



Two Layers SAS: Damping of Torsion Mode

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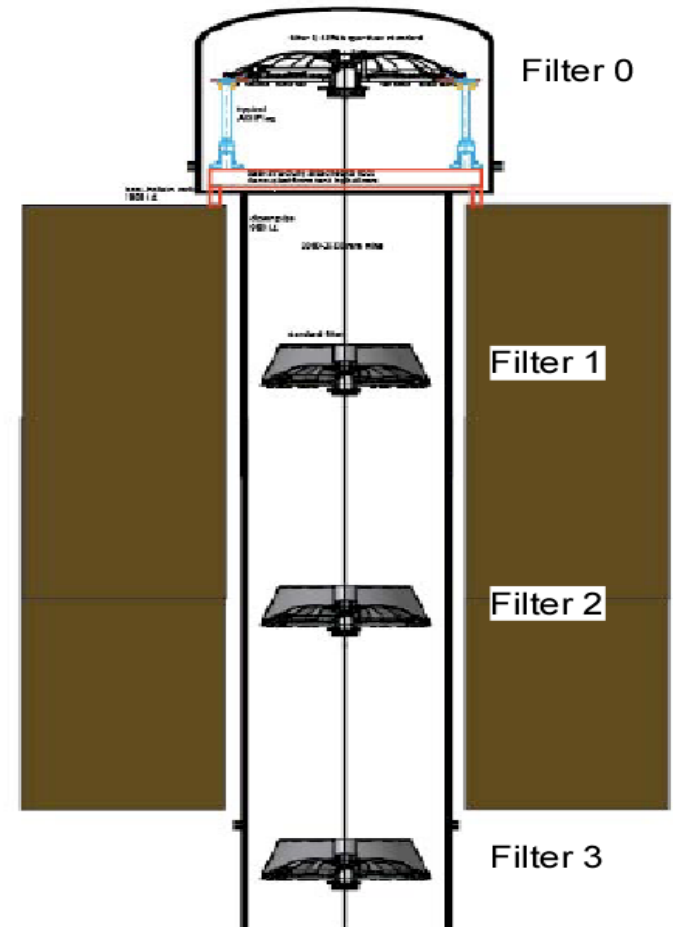
F2F Meeting

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Torsion Mode of Filters

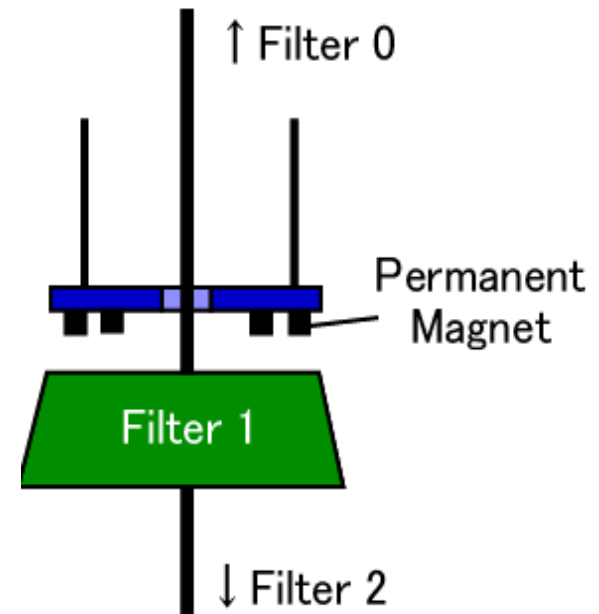
- Each GAS filter is suspended by a single wire.
- Single wire is flexible in torsion mode → good attenuation in yaw mode (rotation about vertical axis)
- This mode has quite low resonant frequency & high Q
- Once this mode is excited, filters keep oscillating for a long time. (During this time, we cannot lock interferometer.)



How to Damp This Mode?

- A disc is suspended from Filter 0.
- Permanent magnets attached to the disc damp the motion of Filter 1 by eddy current.
- The motion of Filter 2 & 3 is also damped, if the torsional stiffness of each wire is optimal.
- We need to optimize the damping strength & torsional stiffness of every wires.

→ Need Simulation

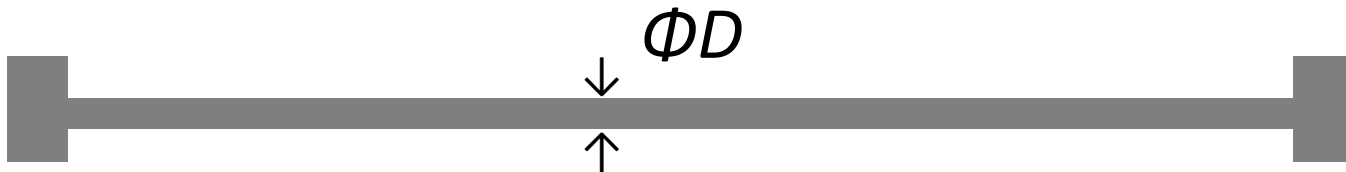


Torsional Stiffness of Wire

- Torsional stiffness depends on the thickness of the wire.

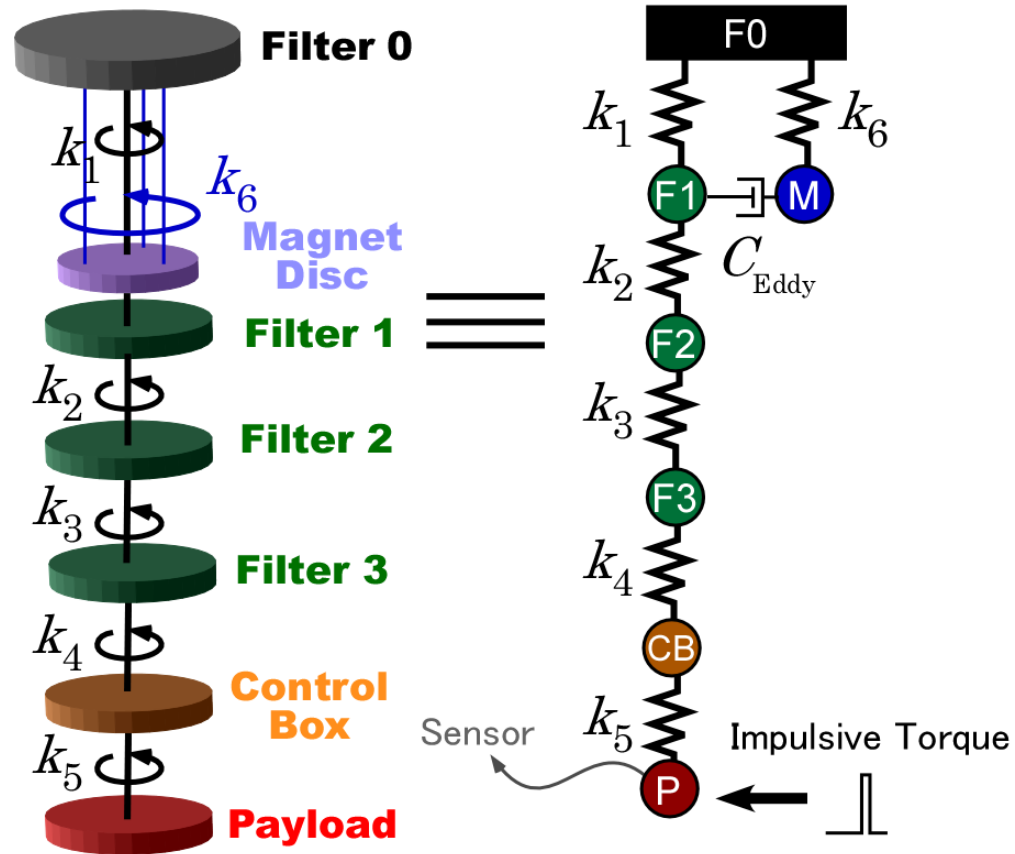
$$k_{\phi} \propto D^4 \quad D: \text{Wire Diameter}$$

- Find optimal torsional stiffness of each wire
= Find optimal thickness of each wire

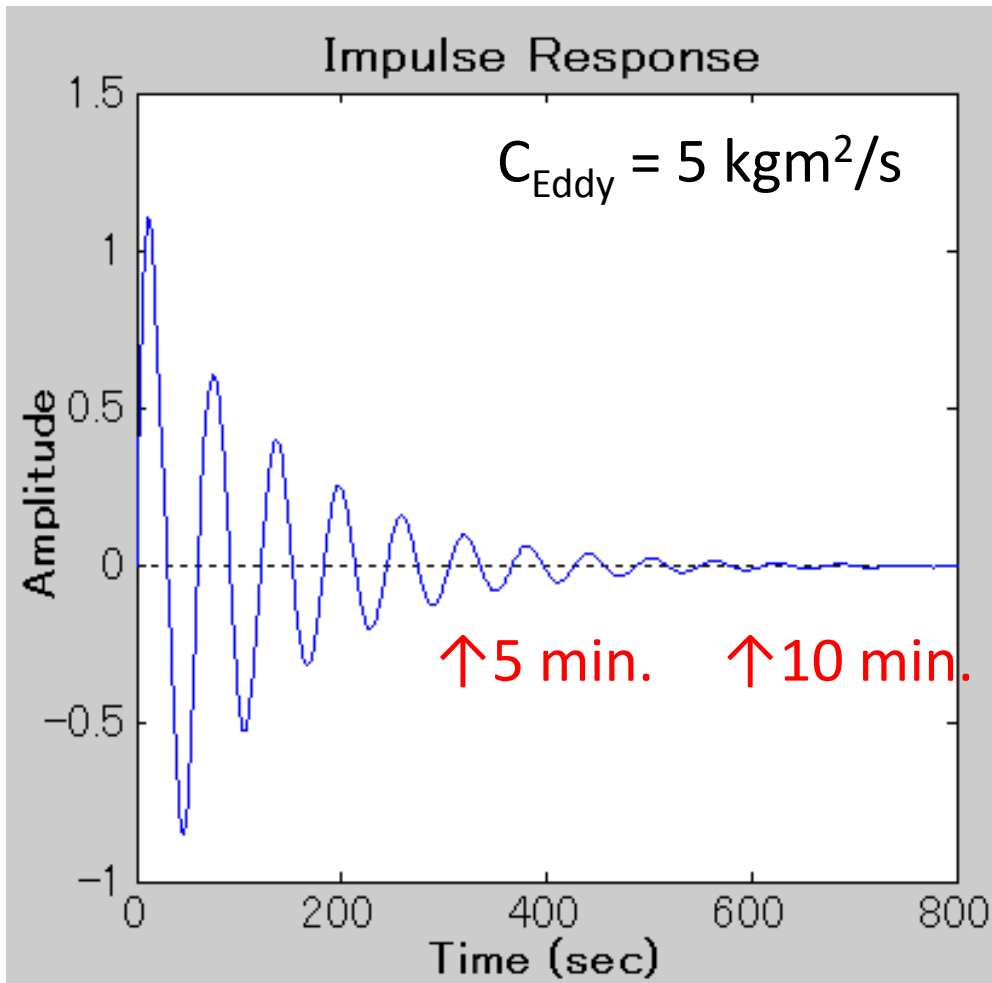


Simulation Model

- 1-D “point-mass” model
- Impulsive torque is exerted to the payload, and angular displacement of the payload is sensed.
- Calculate impulsive response
→ Estimate damping time
- Find best parameters that shorten the damping time most



Simulation Result & Optimized Parameters



| | Value [Nm/rad] |
|-------|----------------|
| k_1 | 0.70 |
| k_2 | 0.70 |
| k_3 | 0.70 |
| k_4 | 0.27 |
| k_5 | 0.27 |
| k_6 | >10 |

Torsional modes are damped in < 10 min.

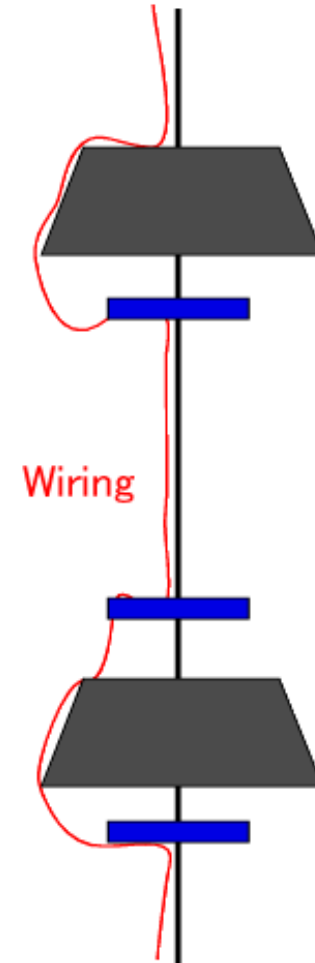
Proposed Wire Design

- From the torsional stiffness, wire diameters are calculated.

| # | Diameter [mm] | Length [m] |
|---|---------------|------------|
| 1 | 3.74 | 2.1 |
| 2 | 3.74 | 2.1 |
| 3 | 3.74 | 2.1 |
| 4 | 2.94 | 2.1 |
| 5 | 2.45 | 1.0 |

Note: Wiring Problem

- In real case, wiring (for sensors and actuators) also introduces distributed damping.
- This damping is less welcome because it may introduce creak noise.
- We will study wiring configuration experimentally with prototype.



End

Appendix

Parameters Used in the Simulation

- Moment of inertia of each filter

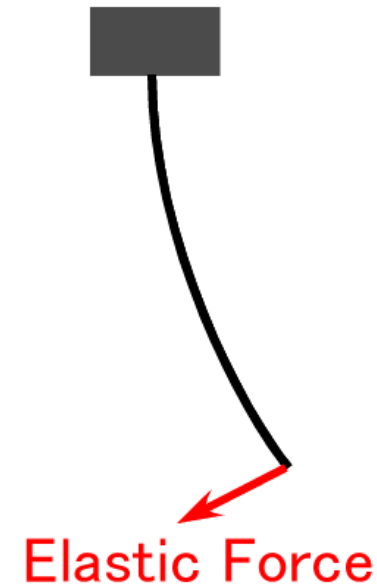
| Filter Name | Detail | Moment of Inertia [kgm ²] |
|-----------------|-------------------|---------------------------------------|
| Standard Filter | Filter 1~3 | 6.24 |
| Payload | Platform | 1.95 |
| | Magnet Box | 2.35 |
| | Intermediate Mass | 0.69 |
| | Recoil Mass | 0.45 |
| | Test Mass | 0.17 |
| | Total | 5.61 |
| Control Box | | 2.0 |
| Magnet Disc | | 1.84 |

- As the design of the control box is not fixed, its moment of inertia may change.

Proposed Wire Design (Detail)

- From the torsional stiffness, wire diameters are calculated.

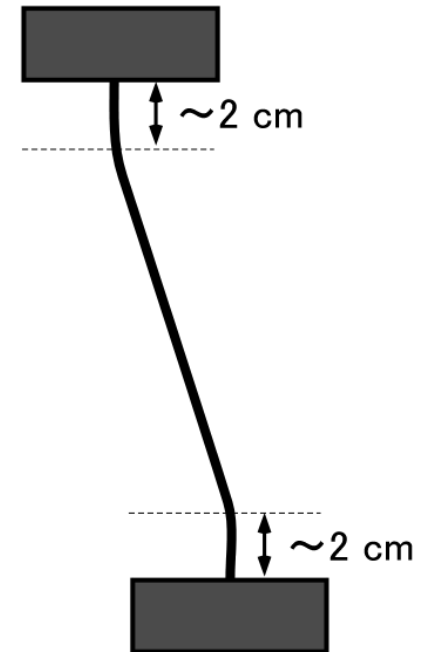
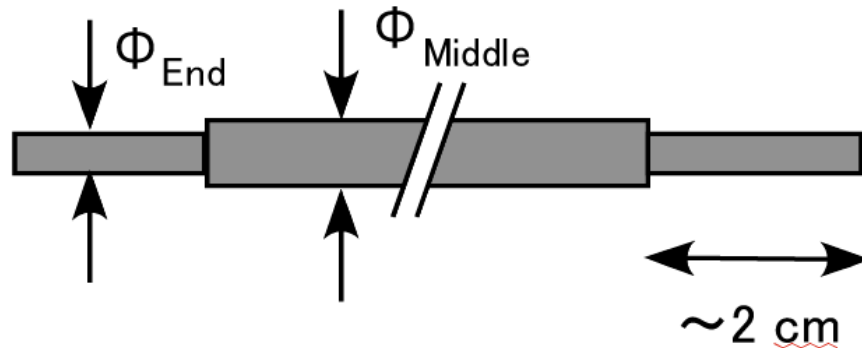
| # | Diameter [mm] | Length [m] |
|---|---------------|------------|
| 1 | 3.74 | 2.1 |
| 2 | 3.74 | 2.1 |
| 3 | 3.74 | 2.1 |
| 4 | 2.94 | 2.1 |
| 5 | 2.45 | 1.0 |



- Relatively thick wires (almost rods)
- Thick wires are not good for horizontal attenuation.
(Elasticity of wires may worsen the performance of SAS.)

Proposed Wire Design (Detail)

- This elastic stiffness only depends on the thickness of first few cm from the head of the wire.
- We can make wires **thick in the middle** & **thin at the ends**.



| # | Diameter (End) [mm] | Diameter (Middle) [mm] | Length [m] |
|---|---------------------|------------------------|------------|
| 1 | 3.74 | 3.74 | 2.1 |
| 2 | 3.44 | 3.74 | 2.1 |
| 3 | 3.11 | 3.74 | 2.1 |
| 4 | 2.74 | 2.94 | 2.1 |
| 5 | 2.45 | 2.45 | 1.0 |