<u>Vertical Suspension-</u> <u>Point Interferometer</u>

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イントロダクション (余談)

Outline



Original motivation and design of VSPI
Possible application in KAGRA and TOBA
Analytical calculation and results
Conclusion





This presentation may contain incomplete or incorrect descriptions. The purpose is not to show an achievement, but to stimulate discussions for new ideas.

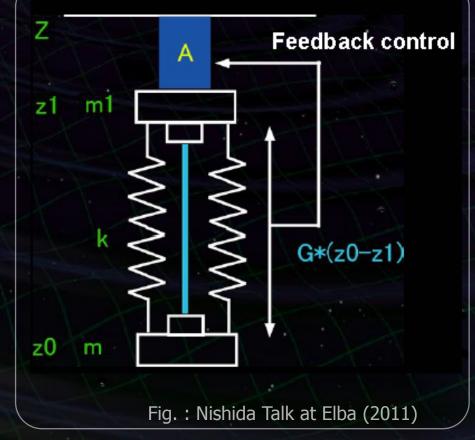
Original Motivation of VSPI



Y. Aso, Ph.D thesis (2006)

•VSPI (Vertical Suspension Point Interferometer)

- Active control scheme to suppress vertical fluctuation, mainly for seismic noise.
- Control upper mass to follow the better-isolated lower mass.
- Fundamental performance is limited by sensor noise .
 - \rightarrow Interferometric sensor.
- Sensitivity is relaxed by the H-V coupling factor.



Resonant Frequency



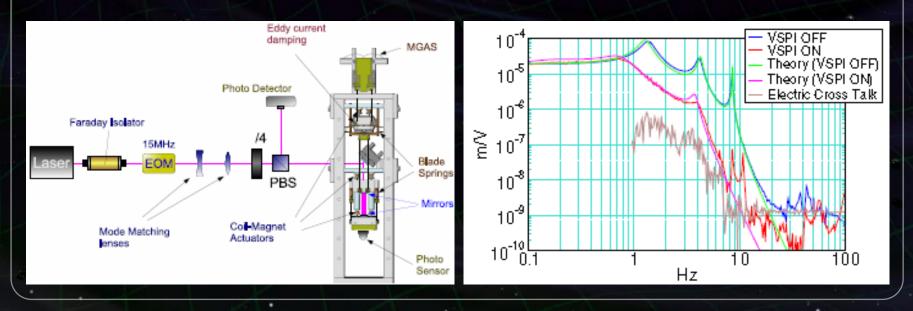
Reduction of resonant frequency by VSPI

- Smaller restoring force at suspension \rightarrow lower resonant freq.
- Reduction factor : $\sqrt{1+G}$ for res. freq. and Q-value.

Ref: Y. Aso, Appendix D in Ph.D thesis (2006).K. Kawabe, in 'Detection of GW' p.257 (1998) in Japanese.N. Mio, in 'GW handbook' p.201 (1992) in Japanese.

VSPI response measurement by Aso

Y. Aso, Ph.D thesis (2006)

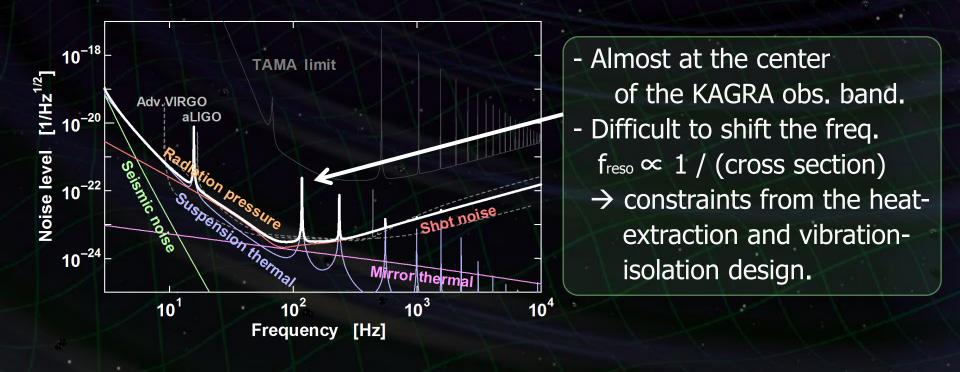


VSPI possibility in KAGRA??



E.Nishida Talk at GWADW Elba (2011)

 Thermal noise peak at ~120Hz, by a vertical resonance at the sapphire suspension
 VSPI may solve this problem



KAGRA face-to-face meeting (Aug. 1st, 2012, ICRR, Kashiwa)

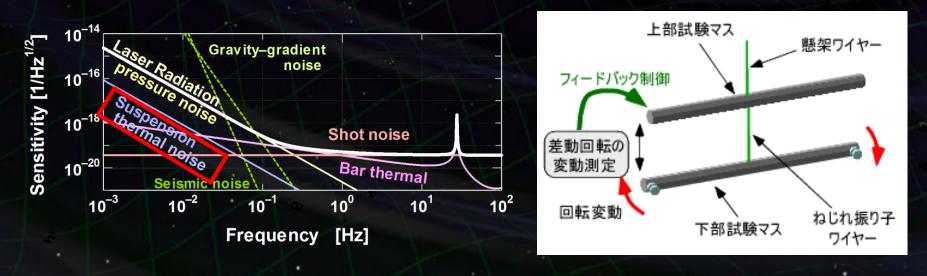
SPI in TOBA



M.Ando (2011)

• 'Tuning bar' technique in TOBA, same concept as VSPI, but for rotational DoF.

- Suppression of the suspension thermal noise?
- Lower resonant frequency
- Seismic noise suppression
- > Very low-freq. GW observation



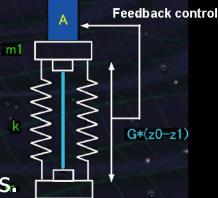
Physical Explanation



•Why thermal noise is suppressed by SPI???

 Explanation 1 : FDT (Fluctuation-Dissipation Theorem) According to FDT, thermal noise depends on the imaginary part of the mechanical conductance:

By SPI, mechanical response becomes zero, no deformation in the suspension wire by external forces.



- Explanation 2 : Reduction of effective mass

 $\tilde{x}_{\text{thermal}}^2 = -\frac{4k_{\text{B}}T}{\omega} \text{Im} \left| \frac{\tilde{x}}{\tilde{f}} \right|$

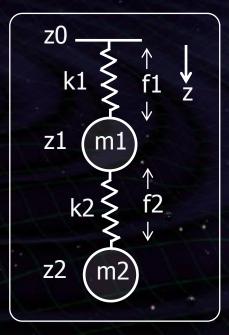
SPI effectively reduces the inertial mass of the upper mass. If the mass of the upper mass is small enough, the lower mass does not move by the force noise at the suspension wire.

Calculation Model



Equation of Motion

$$\begin{cases} m_1 \ddot{z_1} = -k_1(z_1 - z_0) + k_2(z_2 - z_1) + f_1 - f_2 + \\ m_2 \ddot{z_2} = -k_2(z_2 - z_1) + f_2 \\ f_c = G(z_2 - z_1) + f_2 \end{cases}$$
 (Control force)



fc

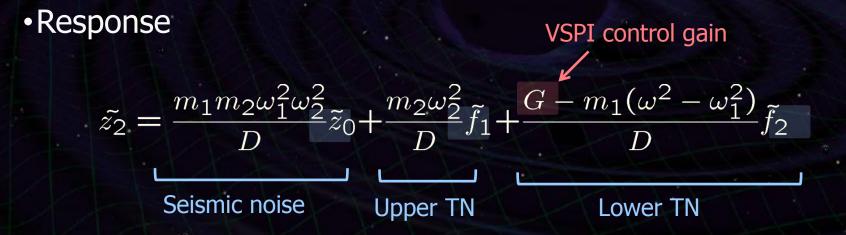
Fourier transformation to obtain

 $\begin{bmatrix} G - m_1 \omega^2 + k_1 + k_2 & -G - k_2 \\ -k_2 & -m_2 \omega^2 + k_2 \end{bmatrix} \begin{bmatrix} \tilde{z}_1 \\ \tilde{z}_2 \end{bmatrix} = \begin{bmatrix} k_1 \tilde{z}_0 + \tilde{f}_1 - \tilde{f}_2 \\ \tilde{f}_2 \end{bmatrix}$

 \Box Can be solved for z_2

Calculation Results





Determinant

 $D = -m_1 m_2 \omega_1^2 (\omega^2 - \omega_2^2) - m_2 \omega^2 \left\{ G - m_1 \omega^2 + (m_1 + m_2) \omega_2^2 \right\}$

•Resonant frequencies (for single pendulum) $k_1 = m_1 \omega_1^2$ $k_2 = m_2 \omega_2^2$ Results (1/3)



•High-frequency and infinity gain limit:

- $\omega \gg \omega_1, \, \omega_2, \, (k_1 = m_1 \omega_1^2, \, k_2 = m_2 \omega_2^2)$ $G \to \infty$
- Seismic noise (z0) and thermal noise at the upper stage (f1) are suppressed by the VSPI.
- Thermal noise at the lower stage (f2) is not suppressed.

 $ilde{z}_2 \sim -rac{f_2}{m_2\omega^2}$: Same as the free-mass limit

Results (2/3)



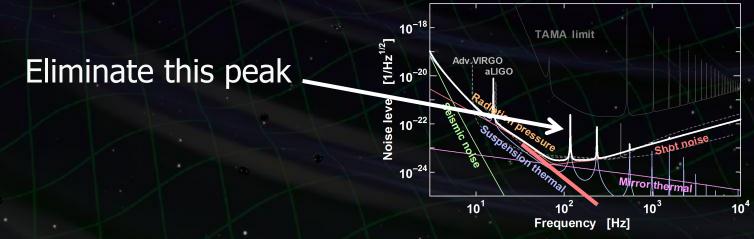
•At the resonance of lower suspension

 $\omega \simeq \omega_2$

 $G \to \infty$

- VSPI is effective to damp the resonance.

 $\tilde{z}_2 \sim -\frac{f_2}{m_2 \omega_2^2}$: Same as the free-mass limit



Results (3/3)



 Tune the VSPI control gain $\omega \gg \omega_1, \, \omega_2, \, (k_1 = m_1 \omega_1^2, \, k_2 = m_2 \omega_2^2)$ $G \sim m_1 \omega^2$ $\overline{}$ - Thermal noise at the lower stage (f2) is cancelled. $\tilde{z}_2 = 0$ - Thermal noise at the upper stage (f1) is not cancelled. $\tilde{z}_2 = -\frac{f_1}{(m_1 + m_2)\omega^2}$

*Thermal noise fluctuation is monitored and fed back to the upper stage to cancel its motion. Same suppression may possible by off-line data processing.

Physical Interpretation



•What is the SPI's effect???

- Explanation 3 : Change the mechanical response SPI changes the spring constant of the suspension $\frac{21}{21}$ minutes wire : $k_2 \rightarrow 0$. The thermal fluctuation force by the suspension remain.

Z1 m1 k z0 m

Free mass with suspension thermal noise.

Discussion



Consistency between physical explanations ????
(1) FDT, (2) Effective mass, (3) Mechanical response.

- \rightarrow K.Yamamoto's comment
 - FDT is valid only in a thermal-equilibrium system, and cannot be applied to an actively-controlled system like this.
 - Langevin equation would be valid even in this case.
- •Use VSPI in KAGRA ???

Please discuss considering: necessity, technical readiness, interface constraints, ...

Conclusion



- •VSPI is effective to suppress vibrations from upper stages, such as seismic noise, thermal noise and so on.
- •Simple VSPI control ($G \rightarrow \infty$) would not be effective to suppress the thermal noise at the lower-stage suspension thermal noise. However it damps the resonant peak.
- •With gain tuning, it is possible to cancel the lower-stage thermal noise. However, upper-stage thermal noise would be a problem in this case.
- •For TOBA, SPI ('tuning-bar technique') will be useless for thermal noise, but wil be useful to expand the observation band to lower frequencies.

Conclusion



