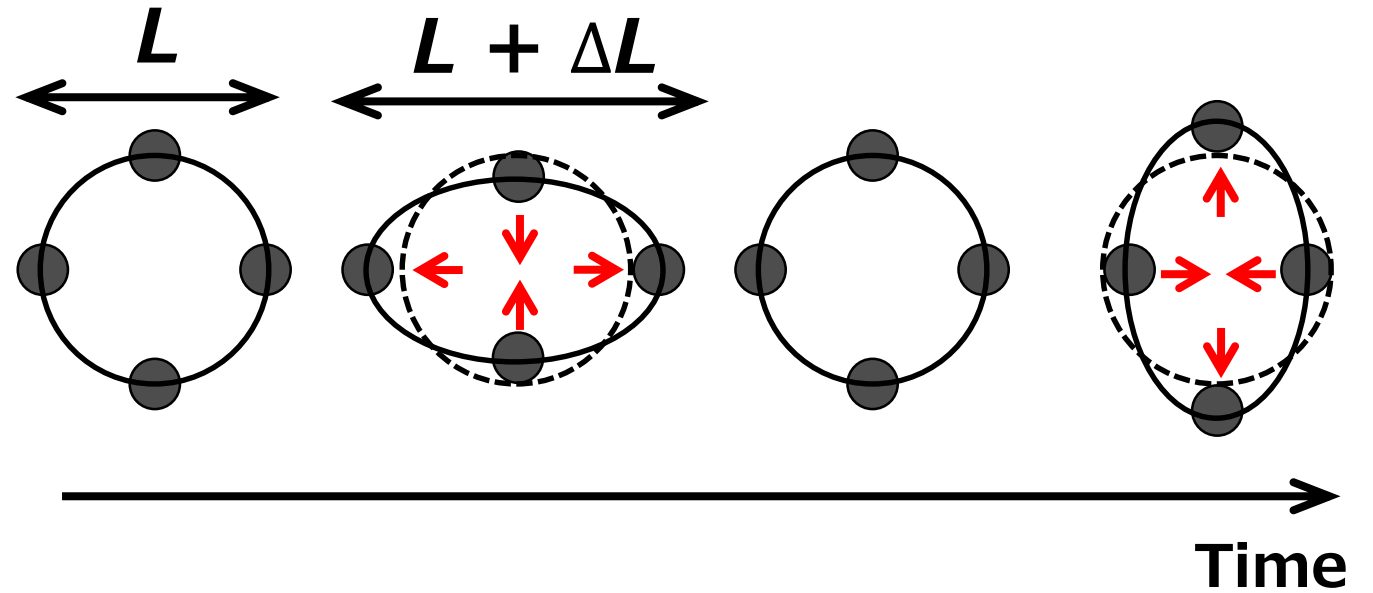
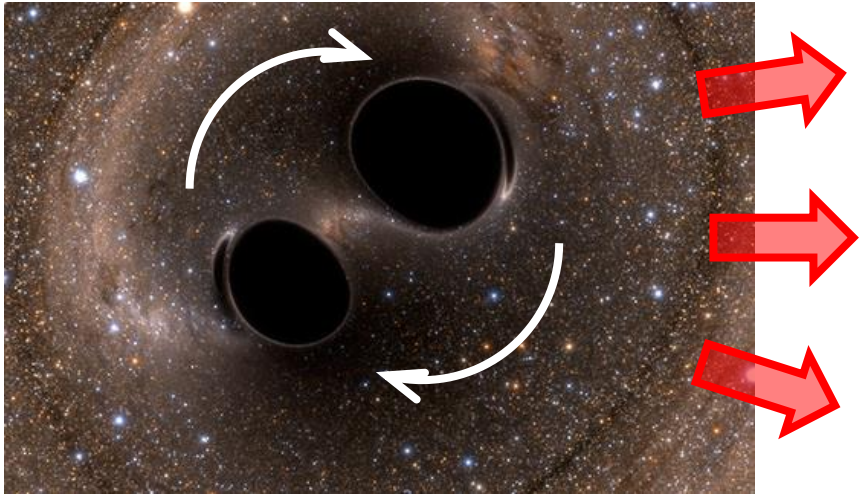
Results: force transfer functions, from BFL



The transfer function up to 4Hz was agree with the model.



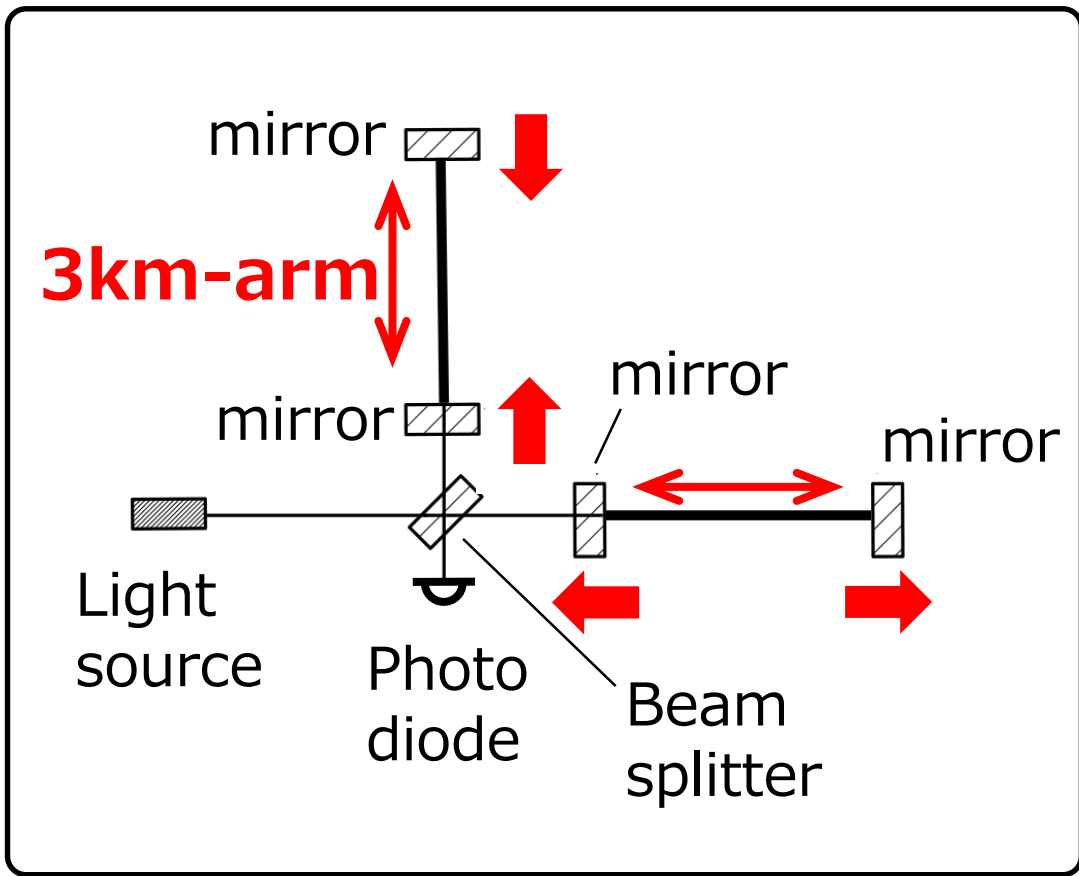
重力波?



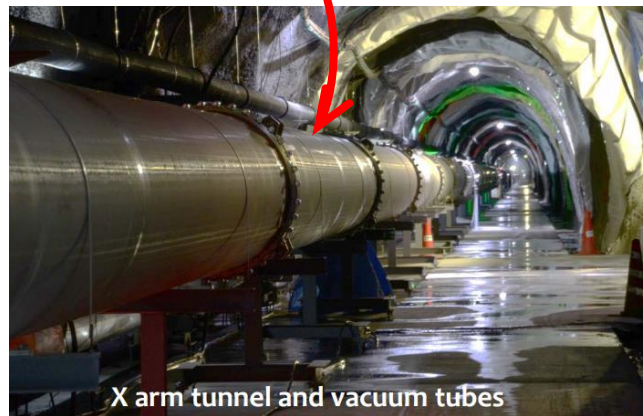
- 2015.9.14 初検出!
- BBH, BNS 検出!

→ 新しい天文学!

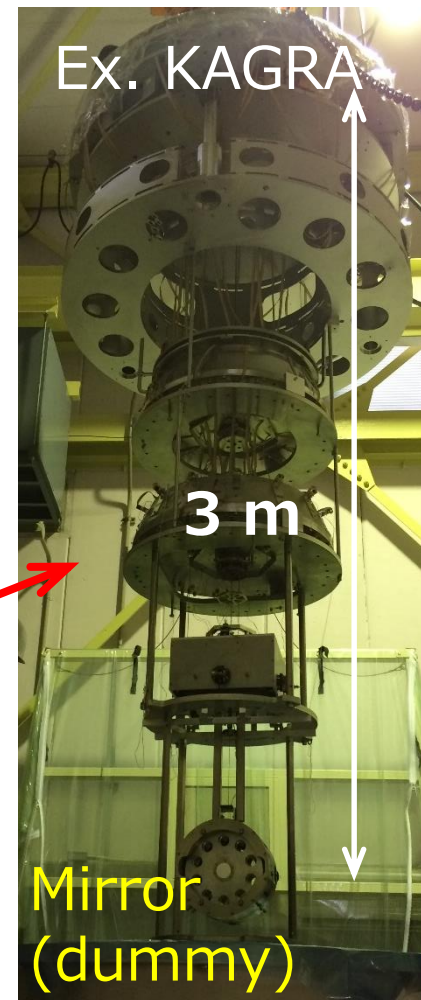
重力波検出器と、サスペンション



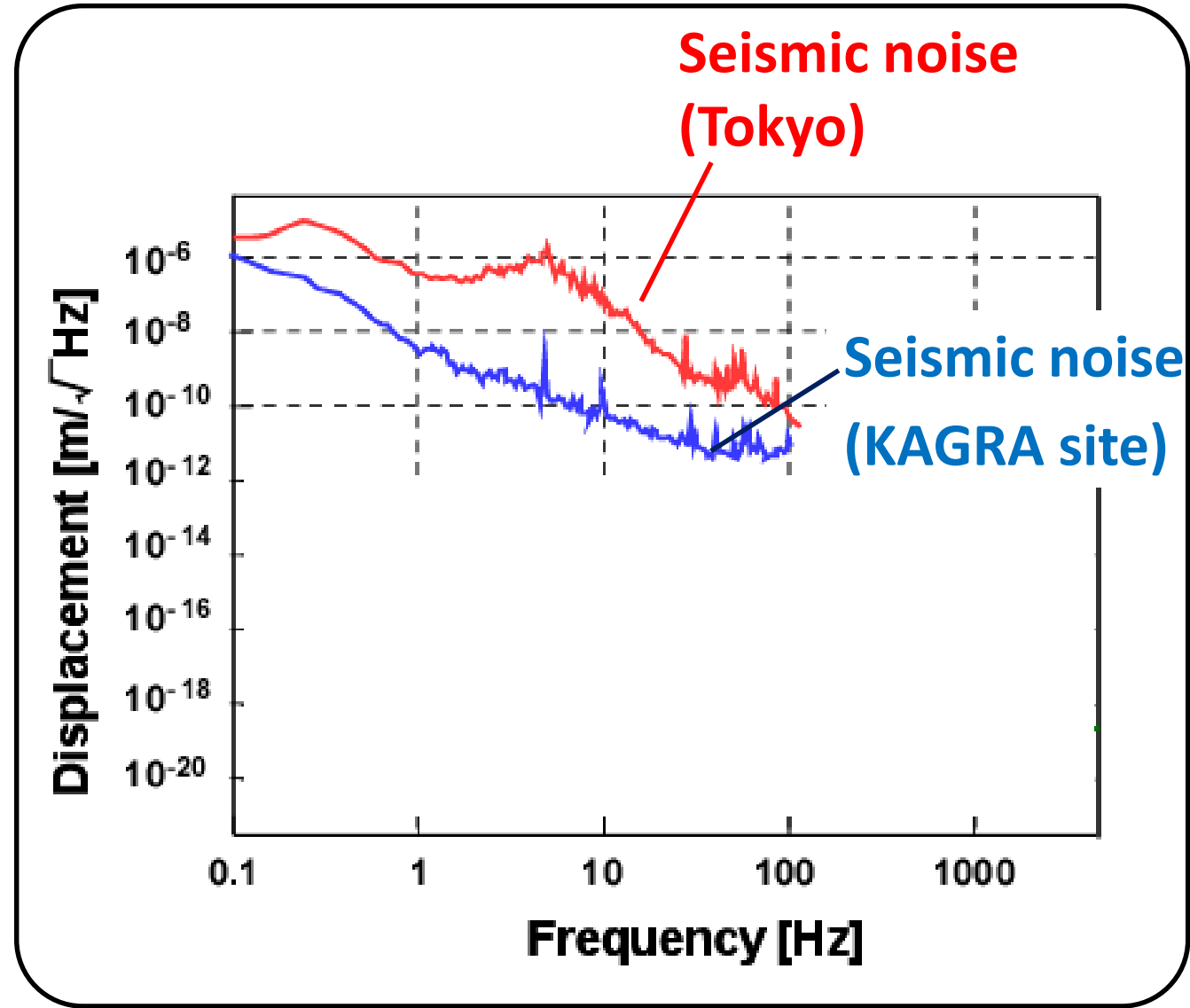
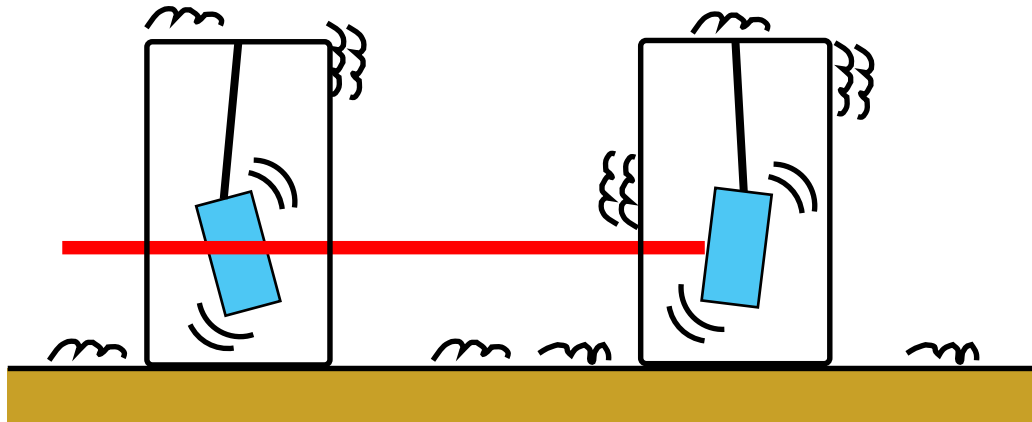
- 1) Michelson-based interferometer
- 2) Fabry-Perot cavities
- 3) 3km-arm



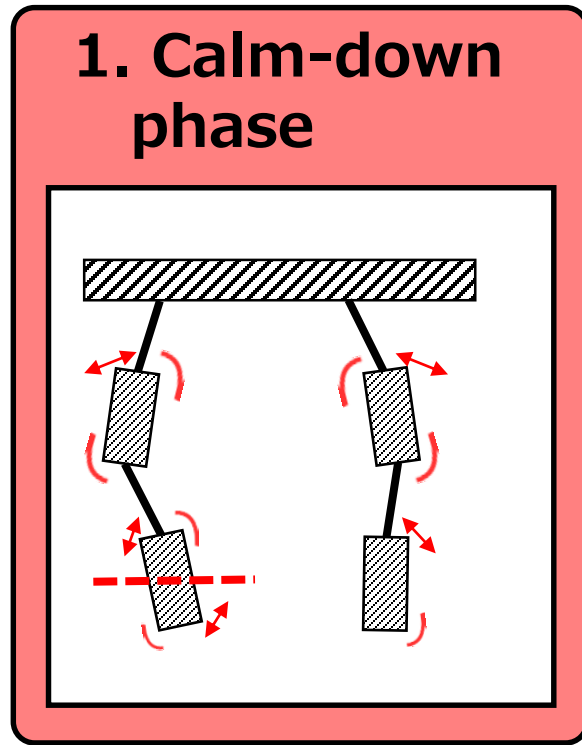
- 4) Suspended core optics



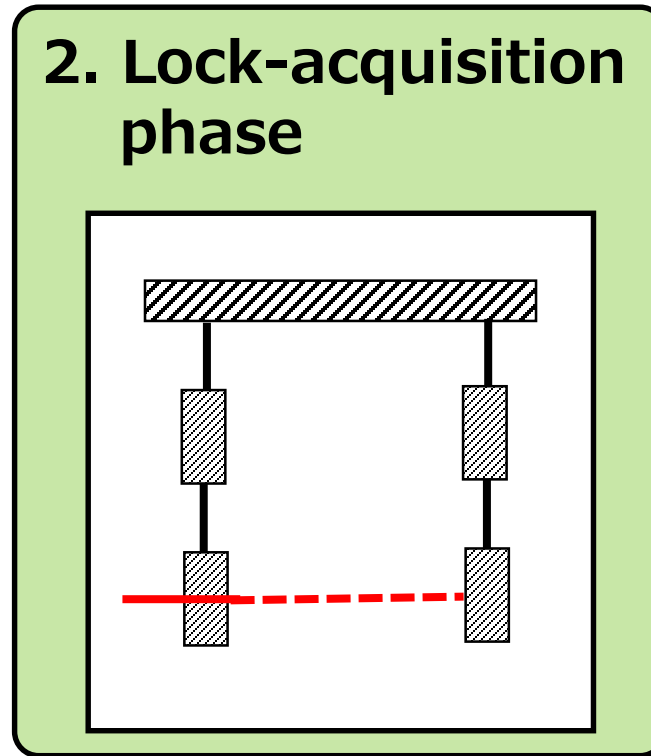
地面振動雑音



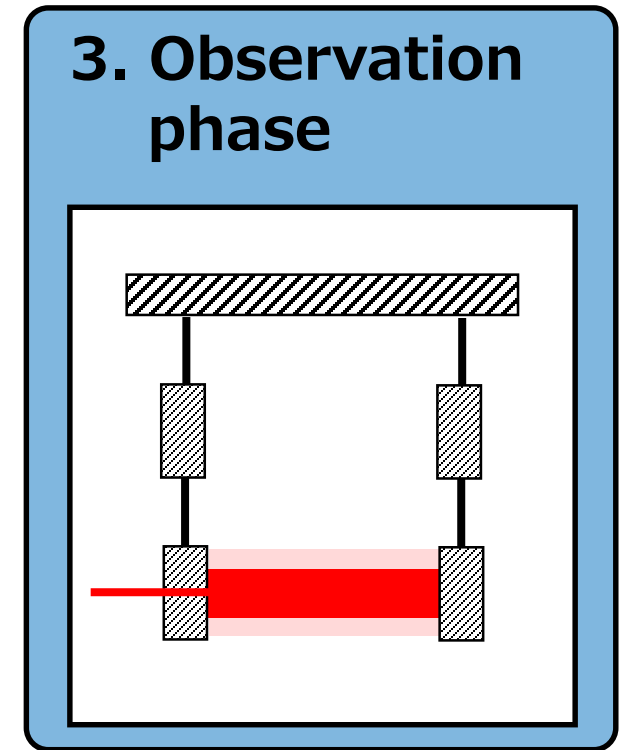
Designing active control system / Control phase



Suppress
large disturbance

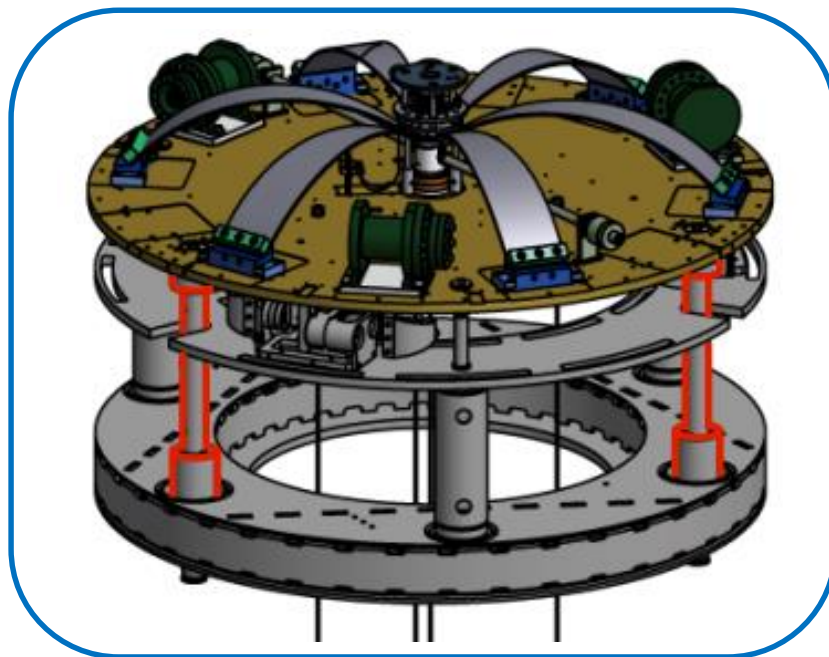
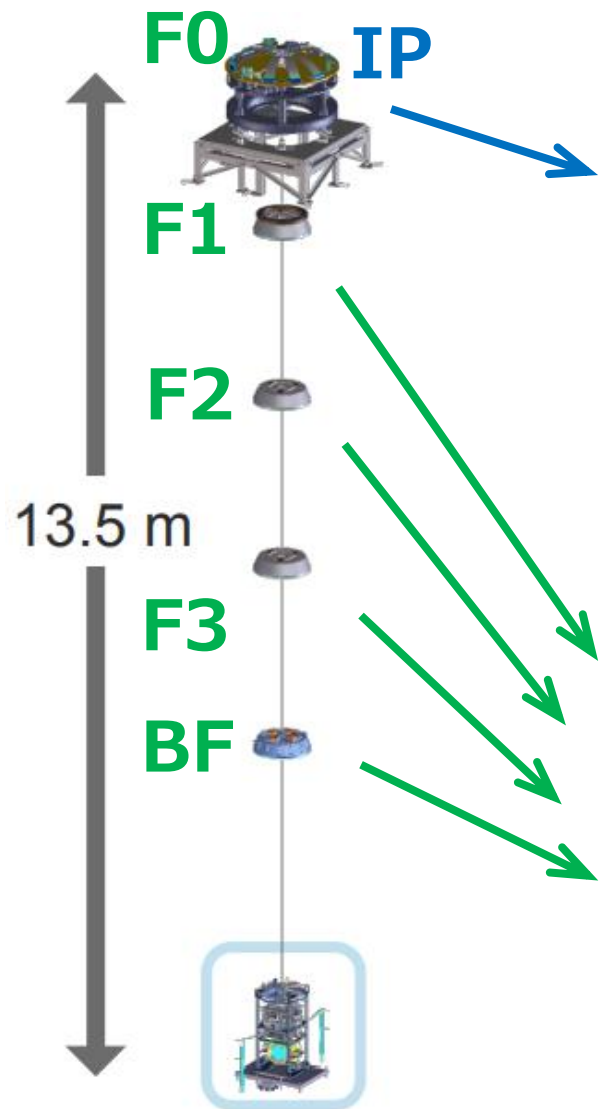


Reduce RMS velocity
RMS angle
(**R**oot-**M**ean-**S**quare)

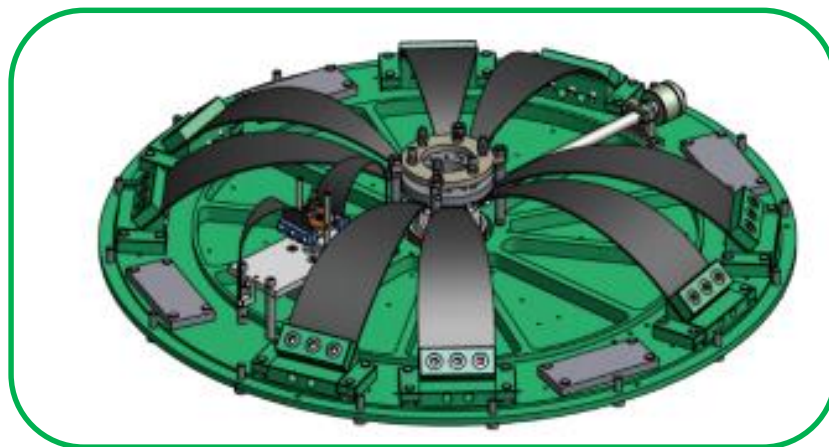


Keep position
with low noise
control

メイン鏡用の防振装置

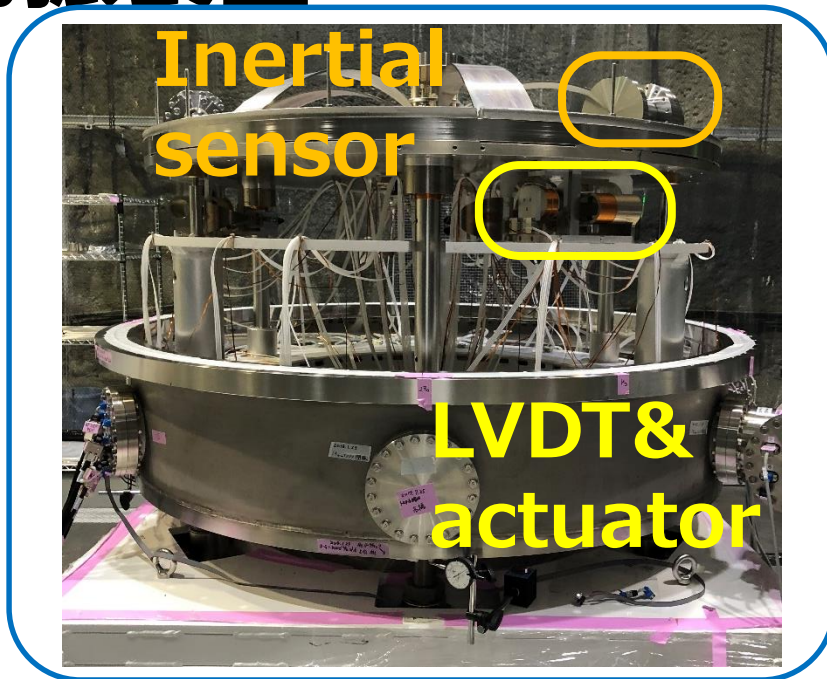
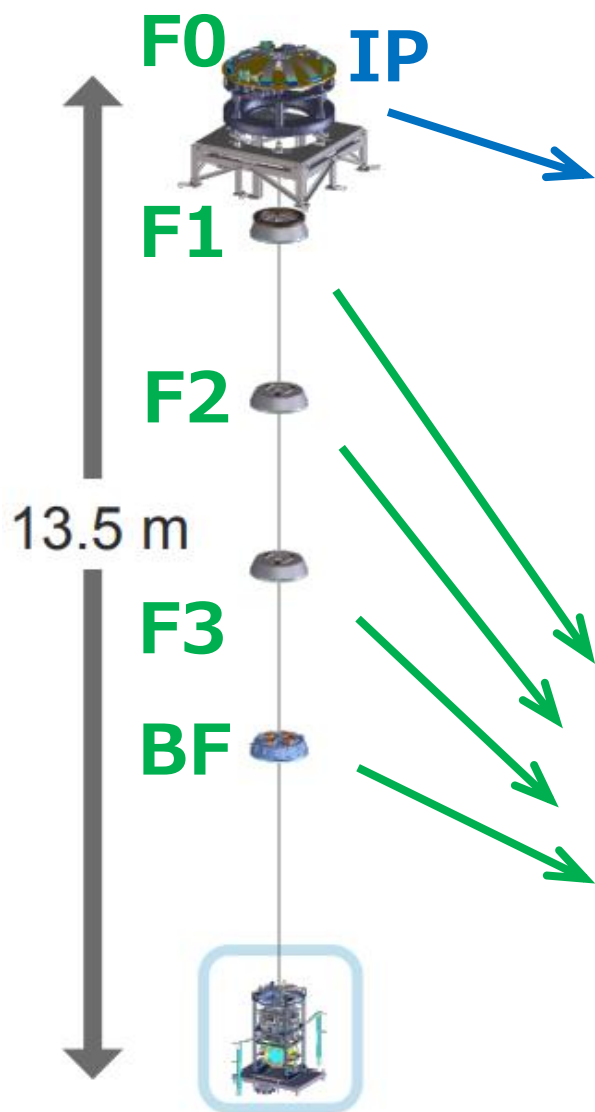


INVERTED PENDULUM
(~ 70 mHz)

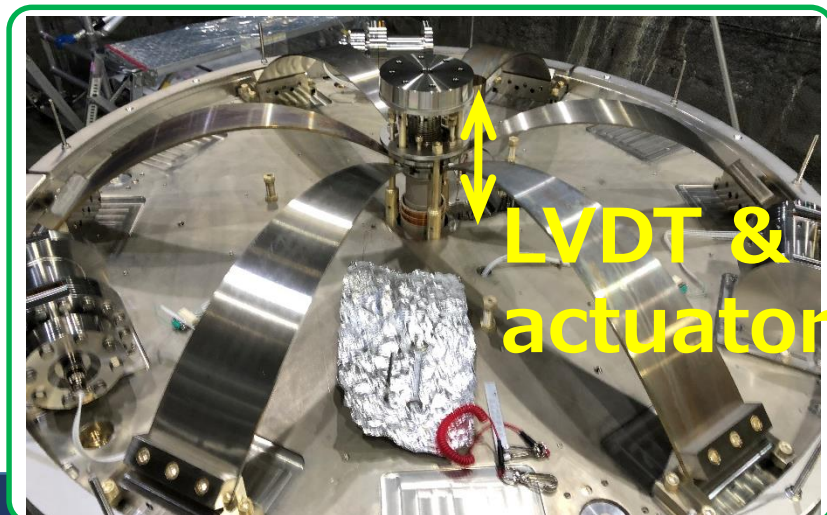


GEOMETRIC-ANTI SPRING
(~ 0.4 Hz)

メイン鏡用の防振装置



INVERTED PENDULUM
with 3 horizontal
-- LVDT & actuator units
-- inertial sensors



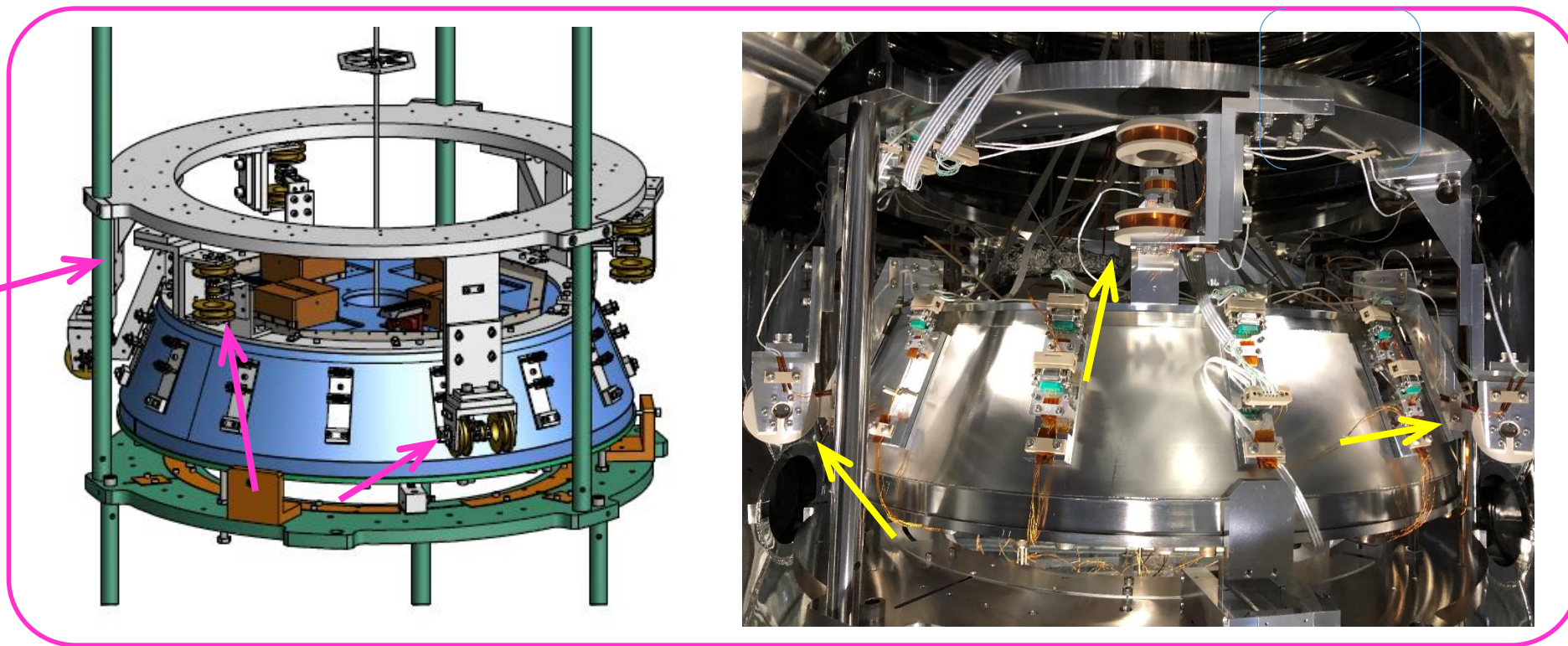
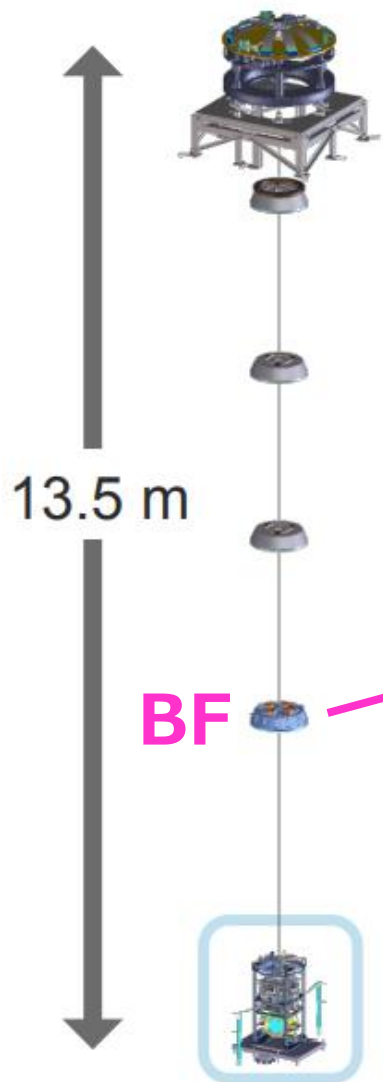
GEOMETRIC-ANTI SPRING
with 1 vertical
LVDT & actuator unit

メイン鏡用の防振装置

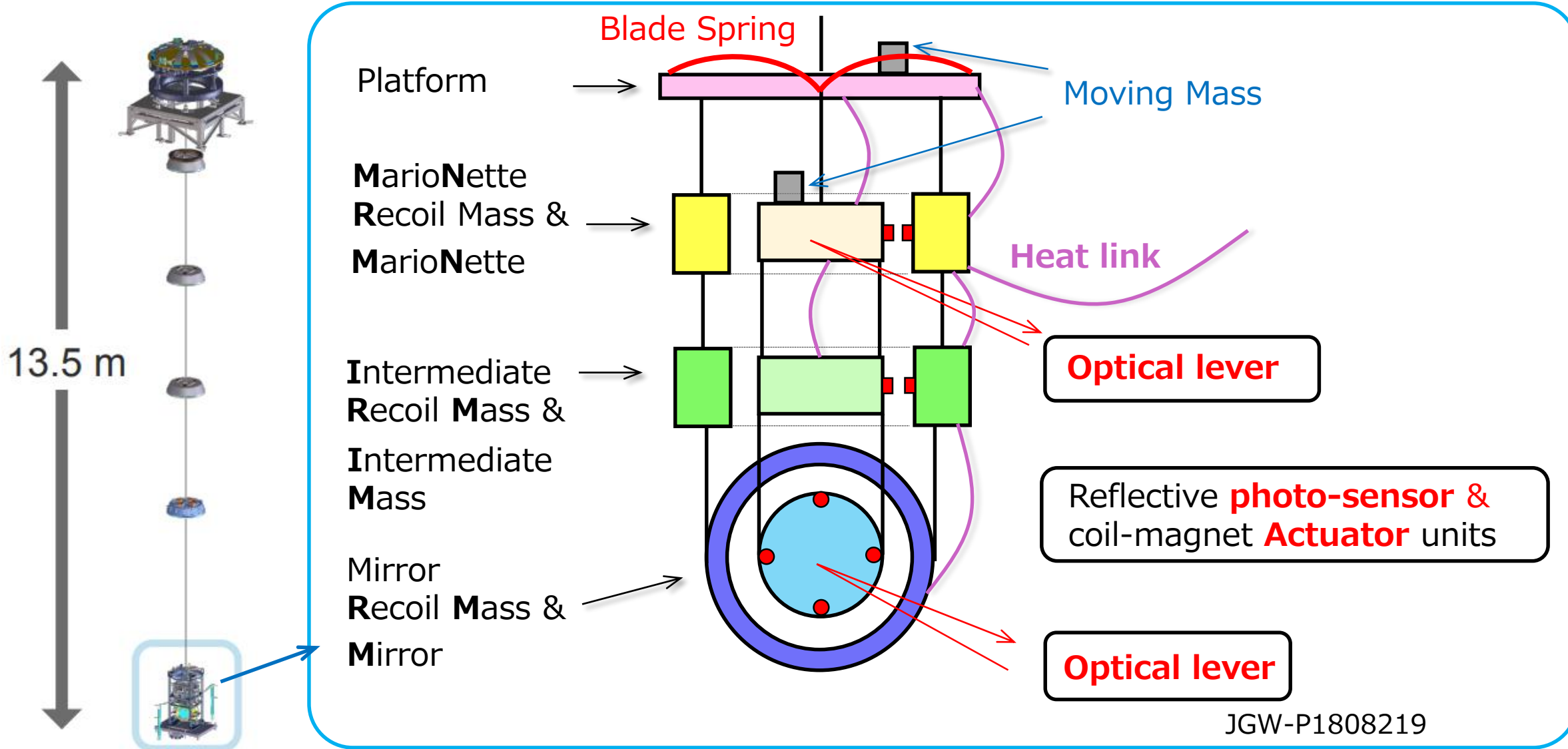
(イタリアのグループの協力のもと開発)

BOTTOM-FILTER DAMPER

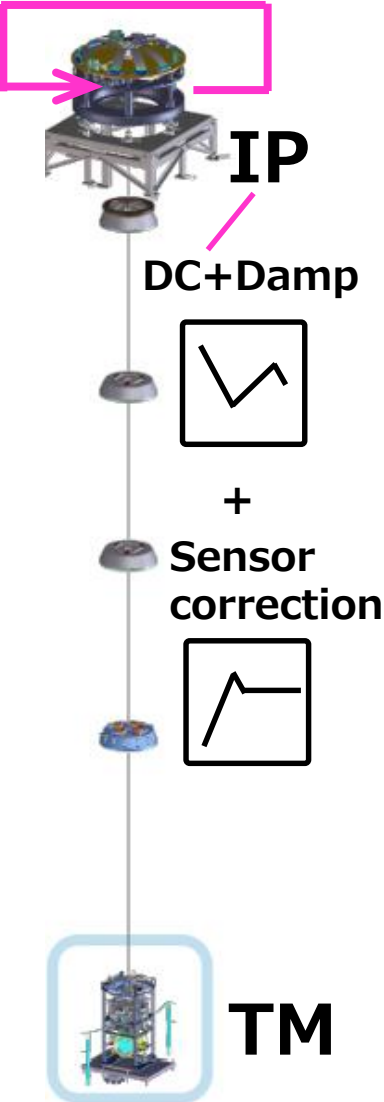
with 3 horizontal & 3 vertical LVDT & actuator units



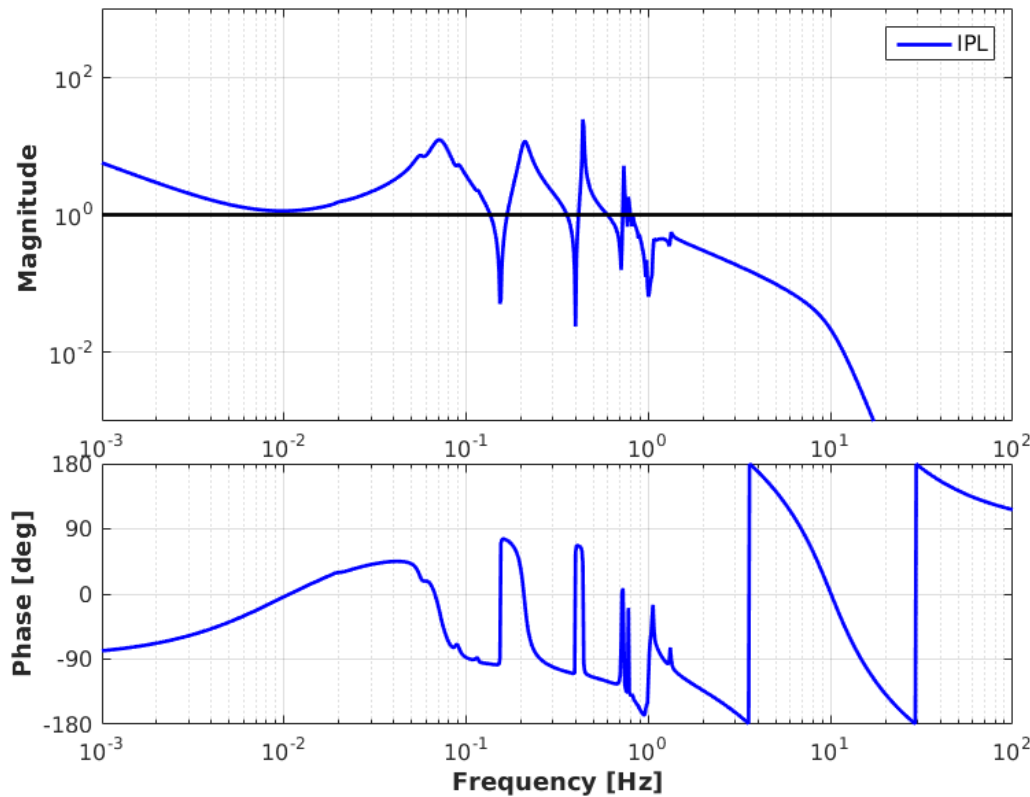
メイン鏡用の防振装置



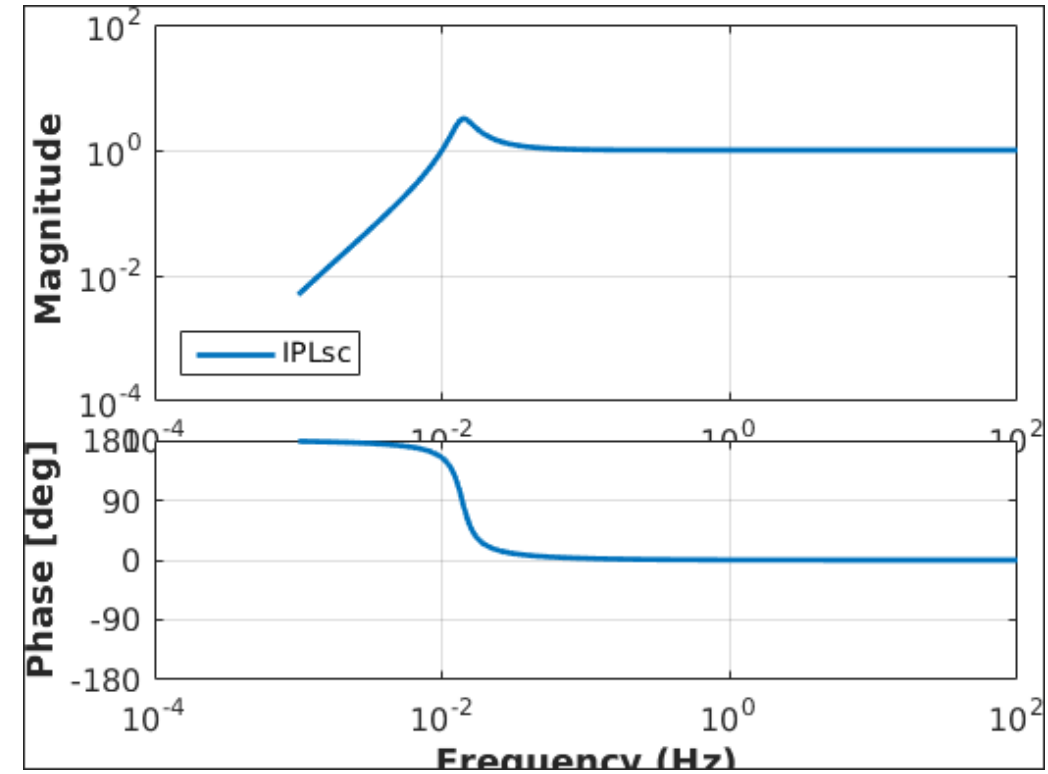
制御ループ^o at IP-stage:



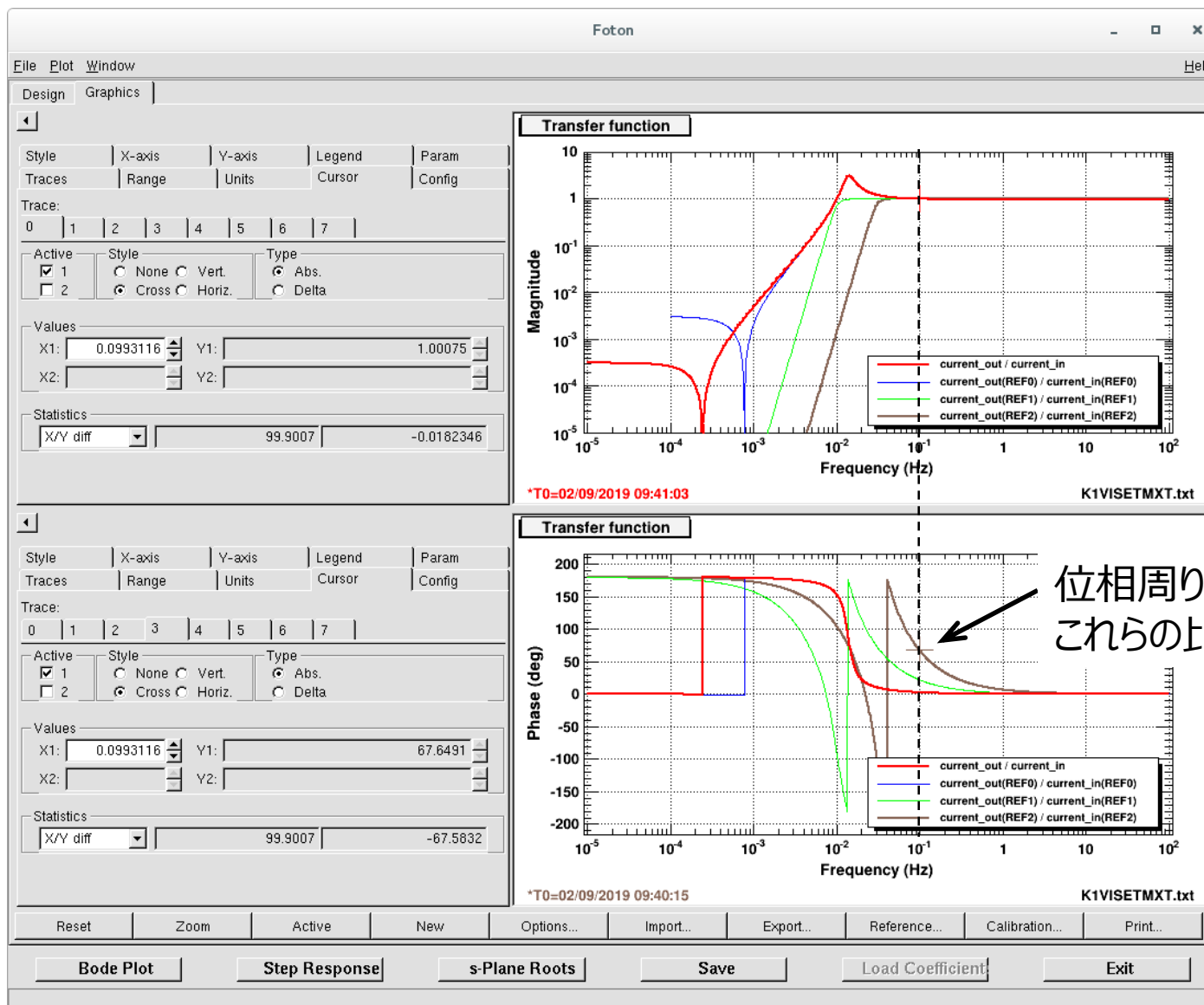
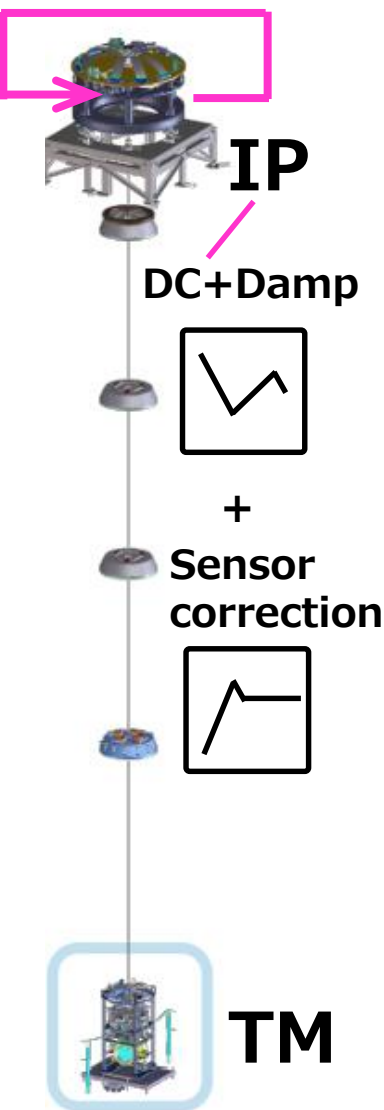
1. OLTF for IP-LVDT



2. Sensor correction filter



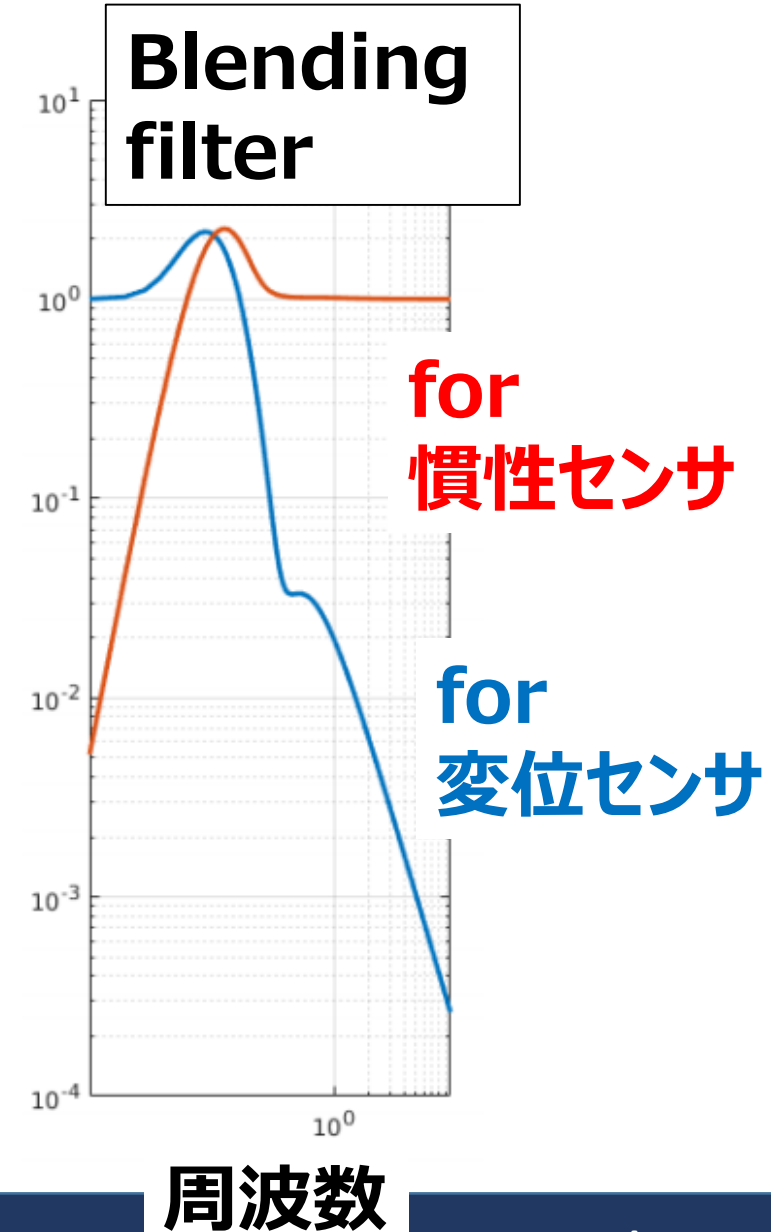
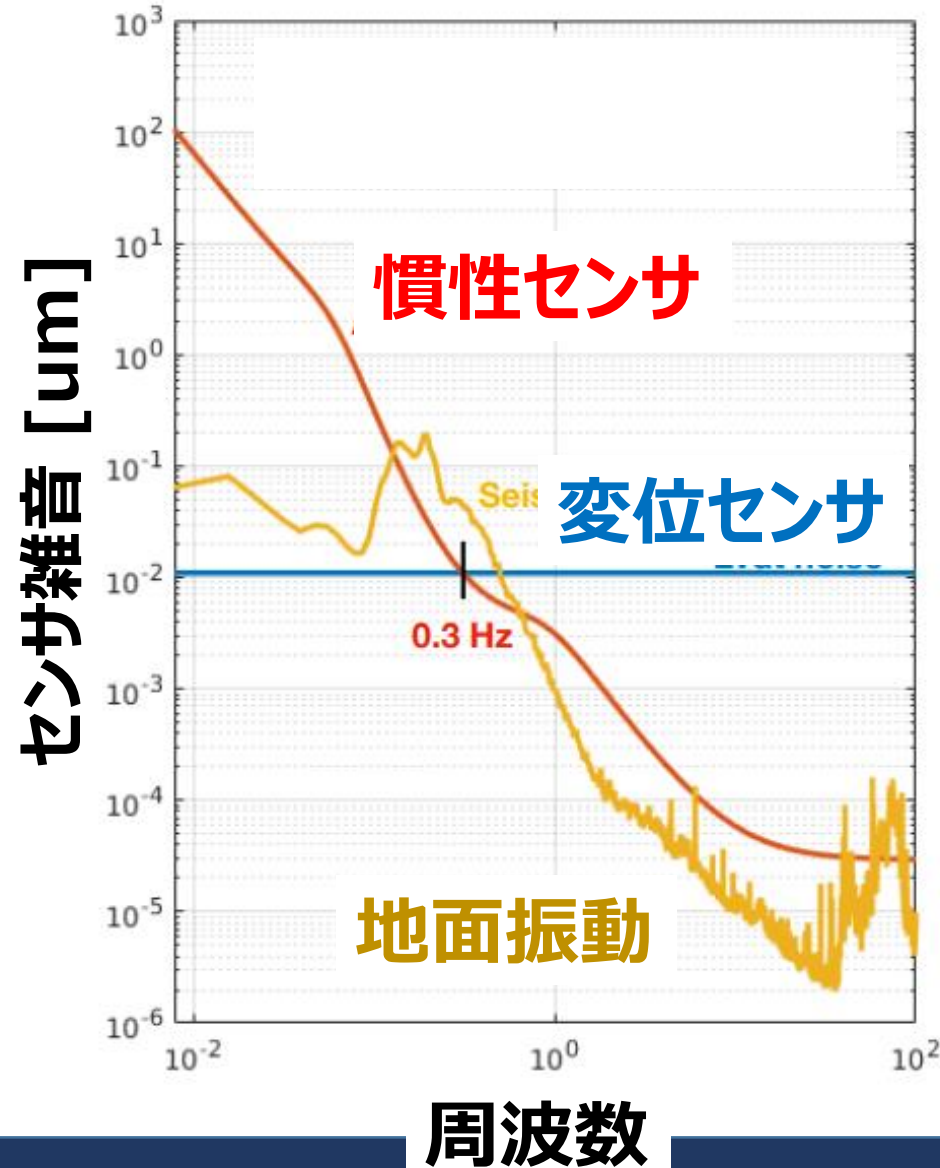
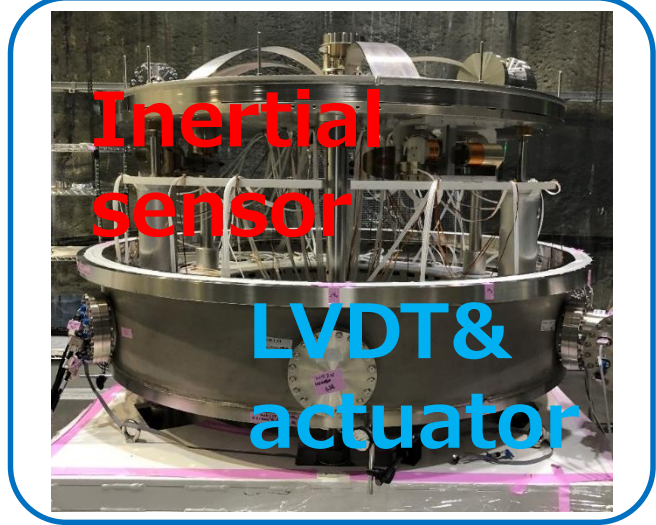
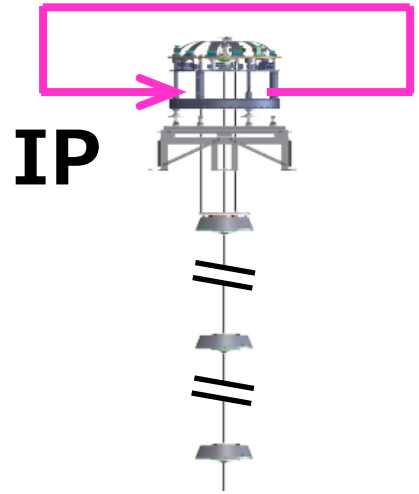
Sensor correction filter の検討



位相周りがなるべく少ないものを採用した。
これらの比較plotを作成しておく。

Sensor blending & inertial damping

(From JGW-G1909932)



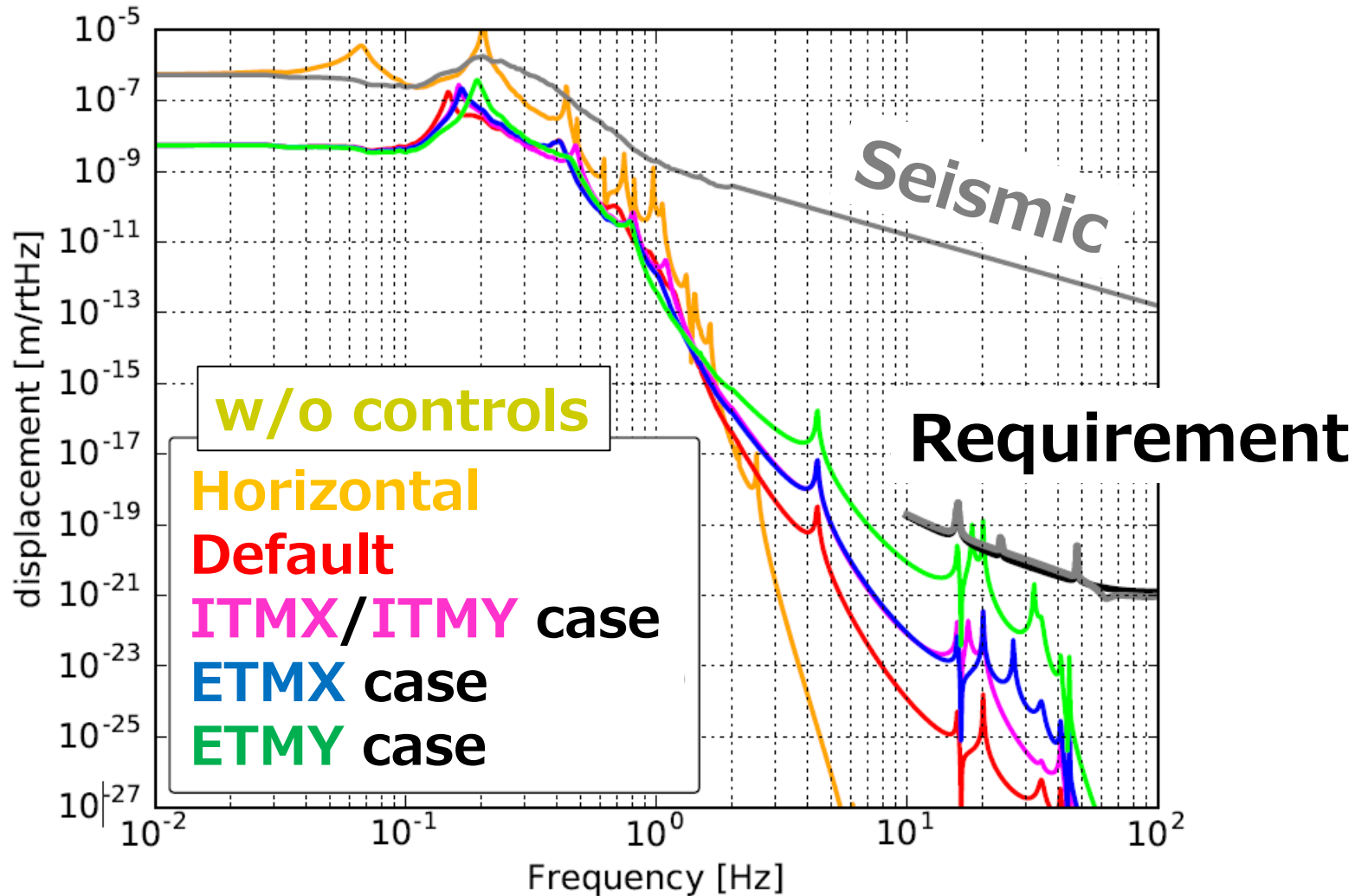
Requirements, Type-A suspensions

Calm-down phase		
Item	Requirement	For/Determined by
1/e modal decay time	< 1 min	Quick recovery
RMS displacement (L)	< 50 μm	Smooth transition to next phase
RMS displacement (T, V)	< 0.1 mm	Miscentering
RMS angle (P, Y)	< 50 μm	Smooth transition to next phase
Lock acquisition phase		
Item	Requirement	For/Determined by
RMS velocity (L)	< 240 $\mu\text{m/s}$	Auxiliary laser locking
RMS displacement (T, V)	< 0.1 mm	Miscentering
RMS angle (P, Y)	< 880 nrad	Optical gain degradation < 5%
Observation phase		
Item	Requirement	For/Determined by
Displacement noise (L) @ 10 Hz	< 8×10^{-20} m/Hz ^{1/2}	Sensitivity
Displacement noise (V) @ 10 Hz	< 8×10^{-18} m/Hz ^{1/2}	Sensitivity (1% coupling to L)
RMS displacement (T, V)	< 0.1 mm	Miscentering
RMS angle (P, Y)	< 200 nrad	Beam spot fluctuation < 1 mm
DC drift (P, Y)	< 400 nrad/h	Sustainable lock for 1 day left

(P, Y) are set as 50 μm and 50 μrad , respectively [28]. The RMS displacement for the other translational DoFs (T, V) are required for another reason which is mentioned shortly later.

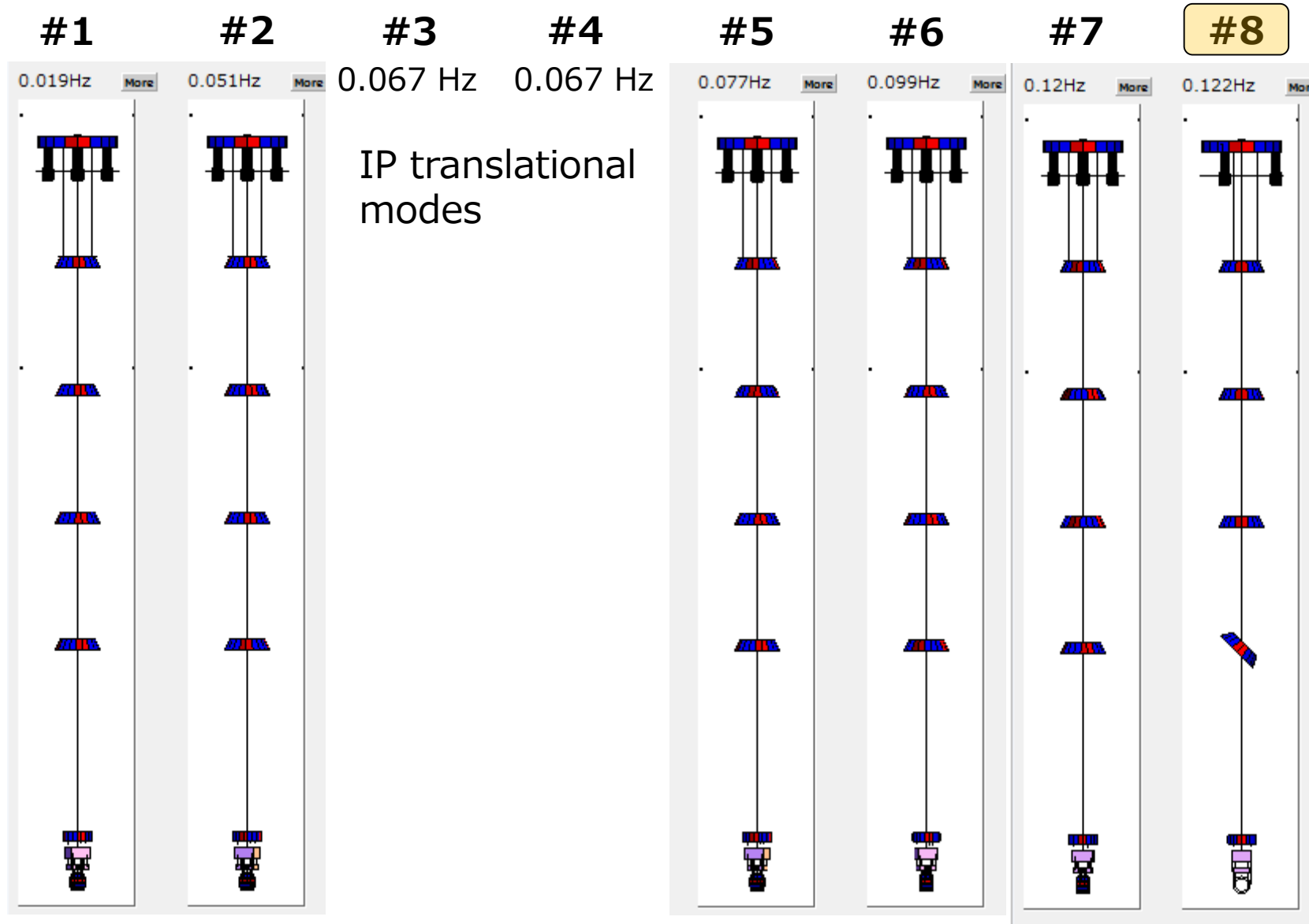
Mechanical installation has done! **HOWEVER ..**

According to a simulation, assuming **1%** coupling,



“acceptable for the O3-run”
(should be)

Note:
-- Modeled w/o Heat-links
-- params are not tuned.

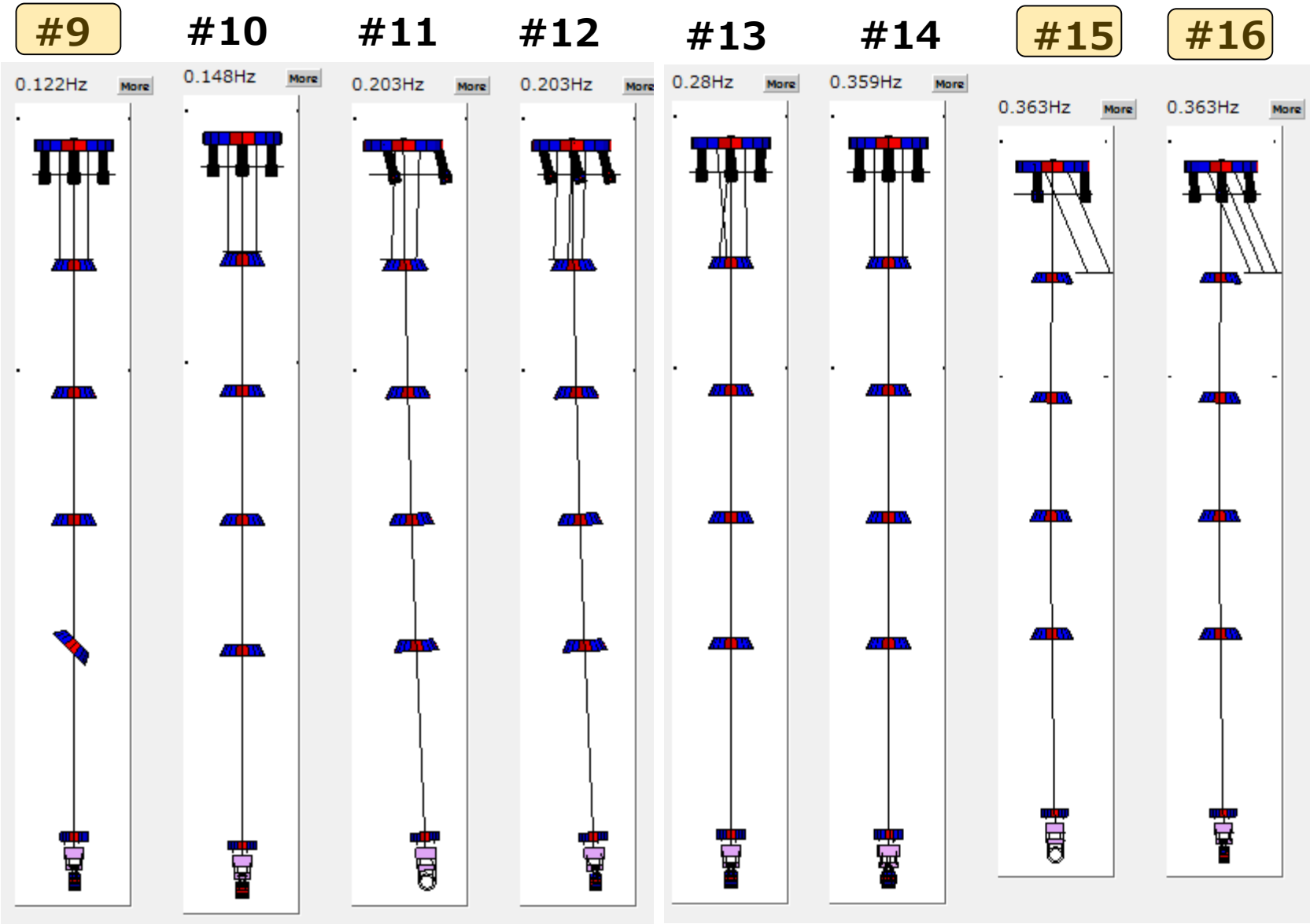


Type-A SAS,

'TypeA180429_20K'

Eigen mode: 75 modes

Less interest now



#9 Less interest now

#17

#18

#19

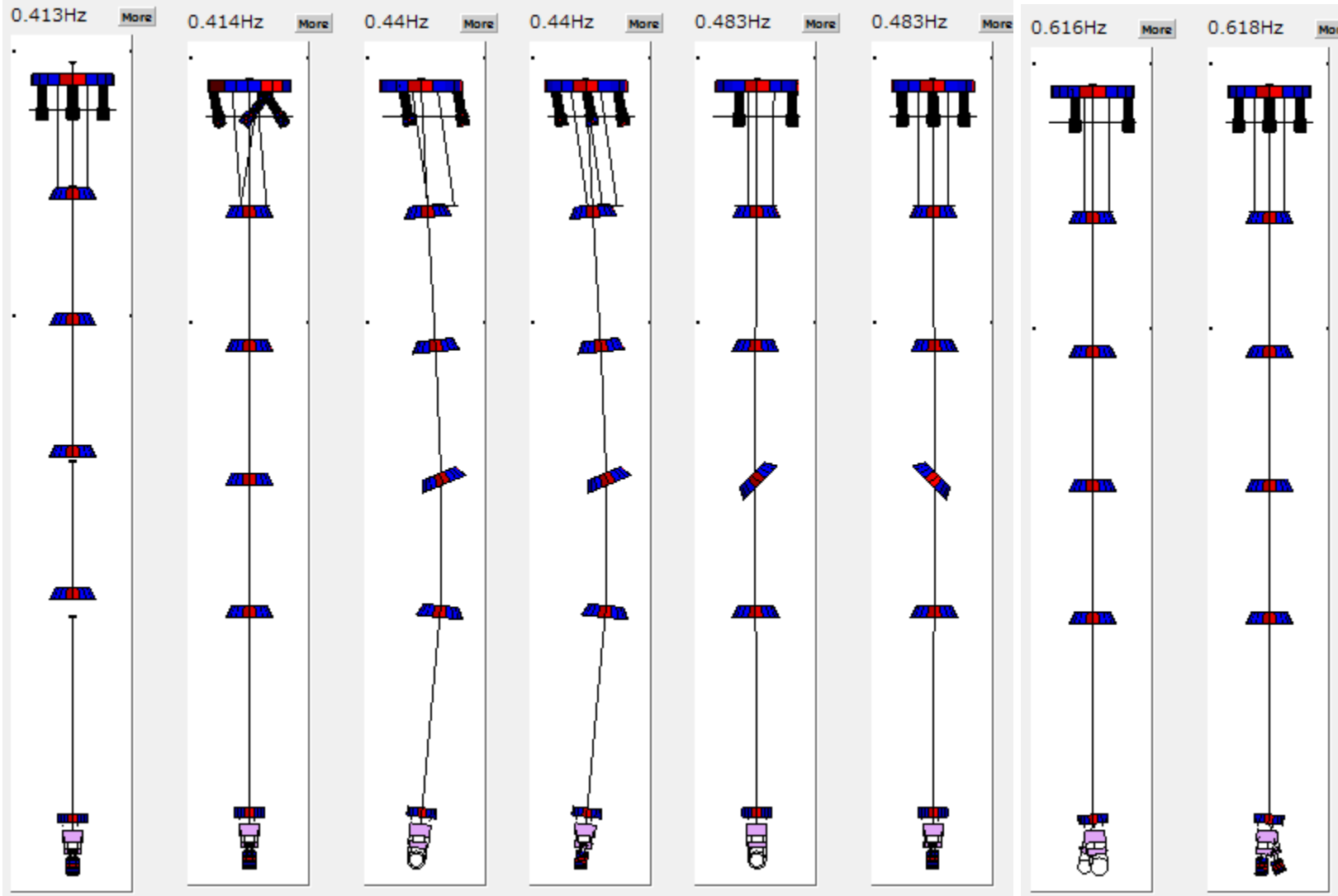
#20

#21

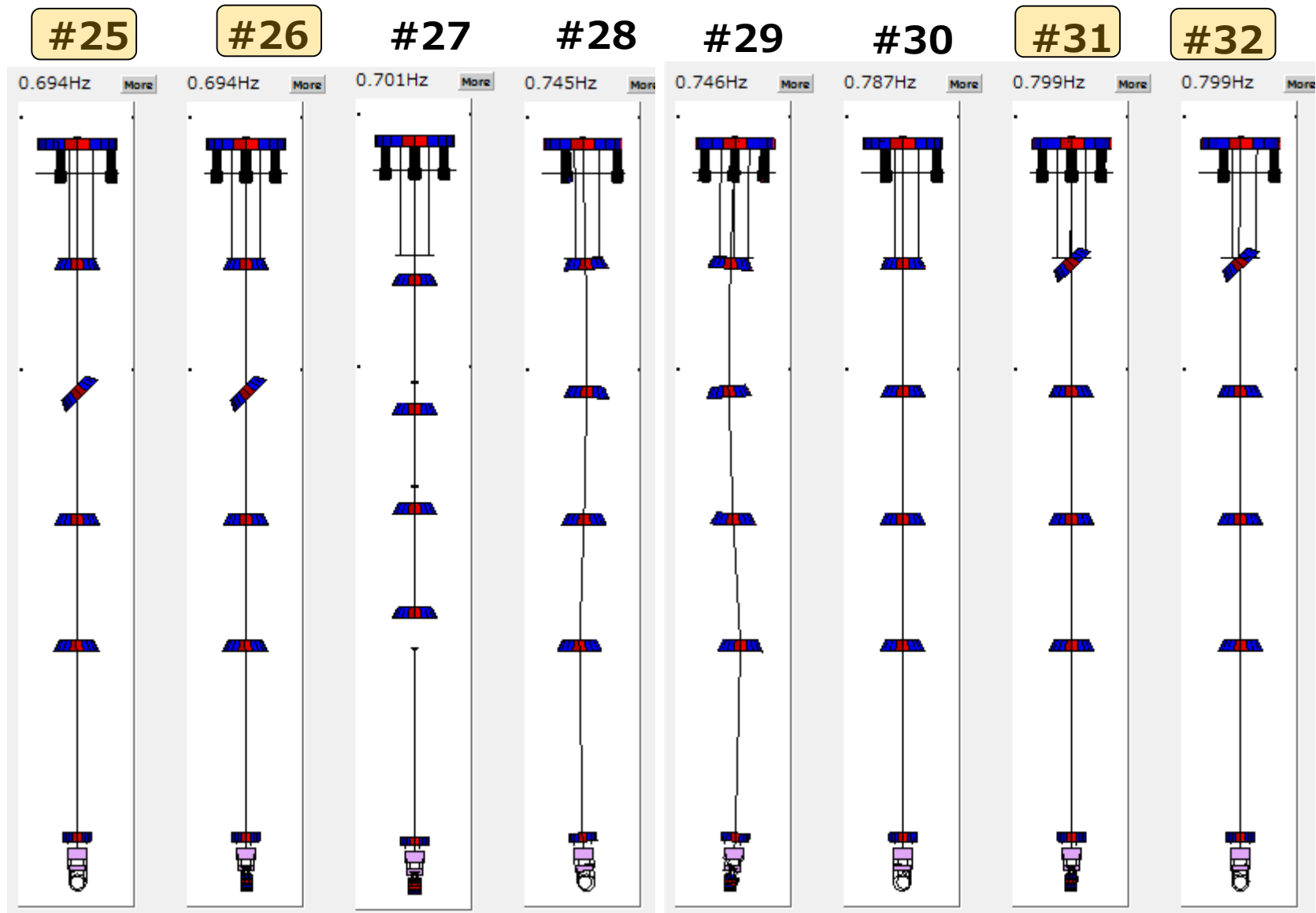
#22

#23

#24



Less interest now



Less interest now

#33

#34

#35

#36

#37

#38

#39

#40

0.817Hz

More

0.951Hz

More

0.972Hz

More

0.974Hz

More

1.061Hz

More

1.061Hz

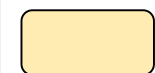
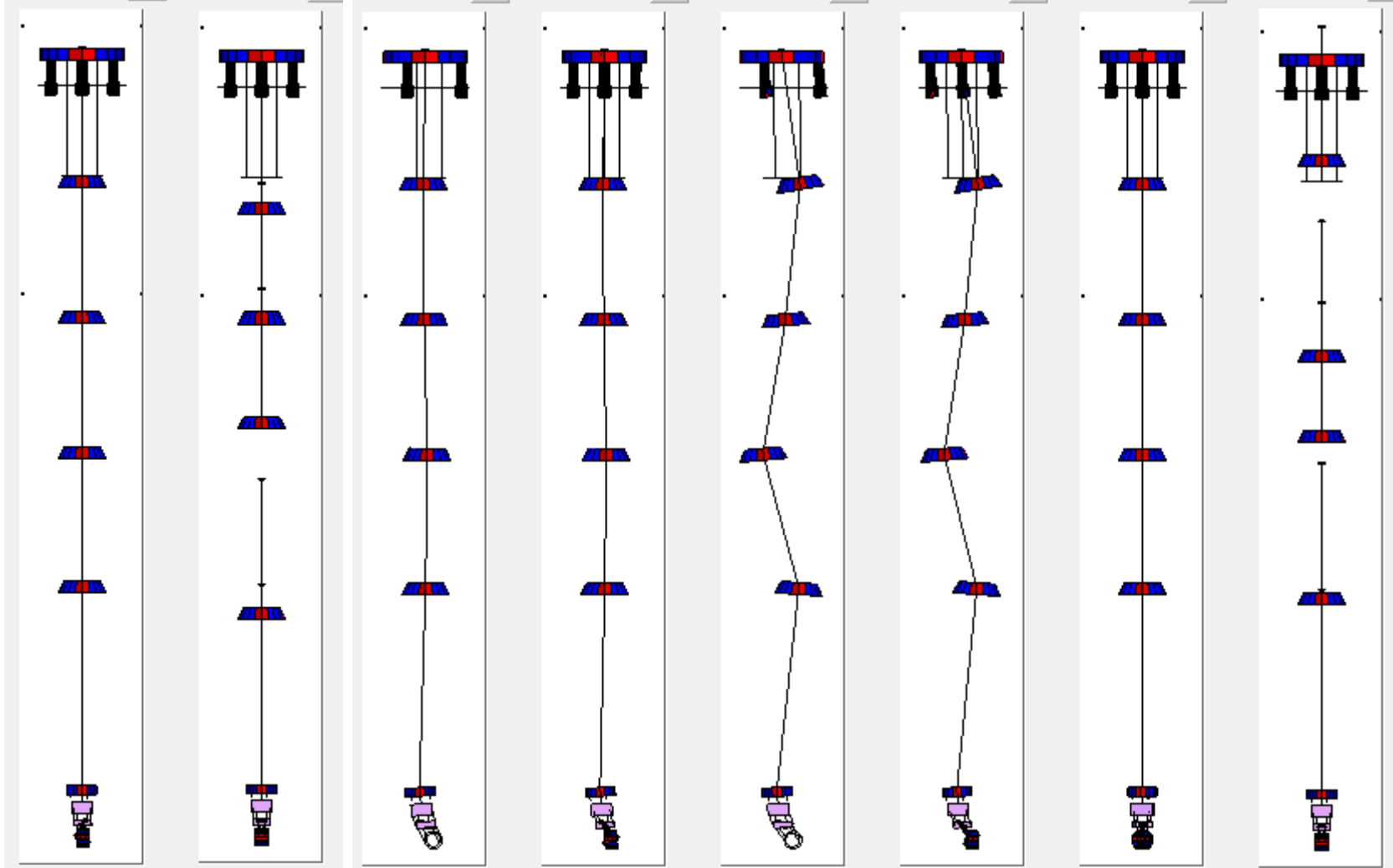
More

1.171Hz

More

1.189Hz

More



Less interest now

#41

#42

#43

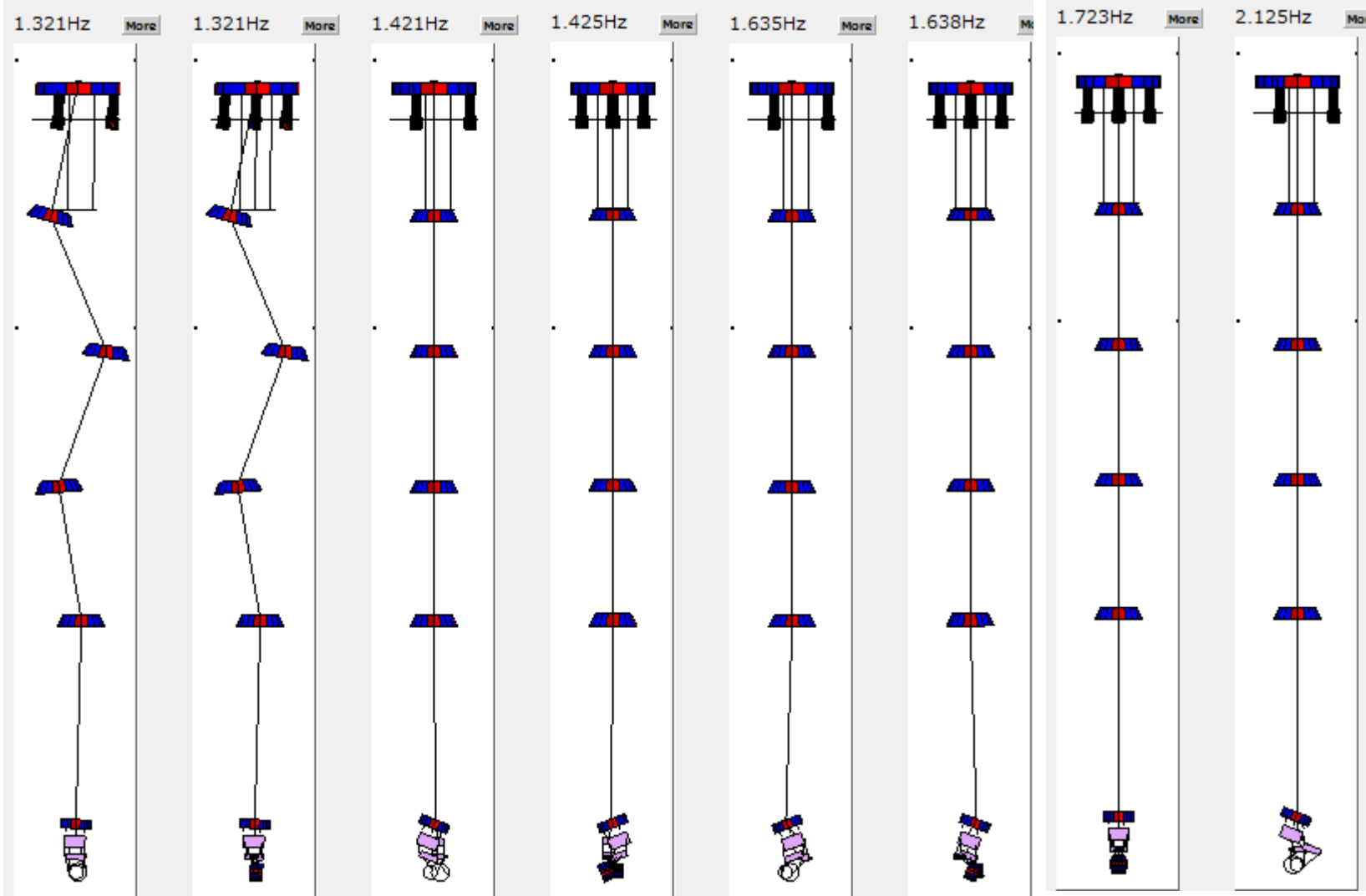
#44

#45

#46

#47

#48



 Less interest now

#49

#50

#51

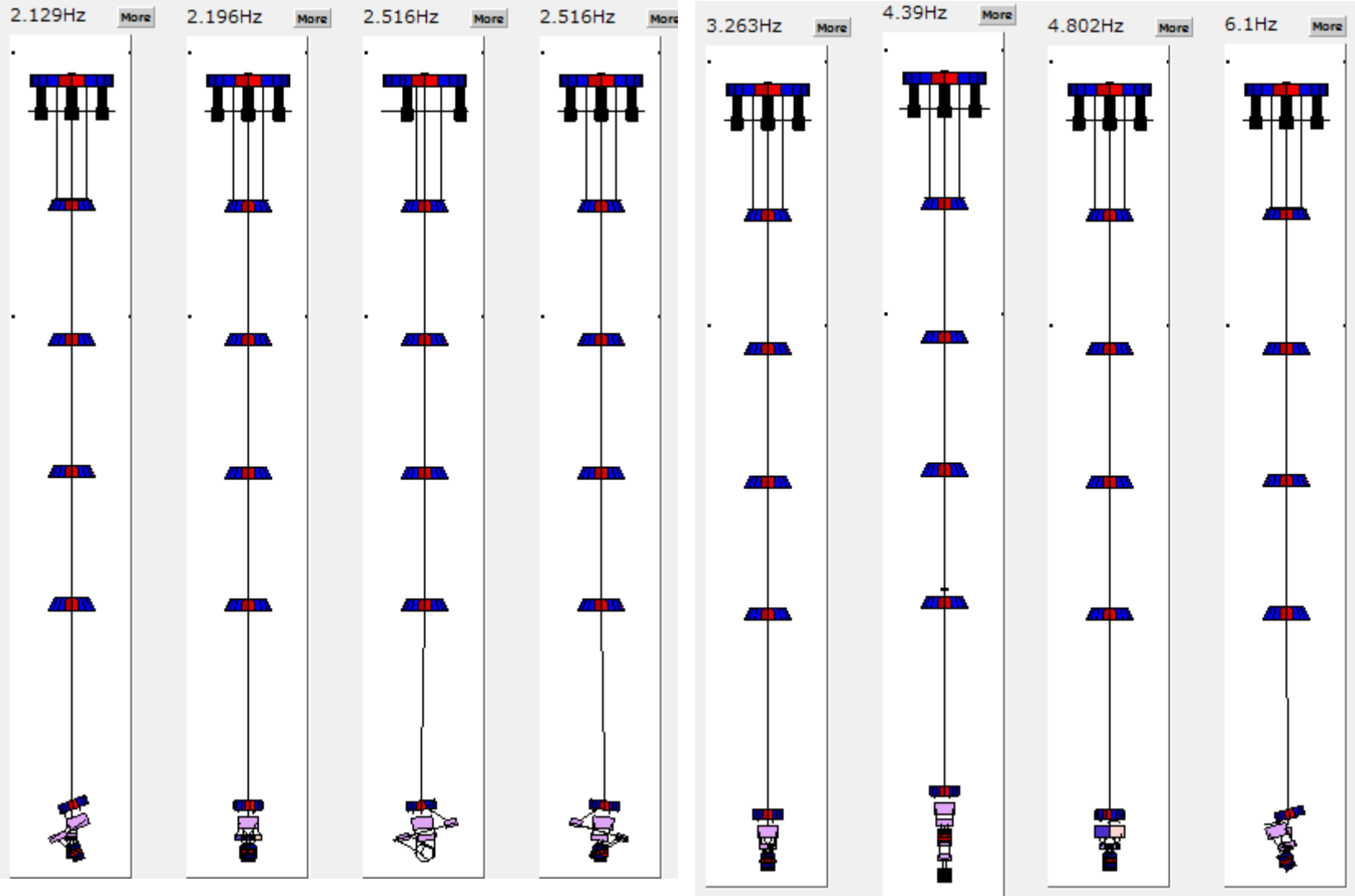
#52

#53

#54

#55

#56



 Less interest now

#57

#58

#59

#60

#61

#62

#63

#64

6.19Hz

6.646Hz

9.771Hz

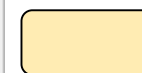
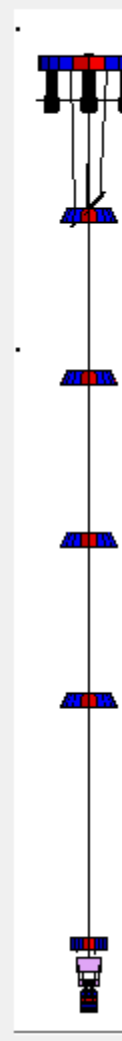
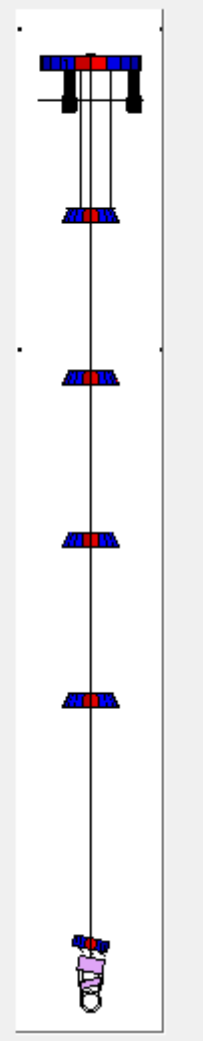
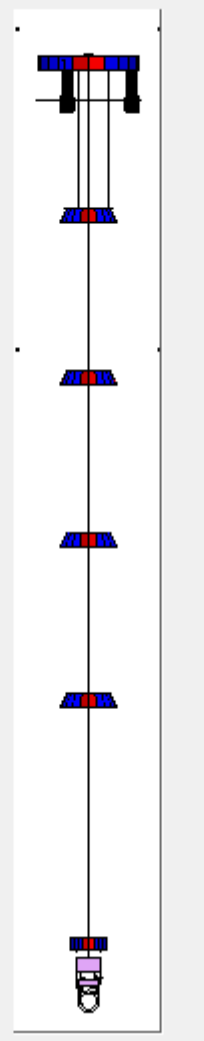
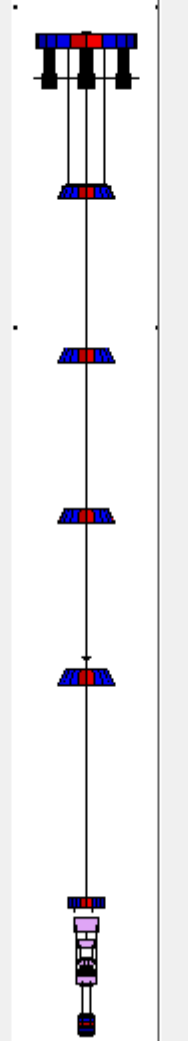
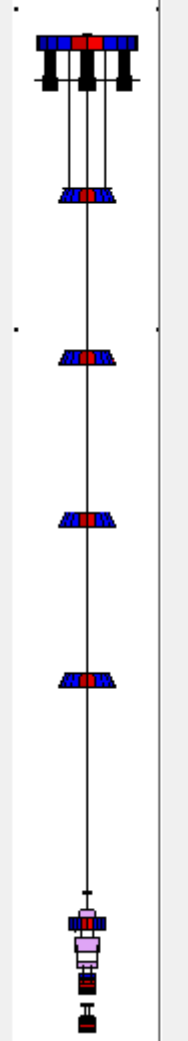
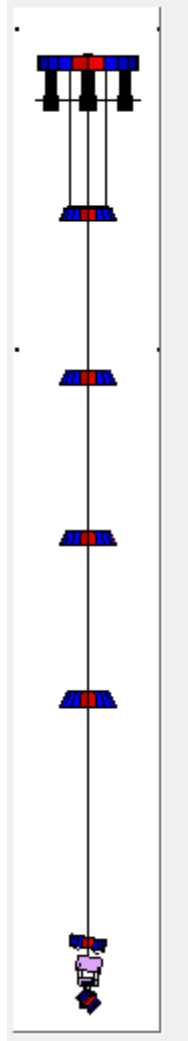
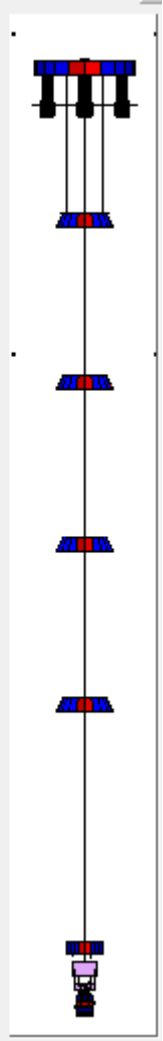
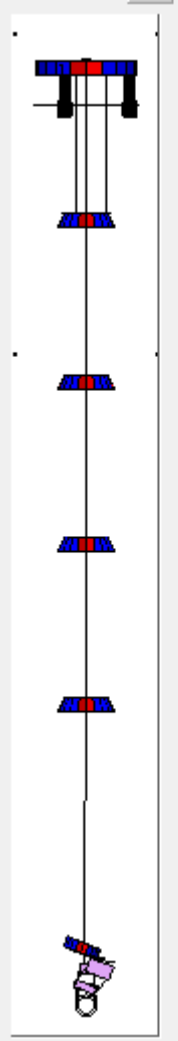
15.92Hz

20.17Hz

21.923Hz

23.685Hz

26.265Hz



Less interest now

#65

#66

#67

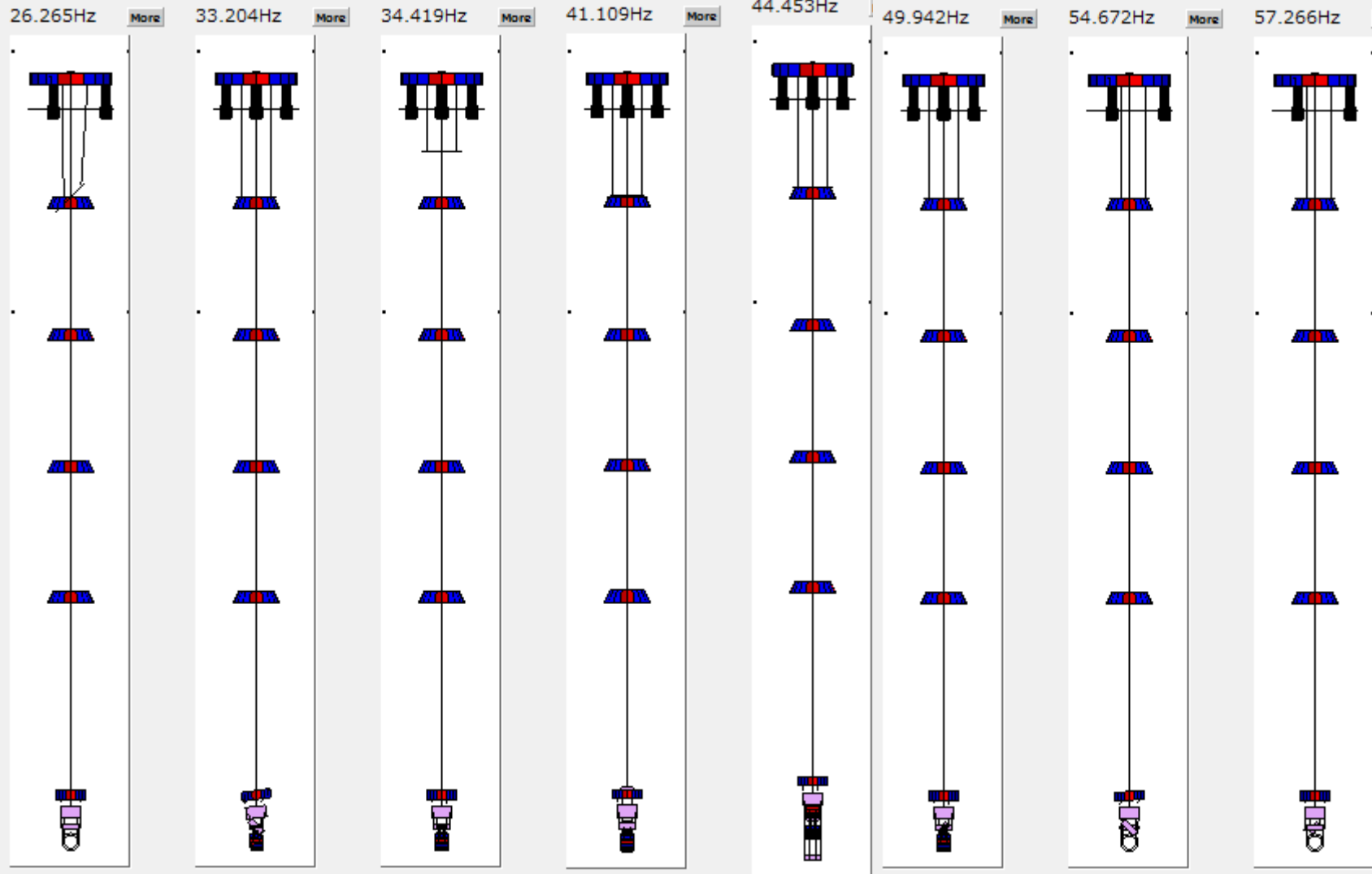
#68

#69

#70

#71

#72



Less interest now

#73

#74

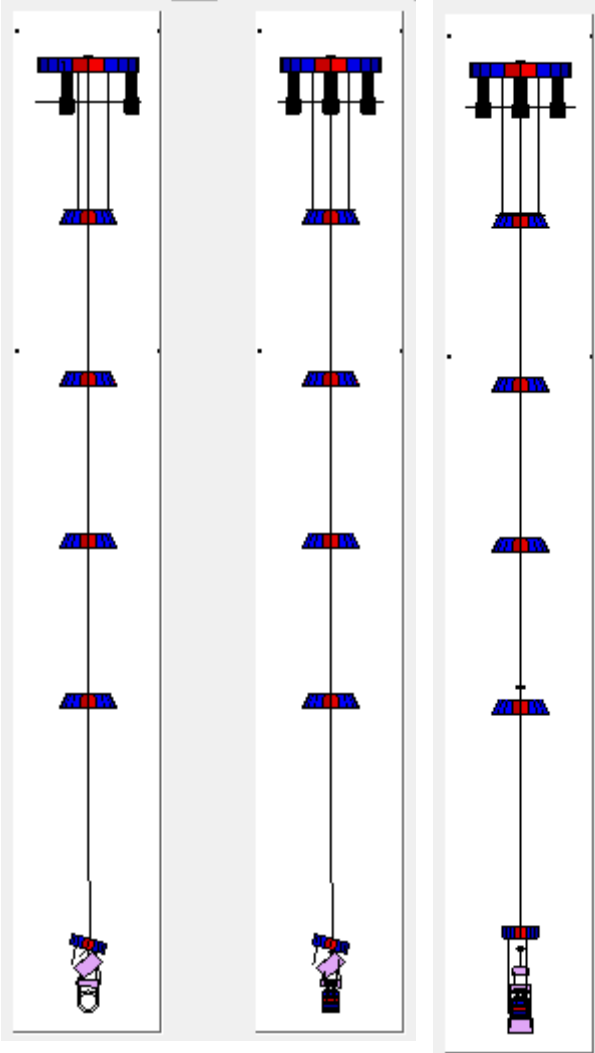
#75

150.563Hz

More

151.723Hz

183.437Hz



 Less interest now