

Control and tuning of suspension

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Contents

- Overview of suspension control systems
- Control design (for type-B SAS)
- Implementation of digital control system in prototype SAS



Local controls for SAS

Purpose

- Compensate the **DC drift** of low-frequency oscillators.
- Damp the mechanical resonances after earthquakes.
- Reduce the mirror **RMS velocity** for quick lock acquisition.





SAS Local control overview



IP servo:

- DC position (X,Y) control
- Thermal drift control
- Pendulum mode damping

GAS filter servo:

- DC position (Z) control
- Thermal drift control
- GAS filter mode damping

Payload servo:

- DC alignment control
- Pendulum/rotation mode damping



IP Controller

Sensor/actuator configuration **Control Servo** Motorized spring LVDTs ADC DAC calibration sensor matrix LVDT-1 voice-coil actuator matrix polarization sensor blending feedback filter Geophone (L-4C) actua tors S LVDT-2 $F_{\rm x}(s)$ ACT-1 LVDT-3 LVDT $F_v(s)$ A ACT-2 geophone geophones response $F_{\rho}(s)$ ACT-3 GEO-1 S Voice-coil 1/sH(s) GEO-2 Actuator GEO-3

- Similar configuration as TAMA-SAS/Virgo SA
- L-4C geophones as inertial sensors
- Sensor blending + SISO servo in Euclid coordinate space



Control noise study (in Type-B SAS)



- Simple viscous-damping control with various blending frequencies
- Blending frequency should be set <100 mHz to reduce mirror RMS velocity
- Noise coupling from geophone might be a problem if we raise the control gain at low frequencies for DC control





GAS filter controller

Sensor/actuator configuration

Fishing rod



- Simple DC servo will be applied to each GAS filter to compensate the thermal drift (typically ~0.7 mm/°C)
- Damping servo will not be required because of low Q(~10) of oscillation modes
- Motorized spring (fishing rod) for initial DC positioning



Payload Controller

Sensor/actuator configuration

Control system diagram (OSEM)



- OSEMs are used to damp the resonant modes.
- DC alignment should be set by optical lever/WFS signals.
- The intermediate mass has moving (motorized) masses to set DC tilt.



Active damping performance



- Resonant frequency vs decay time in simulation
- 1/e decay time gets less than 1 min. for most resonant modes
 - One exception: the mirror-RM lateral mode...





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Sensor noise coupling



* Assuming the sensing noise measured in prototype OSEM

- Sensor noise coupling with strong damping by OSEMs.
- Damping servo should be turned off during science run (at least about length DoF)

Damping servo about other DoFs might be kept on (TBC)

BS requirement in science mode



Notes

- Actuator noise couplings are studied by Y. Michimura (<u>JGW-</u> <u>T1503453</u>).
- MATLAB codes and Simulink models for suspension control simulation: <u>JGW-T1503606</u>



Implementation, Control Test in Prototype SAS



Digital control system for prototype SAS

User interface (MEDM) for prototype SAS control







Digital control system for prototype SAS

Signal flow in each system:



Sensing matrix is determined from sensor layout geometry Coil matrix is set from the measurement so that it diagonalizes virtual actuators.



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Actuator diagonalization

- 1. Coil matrix elements are set arbitrary (roughly set from actuator layout).
- 2. Measuring the coupling from virtual actuators to virtual (local) sensors by actuating in different DoFs separately at a certain frequency (10-100 mHz).
- 3. Calculating a new coil matrix from the measured coupling coefficients.
- 4. Repeat the procedure 2 and 3 for several times.

We diagonalize actuators with local sensors in DC (at low freq.)

- Simple, but time consuming!
- Now writing a script to automatize it...





Transfer function measurement



Measuring transfer functions from diagonalized actuators to virtual sensors

Comparing with models → Check if any fricting/broken part exists



Watchdog

- After large excitation:
 - Geophone signals saturate and the control gets crazy.
 - Optical lever beam spot goes away from QPD.
 - Actuator signals saturate and no AC control is applied.

- We need a system automatically turns off the control / switch it to more robust one.
- Now we are designing software watchdog system (firstly aimed for long-term measurement of prototype SAS)



Summary

- We are developing local control system with control noise taken into account.
- Active damping works well, while one un-damped mechanical mode is found.
- Need to make control strategy: switching controls after lock acquisition?
- Control digital system is implemented for SAS prototype and now is working.
- Efficient way of actuator diagonalization and watchdog system are to be developed.



Appendix



Sensor/seismic noise model

- LVDT: measured noise
- Geophone: measured preamp noise
- Seismic: (<2 Hz) 90% percentile data of 1.5 year measurement in CLIO site, (>2 Hz) 1/f² interpolation



Blending filter



- Geophone noise raises at low frequency by ~f^{3.5}
- Steeper high-pass is required to avoid noise coupling
- Blending filter with 7th order polynominal

$$H_{HP}(s) = \frac{s^7 + 7s_0s^6 + 21s_0^2s^5 + 35s_0^3s^4}{(s+s_0)^7},$$
$$H_{LP}(s) = \frac{35s_0^4s^3 + 21s_0^5s^2 + 7s_0^6s + s_0^7}{(s+s_0)^7}$$

